

Photo 1



HANDBOOK

For *Automatic SCREW MACHINES*

* *Swiss Type* *

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GEORGE **BRIDOK** MACHINE CO.
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Photo 3

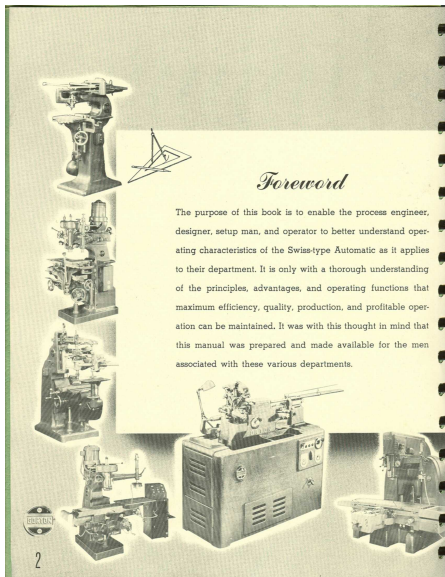


Photo 4

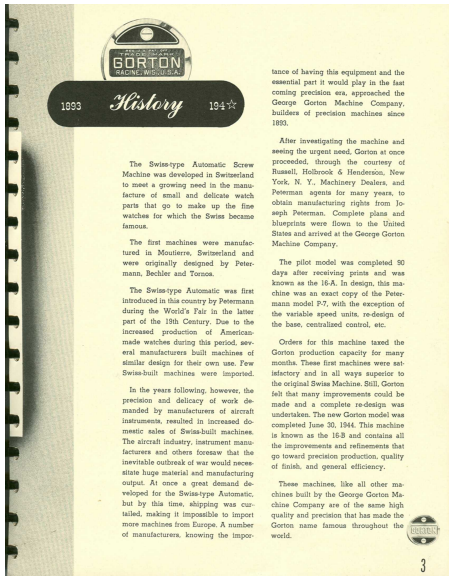
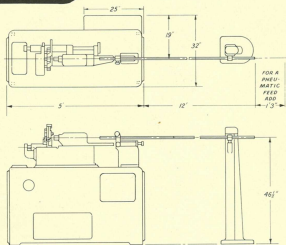


Photo 5

FLOOR PLAN and SPECIFICATIONS



SPECIFICATIONS

1947-H

Maximum Chuck Diameter	3/16"	Camshaft Motor—Ball Bearing	1 1/2 h.p.
Minimum Chuck Diameter031"	Coolant Pump Motor	1/4 h.p.
Max. Turning Length—Flat Cam	1 1/4"	Height (Flare to Center of Spindles)6015"
Max. Turning Length—Ball Cam254"	Flare Spindle Swept, 180° x 32°—	plus 8 H. clearance beyond
Range of Spindle Speeds	1100 r.p.m. to 10,000 r.p.m.	left end of machine for retooling	
Range of Camshaft Feeds50 to 750 Pieces Per Hour	Max. Threading Cap. Int. or Ext. In. Mild Steel or Brass	1/2"
Optional Range50 to 1440 Pieces Per Hour	Max. Length of Thread2"
Size of Tool Bit	3/16" x 3/16"	Max. Drilling Depth2"
Spindle Motor—Ball Bearing—Two Speed	2 and 1 1/2 h.p.	Max. Length of Part That Can Be Shaped	3 1/2"
		Max. Dia. of Drill That Can Be Used Approx.	3/16"



Photo 6

CORRECT INSTALLATION

FOUNDATIONS—

Special foundations are not necessary for the Gorton High Precision Automatic. However, it should be placed on a substantial floor, wood or concrete reasonably flat and free from vibration. If placed on an upper floor or on a balcony, locate the machine close to or over a supporting beam to eliminate, as much as possible, any vibration from other shop equipment.

PLACEMENT OF MACHINES—

When erecting a battery of machines the suggested floor layout at left has proven to be very satisfactory, as it permits a single operator to attend several machines. Staggering the machines on an approximately 20° angle is recommended to save floor space and provide easy access for the operator.

Stock carriers must be in proper alignment with the work spindle of the machine before bolting to the floor although permanent anchorage of stock carrier or machine is not required.

IMPORTANT—

**Do not operate machine before
Leveling and Cleaning**

LEVELING—

It is essential that the Machine be carefully leveled. Do not use a carpenter's level or a level in a machinist's combination square. USE A PRECISION LEVEL, preferably one having a micrometer screw adjustment. It is not necessary to grout in the base, or bolt it to the floor. The well-balanced rotating parts of the Gorton 15-B eliminate the necessity of anchoring the base to the floor. Level the base by placing shims under the four corners of the base until the machine is level in both directions.

CLEANING—

Clean machine thoroughly with naphtha, or other grease solvent. It is important that the slushing compound and dirt accumulated in transit be removed before machine is placed in operation. THIS MACHINE IS A PRECISION TOOL, AND GREAT CARE MUST BE TAKEN IN REMOVING ALL GRIT OR OTHER ABRASIVE MATERIAL which collects on the machine while in transit.

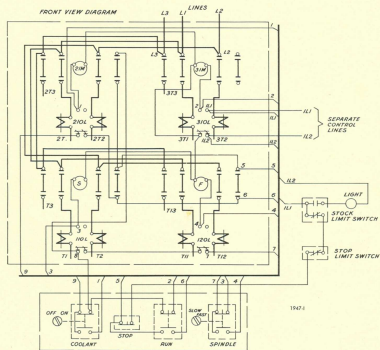
GORTON ENGINEERING SERVICE

In the event problems of unusual nature present themselves, our Engineering Service is at your disposal. Cams and tooling can be obtained at nominal cost in the Eastern United States through Russell, Holbrook & Henderson, Inc., 292 Madison Avenue, New York 17, New York; in the Midwest and Pacific area—George Gorton Machine Co., Racine, Wisconsin.



Photo 7

WIRING DIAGRAM



WIRING AND ELECTRICAL EQUIPMENT

The wiring diagram above is a complete layout of all the electrical controls and equipment on the GORTON Automatic Screw Machine. All motors, relays, switches and control buttons are clearly outlined in their relative positions.

The electrical equipment is located as remotely as possible from all oil used in the machine, but as a precautionary measure, Glyptol insulation is

provided; all wires are housed in metal conduit and concealed in the base of the machine. Oil resistant switches are used on the control panel, thereby eliminating any possibility of power leakage, which might endanger the operator. A fused circuit transformer steps down the voltage to 110 volts for all switches, relays, lights and controls. The motors operate from the service line proper.



CHARACTERISTICS

NOTE: In order to establish a standard, all speeds and feeds are calculated for the employment of high speed steel tool bits. Carbide tipped tools will increase production materially and we heartily encourage their use on applications where they will improve the job.

The GORTON Swiss-type Automatics are Precision Screw Machines primarily intended for precision work. However, they are capable of producing any work within their capacity and differ from the conventional type automatic in that they can handle production of long slender parts with practical facility due to an advantage in design which provides rigid support at the point closest to the cutting tool load.

This unique method employed by Gorton in advancing the bar into the tools, which are mounted radially in back of a guide bushing or steady rest which supports the work a minimum distance from the tools at all times, thus assures accuracies and high finishes regardless of length, slenderness or delicacy of the piece being machined, and frequently minimizes and sometimes eliminates subsequent operations.

To operate the Gorton Automatic at maximum efficiency, it is necessary that the jobs routed to it be of such a nature as to take advantage of the manifold operations afforded, for example: when extremely close tolerances must be held on long runs, when exceptional quality of finish must be obtained, when the concentricity of turned diameters in relation to each other and the O.D. of the stock must be maintained, when long slender turn-

ings exist either in front or in back of a shoulder, when there are irregular or tapered surfaces or any unusual multiple turning operation.

The unlimited use of the five tool positions in the turning of multiple diameters, generating irregular diameters, chamfering, etc., along with spot centering, drilling, boring, threading, and tapping from the attachments, permits a varied combination of distinct operations thereby making the machines quite versatile.

It should be remembered that the condition of the material to be machined determines to a large extent the accuracies to be expected in the finished product. Therefore, it is important on close tolerance work that the stock should be round within a total of .0003" and that the bars should be uniform throughout their length within $-.0005"$. The material itself may be of the ferrous or non-ferrous type; plastic or any other composition as long as it is rated in the machinable group.

Since single point tools are used, and accuracy of all turning depends largely upon the design and accuracy of the cams, it is important that the designer as well as the set up man and operator be thoroughly familiar with the characteristics of the machine and attachments.



THE MACHINE PROPER

The headstock spindle is powered by a 2 H.P. two speed motor through a Reeves Variable Drive. A 1/3 H.P. constant speed motor drives a Graham Variable Transmission which in turn varies the cam feeds.

All operational movements of the machine and attachments are controlled from the camshaft by means of flat or bell type cams. Power is obtained from the Graham Transmission by means of roller chains and sprockets. The chains are located inside the base and run through a special lubricant, thereby reducing wear to a minimum.

The headstock is a moving member working on hardened steel ways and is relative to all lateral motions of the bar. The spindle construction is anti-friction bearing type, precision built and sturdily constructed to withstand all rigors of high speed production — yet retaining the accuracies necessary to produce precision work.

The tool frame carries all of the tool holders relative to turning operations of the machine. Three

of these slide holders working in dovetail ways are often referred to as the overhead holders, because of their position in relation to the bar. The other two holders are mounted upon a single arm, commonly called the rocker arm (derived from its operational motion) and the tools are therefore called rocker arm tools.

The actuation of the rocker arm tools is obtained from a single cam. The overhead tools are mounted individually and independent of each other. Their operational movements are derived from independent cams. All tools may be adjusted axially or radially by means of micrometer screws, thereby gaining accuracies of tool setting in a minimum amount of time.

The cams necessary to operate various moving parts of the machine are usually laid out in sets for a specific job except on pieces of simple nature or relative design. The cams themselves may be of a soft material (cast iron) when small quantity production runs are made, but on long runs, hardened cams are preferable.

THE THREE SPINDLE ATTACHMENT

This attachment mounts against a scraped surface on the front part of the machine and as its name implies, carries three work spindles. Two of these spindles are limited to spot centering, drilling, and reaming. The third spindle may also be used for these operations, but is primarily designed for threading. The travel of these spindles is obtained

from a common lever which derives its motion from a bell type cam. The positioning of spindles is controlled by two flat cams. Power for revolving the second and third spindle is obtained from the main drive shaft in the base of the machine. The first spindle does not rotate and no provision is made for revolving it at any time.



Photo 10

SINGLE SPINDLE DRILLING ATTACHMENT

The mounting of this attachment is identical to the three spindle attachment and as the name implies, it has only one working spindle. This spindle is mainly intended for fine drilling, as very high drilling speeds may be attained. Also, its infeed may be of a positive or sensitive nature. By positive

is meant where the throw on a feed cam forces the drill into the work to a determined depth. By sensitive is meant where springs force the drill into work piece and the drill's cutting edge determines speed and depth of cut, thus reducing drill breakage to a minimum. Power for revolving the spindle is also obtained from main drive shaft.

SINGLE SPINDLE THREADING ATTACHMENT

This single spindle attachment is designed primarily for the threading of diameters from $\frac{1}{8}$ " to $\frac{1}{4}$ ". Typical of this attachment is the use of self

opening die heads. It can be used for external threading only.

SCREW SLOTTING ATTACHMENT

This attachment is, in principle, little different from any of the conventional screw machine slotters. Accuracies and cleanliness of the slot depends largely on the set up, material, and the cutter.

This attachment is mounted in place of No. 3 tool leaving four tools available for turning operations which are usually more than sufficient for producing most screws.

REVOLVING SUPPORT

The revolving support is used primarily where the shape of the bar is irregular such as hexagon, square, etc. In this type support, a sleeve is revolved or driven from the spindle nose and con-

tains a guide bushing which will be turning at the same speed as the bar. The lateral advance of the bar is the same as when using the stationary support.

STOCK GRIPPER

The stock gripper attachment is used in place of the weight or pneumatic feed when bars are not of suf-

ficient diameter to withstand pressure at end of bar that would be set up in the aforementioned method.



Photo 11

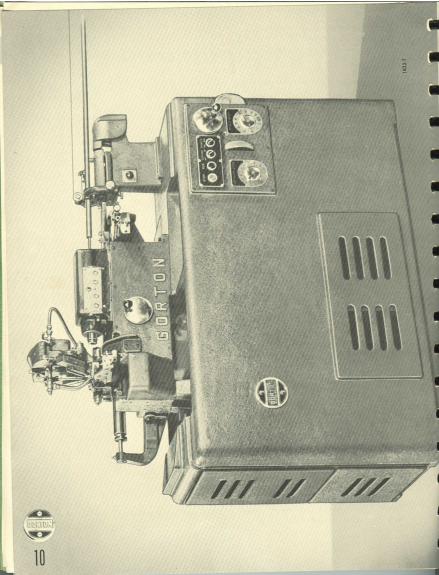


Photo 12

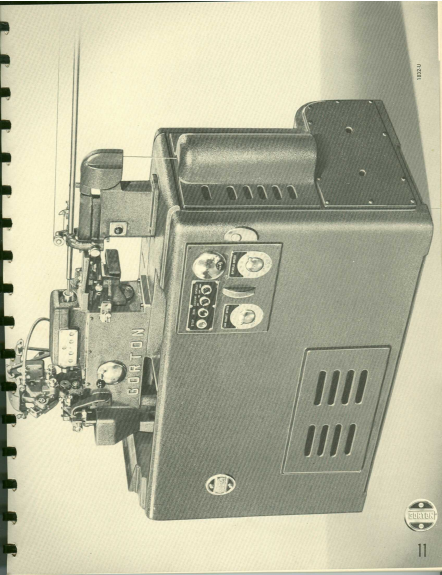
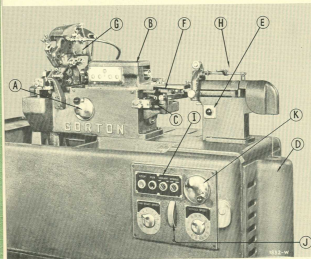


Photo 13

CONTROLS



- A—Manual Operation Hand Wheel
- B—Headstock
- C—Flat Feed Cam Mechanism
- D—Headstock Retraction Weight
- E—Bar Stock Signal Light
- F—Tee Block
- G—Tool Frame
- H—Automatic Stop
- I—Control Panel
- J—Camshaft Speed Adjusting Wheel
- K—Headstock Spindle Speed Adjusting Wheel

In illustration above, a front view of the upper portion of the machine is shown. The handwheel A is used for operating the camshaft manually, also to engage the clutch when putting the machine in automatic operation. Pull out for starting and push in to disengage.

The headstock B carries the main spindle, driving collet and collet closing mechanism. Forward travel of this unit advances the bar into the tools. The transmission of power to this unit from the cam is accomplished through flat feed cam unit C or when turning pieces over 1½" in length, the bell cam unit shown mounted in illustration, page 21, is used.

Retraction of the headstock is accomplished by means of weights enclosed in housing D. Retraction force necessary may be increased or decreased by the number of weights applied.

The setting of the tee block F is quite variable and is essential in positioning the head to leave only a minimum amount of clearance between the spindle nose and the guide bushing holder when the headstock is fully advanced.

The automatic stop and signaling device H shuts off the power when the bar is exhausted and a red signal light E indicates that re-stocking is necessary.

Panel I carries all of the electrical controls necessary for direct operation of the machine; start and stop button, coolant pump off and on switch, spindle low and high speed switch. Wheel J regulates the camshaft speed and is geared to dial on left which is accurately graduated in cycles per minute.

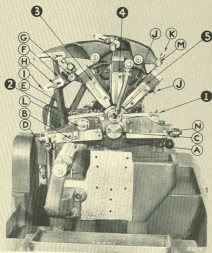
The wheel K controls the R.P.M. of the spindle and is geared to the dial below. The inside figures on dial are read when spindle switch is on low and the outside figures when switch is on high.



Photo 14

TOOL FRAME

- A—Rocker Arm
- B—Toe or Follower
- C—Rocker Pivot Stud
- D—Rocker Tension Spring
- E—Jack
- F—Rocking Lever
- G—Rocking Lever Pivot
- H—Jack Stud
- I—Ratio Adjusting Screw
- J—Tool Centering Screw
- K—Tool Depth Adjusting Screw
- L—Tool Lateral Adjusting Screw
- M—Locking Screw
- N—Locking Screw
- O—Stationary Holder



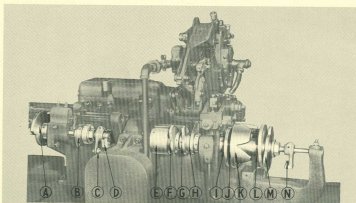
In illustration above, a front view of the tool frame is shown, identifying the tool positions, 1, 2, 3, 4 and 5 respectively. The rocker arm A carries the No. 1 and 2 tools and is actuated by cam action against the toe or follower B. The rocker pivots on hardened stud C. Spring D supplies necessary tension to hold toe against profile of rocker cam. Slide holders 3, 4, and 5 are working in dovetail ways. Cam action against jack E transmits motion to the slide through rocking lever F which pivots on stud G. The travel of the slide may be varied by changing the position of the point of force, jack stud H, in relation to the fulcrum point. Screw I is for accurately setting this position.

The set screws J are used for centering the cutting edge of the tool in its relation to the center of the bar. Micrometer screw K is for accurate setting of the tool in its relation to diameter being turned or operation being performed. Micrometer screw L is for accurate axial settings of the tool. Similar micrometer adjustments are on the 1, 3, 4, and 5 tool holders, but are in back of the tool frame shown plainly in illustration on opposite page.

Screw M is a locking screw to insure maintenance of the tool setting. Screw N is also used for locking the tool when correct setting has been accomplished, but its function is somewhat different from screw M as it locks directly on the gib instead of the micrometer screw.



CAMS & CAMSHAFT



- | | | |
|------------------------------------|------------------|-----------------------|
| A—Flat Feed Cam | F—5th Cam | K—Thread Cam |
| B—Counter Cam | G—4th Cam | L—Inner Index Cam |
| C—Closing Dog | H—3rd Cam | M—Outer Index Cam |
| D—Opening Dog | I—Rocker Arm Cam | N—Clutch Engaging Cam |
| E—Drum for Mounting of Slotter Cam | J—Drill Cam | |

Looking at the machine from the camshaft side (illustration above) all cams essential to operational movement of the machine and three-spindle attachment are shown mounted with correct spacing.

Flat feed cam A actuates the flat feed cam mechanism and is used for producing pieces up to 1 $\frac{1}{4}$ " long. When it is necessary to produce pieces over 1 $\frac{1}{4}$ " in length, the bell feed cam is used. This is shown mounted in place in illustration on page 21.

The counter cam, as its name implies, trips the lever of the countermeter once for every cycle of the camshaft, thereby registering the number of pieces produced over a definite period.

C and D are collet opening and closing dogs. Drum E is for mounting of bell type cam segments used to control lateral motion of slotter pickup arm.

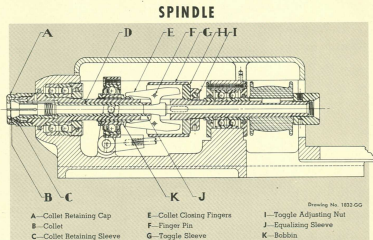
The cams F, G, and H control motions of the 3rd, 4th, and 5th tools. When the slotter attachment is in use, cam H is used to control radial movement of the pickup arm.

Cam I is the rocker arm cam and controls all motions of the No. 1 and 2 tools.

Drill cam J usually carries the lobes necessary to advance the No. 1 and 2 spindles of the three spindle attachment. Threading cam K controls the advance of the threading or 3rd spindle. Note: This cam mounts inside of drill cam which is chiefly to add to the flexibility of setup.

L and M are indexing cams essential to positioning of the attachment spindles relative to work piece. N is the attachment clutch engaging cam.





The headstock spindle construction is of the anti-friction type, precision built and dynamically balanced. It is capable of running and maintaining speeds up to 10,000 R. P. M. The front precision pre-load duplex radial thrust bearing hold the spindle rigid regardless of cutting pressure and eliminates any possibility of end play in the spindles. The pre-load of the bearing is correctly set at the factory. Should any adjustment become necessary due to natural wear, it is recommended that this should be undertaken only by a skilled mechanic. If trouble develops due to bearing wear, a new bearing with accurate preload determined at the factory can be ordered and installed at a minimum cost for both time and material. The rear bearings are also

of the radial thrust duplex type, but are used to carry the radial load only.

The toggle H carries the collet closing fingers, and movement of the bobbin E forces the fingers outward resulting in a levering action which advances the sleeves D and J forcing the collet retaining sleeve C against taper of the collet, closing same, thus gripping the bar. Tension is increased or decreased by advancing or retarding toggle by means of toggle nut I. To replace worn or broken fingers, the sleeve G is moved back uncovering pin F which holds the fingers in place. These pins may then be driven out with a small punch.

Note: The collet retaining cap is left hand thread.

LUBRICATION

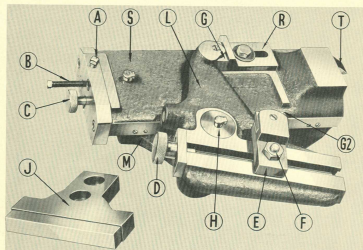
Lubrication of headstock and spindle bearings is provided from a reservoir in housing and is correctly metered to the bearings through visual drip oilers. These drip oilers are correctly set at factory, but should be checked quite frequently to see that proper regulation is maintained. Four to six drops per hour is quite sufficient for the spindle bearings, while the slides demand from 10 to 12 drops per hour.

Access to screws regulating the flow of oil to the respective bearings is obtained by removing the headstock cover. Here, also, will be found the vent closing screw, which acts as a lock on the drip oilers when closed and prevents excessive oil from accumulating on bearings when machine is not in operation. (It is of utmost importance that vent is open when operating.)

Note: For recommended lubricants, see chart on pages 118 and 119.



FLAT FEED CAM MECHANISM



1433 K

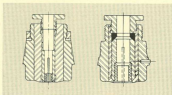
- | | | |
|---------------------------|---------------|---------------------------|
| A—Lock Nut | F—Lock Nut | L—Transverse Lever |
| B—Stop Screw | G—Shoe | M—Mounting or Bed Bracket |
| C—Spring Retaining Button | G2—Shoe | R—Slide Adjusting Block |
| D—Ratio Adjusting Screw | H—Lever Pivot | S—Slide |
| E—Shoe Carrier | J—Tee Block | T—Toe or Follower |

The flat feed cam mechanism is shown unmounted for illustrative purposes. This unit is mounted on the machine proper by means of bed bracket M. Slide S, adjusting block R, and follower T are integral parts; therefore, rise or throw of the cam against the follower will move the slide an equal distance. The headstock slide being at right angles to this slide necessitates transfer of motion. This is

accomplished through lever L. Shoe G is at a fixed distance from the fulcrum point of the lever, thereby force or throw of the shoe and the slide L will be correlative. The setting point of shoe G2 is variable. Movement will be in proportion to its setting from the fulcrum point. The tee block J is mounted on the headstock and rides against shoe G2, thus completing transfer of motion from cam to the headstock.



STATIONARY HOLDER



No. 1822-BA

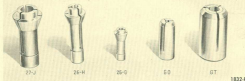
The stationary holder shown above is standard equipment on the automatic. It mounts in the center of the tool frame and is held securely by means of the large nut provided. The two sleeves are furnished to take care of the two size bushings used



1822-BAH

therein. The holder is slotted to allow adjustment of the sleeve in relation to the tools. The nut on end of this sleeve is for accurate setting of the guide bushing in relation to stock O. D.

COLLETS AND GUIDE BUSHINGS



Shown above are the various types of collets and guide bushings used in connection with the machine single spindle drilling attachment and three spindle attachment.

Collet type 27J is a spindle nose or headstock collet and used when operating on bar stock from 1/4" to 7/16" in diameter.

Collet 25H is also a spindle nose or headstock collet and is used on bar stock from 3/64" to 7/32" in diameter.

The small collet in center is 26G and is used in connection with the single spindle drilling attachment and No. 1 and No. 2 position, of the three spindle attachment.

Bushing 23 is a stationary type for bar stock 3/64" to 7/32" in diameter.

Bushing 27, also of the stationary type, is for stock from 1/4" to 7/16".

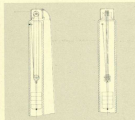
The stationary guide bushings are slotted for adjustment. This adjustment feature is not to be considered adjustable over a wide range. Reduction of .001" is all that is recommended. It can readily be understood that more than .001" adjustment would distort the hole. A carbide insert is brazed in the center of the bushing through which the guide hole is lapped. The guide hole is held to a tolerance of $\pm .0005$ ". Carbide inserted bushings are recommended for durability and long wear. Seizure or freezing of stock in the guide hole is practically eliminated. High speed steel bushings may be used satisfactorily when operating on material in the non-ferrous group, but their wearing qualities are not comparable to the carbide type.



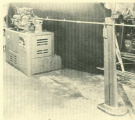
BAR FEED



1721-001



Drawing No. 1822-JJ



1721-002

Illustrations above present a rear view of the automatic with stock feeds in position. At left, pneumatic feed; right, weight feed.

The only purpose served by the feeds is to hold end of bar being machined against the cutoff tool

during opening of the collet, retraction of the headstock, and closing of the collet. After headstock collet re-grips bar, the stock feeds are inoperative while turning operations are being performed.

AUTOMATIC STOP MECHANISM

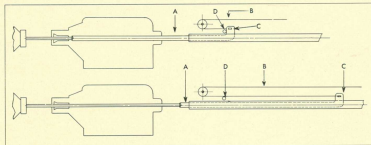


Fig. 1

Drawing No. 1822-II

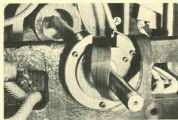
Following the bar is push rod A. Force necessary to keep push rod in contact with bar is obtained by means of weights and cord B. When the stock is nearly exhausted, as in figure 1, flag C of push rod moves lever D actuating a micro switch which breaks the circuit and stops the machine.

When setting stop rod D, the headstock should be fully retracted. The push rod is advanced in the spindle until it contacts the headstock collet, then withdrawn approximately $\frac{1}{16}$ " to $\frac{1}{8}$ ". Lever D is positioned and locked so that any additional forward movement of the push rod or flag will break the circuit and stop the machine.



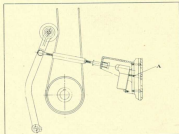
BELT REPLACEMENT

To fit a new spindle belt, remove the three outer hollow head cap screws that fasten the bracket as shown in illustration of main drive shaft. Place one loop of the belt in the opening (at A), then turn the bearing housing one revolution by means of a rod inserted in one of the cap screw holes. Belt is now in position inside of the bearing housing and is easily placed on pulleys.

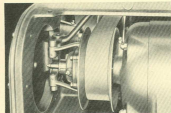


AUTOMATIC SAFETY SWITCH

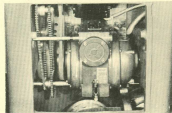
In case of accidental main spindle drive belt breakage, the machine is stopped automatically by a safety switch. This switch is operated by the idler pulley bracket. The turnbuckle which adjusts the belt tension is connected to the bracket housing the switch. When belt breaks, turnbuckle slides into bracket, forces button on switch and stops all motors.



Drawing No. 1832-4X



Reeves Drive



Graham Transmission

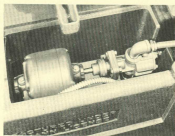


Photo 21

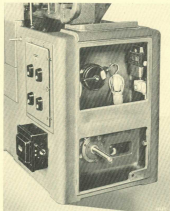
COOLANT PUMP

An angle type check valve holds a perpetual prime in the intake line. The coolant is returned to the sump through a screen which removes the larger chips. The sump pan, just below this screen, is provided with baffles. Smaller chips not removed by screen settle in this pan before coolant returns to coolant tank. A filter on the pump intake removes all foreign bodies that may still be present in coolant.

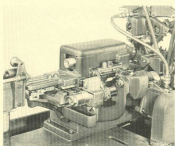
Filters should be removed and cleaned at regular intervals; time between cleaning is dependent upon the type of stock being worked and condition of coolant.



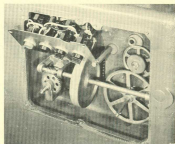
1721-4A



The illustration above shows the main disconnect switch and flash type motor control panel. The grills have been removed to show the fused circuit transformer and trouble lamp, also the main drive shaft upon which the pulleys are placed for operating the various attachments.



General Upper View, Comshaft Side¹⁰²²

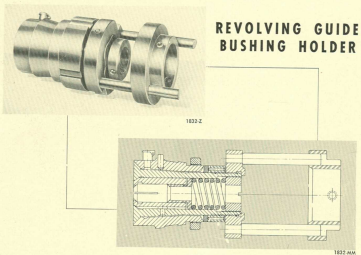
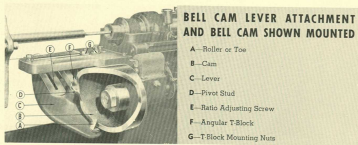


1022

Control Panel Box



Photo 22



The revolving guide bushing used mostly for square and hexagon stock is held in a sleeve. This sleeve rotates in a bronze bearing and thrust is carried on a hardened steel thrust bearing. Guide bushing automatically adjusts itself to inequalities of stock diameters by means of slots cut through guide hole which makes the bushing slightly adjust-

able. A coiled compression spring exerts pressure against the rear of the guide bushing. Pressure is regulated by an adjusting nut on the rear of the sleeve. Spring pressure pushes the guide bushing against the taper causing the sections between the slots to close the diameter of the hole, thus holding the stock in a centered position.



Photo 23

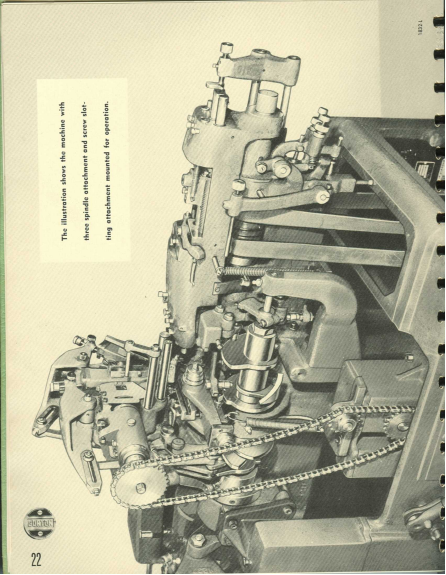
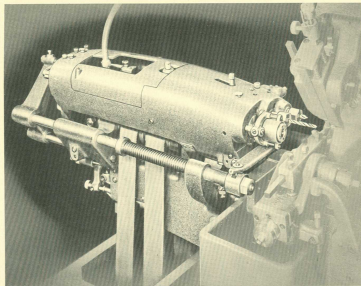


Photo 24

THREE SPINDLE ATTACHMENT



No. 1721-B

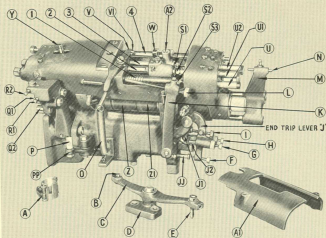
The Gorton three spindle attachment is quite versatile in its operation and its three work spindles are readily adaptable to spot centering, drilling, reaming, boring, tapping and threading.

In the above illustration, the attachment is shown mounted for spot centering, drilling and threading. The die being used above is a geometric EJS and

is easily adapted to the threading spindle. The flexible oil tube connecting the rear of threading spindle permits the flow of oil through the spindle and the die, assuring maximum tool or chaser life and maximum threading efficiency. The oil guard covering the die head controls excess coolant thrown off by rotating head.



GORTON 3-SPINDLE ATTACHMENT



1932 F

- A—Oil Guard Unit
- A1—Housing Cover
- A2—Oil Coupling Housing
- B—Advance Arm Engaging Pin
- C—Advance Bar Actuating Lever
- D—Mounting Bracket
- E—Cam Follower
- F—Trip Lever Adjusting Screw
- G—Clutch Tension Adjusting Screw
- H—Tension Adjusting Screw Idling Speed
- I—Friction Adjusting Screw Idling Speed
- J—End Trip Lever
- J1—Clutch Engaging Fork
- J2—Clutch Tension Spring
- JJ—Clutch Engaging Rod
- K—Vertical Trip Lever
- L—Vertical Trip Lever Adjusting Screw
- M—Advance Arm
- N—Advance Pin
- O—Retaining Spring
- P—Clutch Engaging Follower

- PP—Clutch Engaging Lever
- Q1—Roller Follower Governing No. 1 Spindle
- Q2—Roller Follower Governing No. 2 Spindle
- R1—Adjusting Nut for Centering No. 1 Spindle
- R2—Adjusting Nut for Centering No. 2 Spindle
- S1—Lead or Depth Adjusting Nut
- S2—Depth Adjusting Nut
- S3—Depth Adjusting Nut
- U—First Spindle Push Rod
- U1—Second Spindle Push Rod
- U2—Third Spindle Push Rod
- V—Horizontal Trip Lever
- V1—Horizontal Trip Lever Pin
- W—Guide Shaft Trip Block
- Y—Gear Shift Pin
- Z—Fiber Pulley
- Z1—Fiber Pulley
 - 1—First Spindle
 - 2—Second Spindle
 - 3—Third Spindle
 - 4—Guide Shaft



MOUNTING GORTON 3-SPINDLE ATTACHMENT

Mounting is relatively simple. First make sure the machined surfaces of attachment and machine are clean and free of any foreign particles. Then place attachment on pad in front of tool frame and secure by means of the four cap screws provided. Screw the three shoulder studs holding jack screws (provided) into the tapped holes in the face of the mounting pad. When in position, there will be a jack screw on each side of the attachment pad and one directly below the bottom. These are the centering screws for the spindles.

To center the attachment in relation to workpiece, one of the most simple methods is to place small indicator in either the No. 1 or No. 2 spindle position

in such a manner that it can be manually revolved around a small dowel pin placed in the guide bushing. The jack screws are adjusted to obtain proper alignment.

Next, place two pulleys on drive shaft. (Three are furnished—place one of larger pulleys on first, then always follow with smallest.) The two larger pulleys determine the speed of die or tap. The smallest one, always used, governs the speed for backing off or idling.

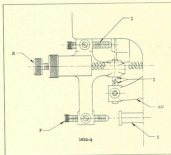
Bells are placed on their respective pulleys. Idler pulley and bracket are mounted, completing the mounting of the attachment proper.

OPERATIONAL CHARACTERISTICS OF 3-SPINDLE ATTACHMENT

(Refer to illustration on opposite page)

The method of differential threading or where the die must run at a speed above that of the work piece to thread on and at a proportionately lower speed for backing off necessitates the use of the two pulleys Z and Z1. These fiber pulleys are the outer halves of a double cone clutch and revolve idly upon a double taper steel clutch shaft. Ratio of their speeds is in proportion to the size of the pulleys used on the main drive shaft. Z must run at a speed sufficient to drive the threading spindle at a speed in excess of that of the machine work spindle. Z1 must run at a proportionately lower speed to facilitate backing off of the die head or tap. The transmission of power to the spindle from the pulley is accomplished by the steel double tapered clutch shaft and intermediate gears.

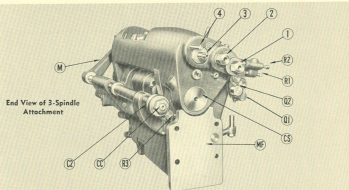
Engagement of the overspeed clutch is controlled by the cam action against the follower P of lever PP. Action of this lever moves rod II into adjusting screw F, thereby lifting lever J a sufficient amount to allow pawls to latch, by holding lever in fixed position until automatically tripped. (See sketch



above.) Here pawls T are shown latched, holding the overspeed clutch in an engaged position. Movement of lever X will move arms KU as they are integral parts, thereby unlocking pawls and disengaging clutch from threading speed and engages the opposite clutch for the lower or backing off speed.



GORTON 3-SPINDLE ATTACHMENT CONTINUED



1832 G

- 1—First Spindle with Collet Retaining Cap in Place
- 2—Second Spindle with Collet Retaining Cap in Place
- 3—Third or Thread Spindle
- 4—Guide Shaft
- R2—Adjusting Nut for Centering No. 1 Spindle
- R1—Adjusting Nut for Centering No. 2 Spindle
- Q2—Roller for Inner Index Cam of No. 1 Spindle

- Q1—Roller for Outer Index Cam of No. 2 Spindle
- C5—Pivot Shaft
- MF—Machine Surface for Mounting of Attachment
- R3—Stop Screw Used to Control Centering of 3rd Spindle
- CC—Advance Bar
- C2—Advance Collar
- M—Advance Arm

The overspeed clutch engaging fork JI is actuated by spring pressure exerted by the lifting of lever J through spring J2. Thus clutch is held in an engaged position under spring tension. Tension may be increased or decreased by means of knurled screw G. Release of the clutch is controlled by trip block W. This block is mounted on guide shaft No. 4. The guide shaft and threading spindle are integral parts; therefore, any movement of the threading spindle will move the guide shaft a corresponding amount. Block W is set an approximate distance from the pin V1 of the upper trip lever to

allow the spindle to advance correct distance to produce desired number of threads. When the clutch is engaged, lever K will be in such a position that the tip of screw L will be almost touching lever V. When spindle is advanced, block W hits pin V1 of lever V causing lever K to raise. This unlatches the pawls, disengaging clutch from the threading speed, and by spring pressure engaging the opposite or lower speed clutch.

Instead of spindles 1, 2, and 3 is accomplished by cam action against the common lever C. This lever is mounted on bed of machine proper by means of



GORTON 3-SPINDLE ATTACHMENT CONTINUED

bracket D. E is the cam riding piece and pin B engages the slotted collar of the advanced bar. Advance arm M is a fixed member of the advance bar and any forward travel of the bar will, therefore, bring advance pin N in contact with shoe of push rod bar of the spindle in work position thus advancing spindle toward the work piece.

Positioning of the 1 and 2 spindles in relation to the work piece is controlled by cam action against rollers Q1 and Q2; adjusting nuts R1 and R2 are for very accurate centering of the spindle in relation to the work. The positioning of the threading or third spindle is accomplished by stop screw R3. (See End View, page 4.)

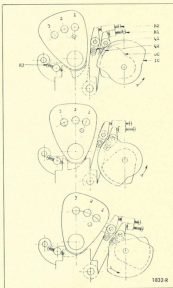
Spindle 1 is a stationary spindle and no provision is made for revolving it at any time.

Spindle 2 may be used as a stationary or revolving type spindle by means of gear disengage pin Y. (See illustration, page 24.)

Spindle 3 is designed primarily for threading and tapping. It is a live spindle and the only means of using it as a stationary type is to remove the belts from driving pulleys Z and Z1.

Spring O facilitates indexing of the head toward the camshaft, also insures proper contact of rollers Q1 and Q2 against profile of their respective cams. Movement of the head away from the camshaft is a positive cam action.

The Schematic Views (at right) better illustrate the independent control over the centering of each spindle individually although spindles are housed as a common unit.



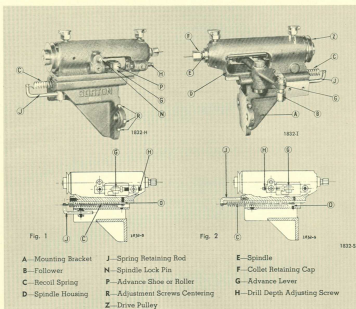
In Fig. 1 (above) the head is shown in the No. 1 position. Roller Q2 is riding on the maximum radius of its respective cam OC (outer cam). Roller Q1 is idle and will remain so as long as roller Q2 rides the maximum radius of cam OC.

In Fig. 2 (above) the maximum radius of cam OC has passed, allowing roller Q1 to fall to the maximum radius of the cam IC (inner cam), thereby indexing head to the middle position. Roller Q2 is idle.

In Fig. 3 (above) both the rollers Q1 and Q2 are in idle position away from the profiles of their respective cams and the machined pad of the head proper is rested against stop screw R3.



SINGLE SPINDLE DRILLING ATTACHMENT



The single spindle attachment is very simple in design and differs in functional operation from the drill spindles of the three spindle attachment in that the spindle and housing move as a unit, whereas, in the latter, the spindle itself moves and the housing is stationary.

The single spindle housing mounts in dovetail ways and operates either by spring pressure or positive cam action.

In schematic drawing, figure 1 shows the arrangement for positive drilling. Spring C is retained by rod J in the mounting bracket. The opposite end of the spring is attached to stud O which is mounted in the spindle housing proper, thus spring applies the

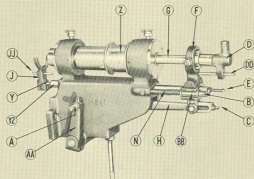
retraction force and movement of lever G applies forward motion or drilling force. Movement of lever G is transmitted directly to the slide through adjusting screw H. Positioning of this screw will determine the drilling depth.

In figure 2, the arrangement is shown for sensitive drilling. Here retaining spring I is placed in spindle housing and stud O is placed in mounting bracket, thus tension of spring C will facilitate forward motion of the slide. The screw H is placed in pad provided at the rear of lever G and positive action against lever applies retraction force.

Note: The cam construction for the latter is just opposite that of positive drilling.



SINGLE SPINDLE THREADING ATTACHMENT



1832-2

- | | | |
|-----------------------------|---------------------------|------------------------|
| A—Roller Cam Follower | DD—Oil Coupling Guide Pin | JJ—Die Close Pin |
| AA—Spindle Advance Lever | E—Trip Adjusting Screw | N—Return Spring |
| B—Guide Bar | F—Spindle Yoke | Y—Fork Advance Rod |
| BB—Yoke Sleeve | G—Spindle | Y2—Fork Advance Sleeve |
| C—Thread Lead Adjusting Nut | H—Advance Bar | Z—Drive Pulley |
| D—Oil Coupling | J—Die Open Fork | |

The method of threading with this attachment is comparable to the threading spindle of the Three-Spindle head. Since self opening die beads are used with this attachment, the spindle revolves constantly at a speed somewhat greater than that of the work piece or head-stock spindle.

The die is advanced and the leading on of the die is accomplished by cam action against the roller follower of the lever AA. Movement of this lever advances the spindle and complete yoke assembly through connecting bar H.

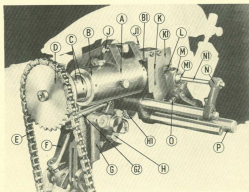
After the die has been started on the work piece, the lead continues to carry the spindle and yoke assembly forward, until screw E hits stop pin, terminating forward travel of the fork advance rod and the opening fork. The die continues to travel forward until enough pressure is exerted to open the die head.

Opening of the die allows the return spring N to retract the spindle and its assembly. Upon return of assembly, cam action against lever AA forces closing pin JJ against casting, thereby closing die head and completing the operation.

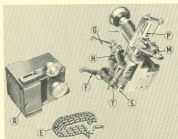


Photo 31

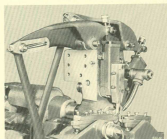
SLOTING ATTACHMENT



No. 1882-A



No. 1832-88



No. 1832-84

- A—Spindle Housing
- B—Adjusting Collar
- B1—Adjusting Collar
- C—Retaining Nut Spindle
- D—Sprocket Gear
- E—Chain
- F—Transverse Lever
- G—Segment Gear Jack

- G2—Recoil Spring
- H—Segment Gear
- H1—Gear
- J—Adjusting Screw
- J1—Adjusting Screw
- K—Saw
- K1—Saw Guard
- M—Pickup Arm

- M1—Pickup Bushing
- N—Knockout Rod Bracket
- N1—Knockout Rod
- O—Stop Screw
- P—Transverse Bar
- R—Gear Box
- S—Collar
- T—Recoil Spring



Photo 32

Illustration 1832-M shows No. 3 tool holder and bracket assembly removed for mounting of the slotting attachment. Illustration 1882-A shows slower mounted ready for operation.

The spindle housing unit A is independent of the main bracket and is mounted on an arbor in such a manner that it may be tilted to allow for centering of cutter spindle in relation to pieces being slotted and is accomplished by means of screws J and J1.

Adjustment of the saw in relation to the center of the piece is accomplished by unlocking either collar B or B1 and drawing up on the other collar thereby advancing or retarding the spindle.

The upward radial movement of the transfer or pickup arm is controlled by cam action against cam C. Its motion is transmitted to segment gear H, actuating gear H1 which is keyed to shaft P upon which the pickup arm is mounted.

The lateral travel of shaft P which governs the pickup, transfer of piece and slotting, is controlled by a drum or bell cam. Action of this cam through lever F is better seen in illustration 1832-8B. Here the transmitting end of the lever is shown engaged in slotted collar S, thus insuring positive action for the slotting operation.

Spring T is for retraction of the lever and to hold follower against profile of cam.

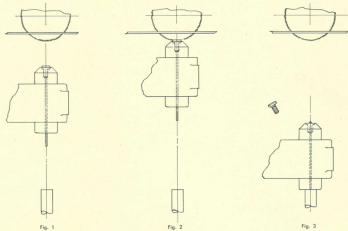


Fig. 1

Fig. 2

Fig. 3

Drawing No. 1832-L1

In schematic drawing above, figure 1 shows the approximate position of arm at completion of pick up; figure 2 shows position of arm during slotting operation; and figure 3 shows relative position of pick up arm at the knock out of piece.



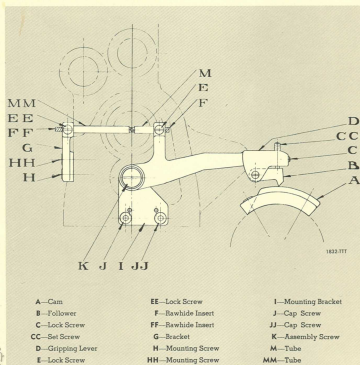
STOCK GRIPPER ATTACHMENT

The stock gripper is used to hold face of bar against the cut-off tool during the opening of collet, retraction of headstock, and closing of collet. Its use is recommended where very small diameter bars are to be used (usually under 1/16") and weight or pneumatic feed would not be practical.

Throw of the cam "A", causes the riding piece or follower "B" to lift, raising lever "D". Movement

of this lever carries tube "M" toward the stock, creating a vise-like action between "MM" and "M". Tubes "MM" and "M" are fitted with rawhide inserts "F" and "FF" to provide maximum friction and minimum wear.

Tension is increased by raising or lowering screw "CC."



COMMON PRACTICES AND INFORMATION PERTINENT TO CAM AND TOOL LAYOUT

CAM AND TOOL LAYOUT

Designing of tools and cams for the GORTON Automatic is relatively simple in procedure and calculation, but it is of utmost importance that the designer understands the machine and its operating characteristics thoroughly to make satisfactory

layouts and to obtain maximum efficiency from the machine.

This treatise is intended to give designer common uses of tools and tool positions and general practices that should be followed.

TOOL POSITIONS

On the GORTON Automatic there are five tool positions. These are numbered clockwise 1 to 5 respectively. (See illustration page 13.) This clockwise numbering is common to all Swiss Type Automatics.

No. 1 position is a rocker arm tool and should be used for front shoulder turning where close tolerances and good finishes are to be maintained. It is seldom used for back of shoulder work or cutoff, as this tool is held in position by spring action, and toe, or follower, merely acts as a stop against cam profile. Infeed or plunge cutting would necessitate dependency on this spring to overcome tool pressure that would be set up in such operation, and is in most cases impractical. Ratio of this tool is 3:1 permanently.

No. 2 tool position is recommended for infeed or plunge cutting and back of shoulder turning, as

forward motion is by positive cam action. The spring in this case merely holds the toe or follower against profile of cam, retracting the tool at completion of operation. Ratio of this tool is 3:1 permanently.

Nos. 3, 4 and 5 tool positions are slide tools. These holders slide in dovetail ways and are not as rigid as the rocker arm tools; therefore, they are not recommended for close tolerance turning. Their primary uses are chamfering, undercutting, spot centering, cutting off, thread rolling, knurling, etc.

Quite often one of these tools will be required to operate in unison with a rocker arm tool. Caution should be exercised when laying out the cams to prevent the possibility of tool interference.

Ratio of these tools is variable from 2:1 to 3:1.

NOTE: The term "ratio" is the amount of throw on cam in relation to the amount of tool travel.

FEED MECHANISM STOP SCREW

The stop screw on the feed mechanism is used to increase or decrease travel of the headstock within limits of total throw of the feed cam.

The chief purpose of the stop screw is to compensate for varying widths of cutoff tool in relation to over-all length of the turned piece. Without this adjustment it would be necessary to use a tool of exact width to produce accurate lengths.

Although it is not difficult to grind a tool to correct width at the original grind, regrinding the cutting edge will be necessary. Each grind reduces width of the tool a few thousandths and the length of piece increases proportionately. The stop screw is adjusted to compensate for this change in length.

Often the operator finds that the calculated width is too narrow to withstand cutting pressure applied.



and unless allowance has been made at time of layout he dare not increase the width since it would shorten over-all length of piece. Therefore, it is a common and recommended practice to make what is generally termed a stop screw allowance. The allowance commonly used is from .015" to .020", although any amount may be used. This allowance is figured in the over-all lengths on the first turned length. This will allow the operator to increase the

width of tool in proportion to allowance made and adds greatly to the flexibility of the setup.

The use of the stop screw is indispensable where pickup sets of cams are used, or where a number of screws, pins, or similar parts of various lengths are to be produced on the same set of cams.

Stop screws cannot be used where the first operation on a part is in the form of a generation.

ROCKER ARM STOP SCREW

The stop screw on the rocker arm is used when two or more diameters are being turned by the No. 1 tool. It can only be used when the No. 1 tool is turning the smallest diameter and the rocker arm follower is at the minimum radius of the cam.

To provide for the use of this screw it is necessary to make proper allowance on the minimum cam radius (.005" to .010" is commonly used). This radius is cut slightly smaller than would normally be required for turning the smallest diameter. By

adjusting the stop screw the rocker cam follower is moved away from minimum cam radius providing adjustment of this turned diameter without depending upon the cam.

Upon completion of turning of the smallest diameter, cam again contacts the follower positioning tool for turning the second diameter, which is adjusted by micrometer screw on the slide. The first turned diameter can now be adjusted without disturbing the setting of the second turned diameter.

MEAN RADIUS OF ROCKER ARM CAM

The mean cam radius is the point on the cam which positions the rocker arm to allow maximum clearance for passage of the 3rd, 4th, and 5th tools, as well as providing sufficient throw between min-

imum and maximum diameter of cam to move No. 1 and No. 2 tools from their neutral positions through maximum tool travel required.

TOOL CLEARANCE

When the rocker arm follower is resting on mean radius of cam, or at its neutral position, there must be a definite amount of clearance between the rocker arm tools and the outside diameter of stock. This clearance is variable and determined from size of bar being machined. It is not only necessary for the advance of bar where unturned portion exists, but also to allow passage of frame tools.

It is readily seen that if the same tool clearance is permitted on very small bars as on those of maximum size, it would necessitate grinding tool clearance angles of frame tools to such a degree that they would become delicate and impractical.

See page 96 for established table of tool clearances for various bar sizes.

GENERATING OF DEVELOPMENT OF IRREGULAR CONTOUR

Generating a curved or irregular diameter or a part is merely a development where coordination

between movement of tool and traverse of stock produces the desired shape to specified dimensions.



UNPRODUCTIVE TIME

In making a layout, any operation where no actual cutting is being done is considered unproductive time and is figured in degrees. In most cases, unproductive time is based on amount of throw of cam to complete respective operations and number of degrees determined from established tables on pages 78 to 87.

The retraction or advance of headstock where no cutting is being done on bar, is unproductive time and can be determined from tables on pages 80 and 81 respectively.

"Out" of any tool at completion of operation is, in the majority of cases, unproductive time and is determined from table on pages 78 and 79.

"In" of tools is mostly productive time and figured as such, but when tool is brought from neutral position to O. D. of bar, or to diameter already

turned, it is figured as unproductive time and is found in table on pages 78 and 79.

There are a number of operations of unproductive nature and time is established from practical and common use.

SAFETY: It is general practice to allow a 2 degree safety at start and finish of a turning operation.

There are three reasons for its use:

1. Simplifies cam manufacture.
2. Simplifies alignment of cams at setup.
3. Gives tools time to dwell at completion of cut which is necessary to obtain clean turning and square shoulders.

OPENING OF COLLET: 10 degrees are allowed for this operation immediately after the cutoff.

CLOSING OF COLLET: 17 degrees are allowed for this operation following retraction of headstock.

UNPRODUCTIVE GENERATION OR TURNING

Where a generation, or turning, normally figured productively takes only a few degrees on profile of cams, it should be considered an unproductive operation. 5 to 10 degrees should be allowed for

an operation of this kind. This is to aid and simplify cam manufacture, also to obtain accurate generation. See page 37.

PRODUCTIVE TIME

When any cutting is being done on the bar, it is considered productive time and can only be calculated after total of unproductive degrees and

total number of productive revolutions of bar have been obtained.

EXPLANATION OF VARIOUS TERMS

"IN" OF TOOL OR INFEDS: Any radial motion of tool in or toward center of bar.

"OUT" TOOL: Radial motion of tool away from bar, usually at completion of an operation.

RECOIL: Retraction of headstock or drill spindles.

GENERATING OR GENERATION: Any turning such as tapers, curves or any irregular shapes not parallel with bar.

DWELL: Any portion of cam where profile is concentric with bore or center of cam.

RISE AND FALL: Distance from one dwell to another, often referred to as throw of cam.

FOLLOWER, TOE OR BEAK: Piece contacting or riding cam.

RATIO: Amount of headstock or tool travel in relation to the throw on cam.

ADVANCE HEAD: Unproductive advance of headstock where no metal is being removed from bar.



DOUBLE CHUCKING

The production of parts over the recommended length when using the flat feed cam, or the bell feed cam, is in many cases quite practical by use of the double chucking arrangement.

Whether or not its use is practical is difficult to say since this depends largely on the amount of time required for all operations other than those where headstock is being advanced. If the number of degrees required for these operations is too great it will cause the rises on the feed cam to become too sharp, thereby creating a binding action against the follower. The maximum rise per degree should not exceed .018" for smooth and efficient operation.

The only means of determining when the arrangement may be used is to make a layout of the part in question. Sum up total degrees for all operations other than those required to advance the headstock. This number subtracted from 360° equals time (degree) for total advance of head. Total length

of part divided by degrees for total advance of head equals rise per degree.

EXAMPLE:

Spot and drill	20 degrees
Unproductive tool advance	8 degrees
Openings of Collet	20 degrees
Recoils of head	57 degrees
Closings of collet	34 degrees
Safetyes	10 degrees
Prod. "in" of tools	35 degrees
	184 degrees

$360^{\circ} - 184^{\circ} = 176^{\circ}$ Advancement of head
5.00" length of part

$5.0^{\circ} \div 176 = .028$ rise per degree which is above recommended figure and would be entirely too sharp. Job would not be practical.

VARIABLE RATIO

In designing cams there is no great difference in the ratio used on frame tools. The main intent is to add flexibility to the machine when pickup sets of cams are used. In making a layout either a 3:1 or 2:1 ratio, or anything therein, can be used successfully in most cases.

One of the chief advantages of the variable ratio on the feed mechanism is the proportionate reduction of error in shoulder lengths resulting from inaccuracies of the feed cam.

It is readily seen that an error of .006" on cam would only result in a .002" error on piece when using a 3:1 ratio, or .003" when using a 2:1 ratio. When a 1:1 ratio is used, any error on cam will result in corresponding error on piece.

The variable ratio for the feed cam is also indispensable where pickup sets of cams are used and adds greatly to the flexibility of the machine.

The ratio to be used is determined by the sum of the length of the piece plus width of cutoff tool

and stop screw dimension. Where sum of length of piece, width of cutoff, and any stop screw allowance is one-third or less than total maximum throw of feed cam, a ratio of 3:1 may be used.

Where sum of length of piece, width of cutoff, and any stop screw allowance is one-half or less than the total maximum throw of feed cam, a 2:1 ratio may be used.

Where sum of length of piece, width of cutoff, and any stop screw allowance is greater than one-half maximum throw of feed cam, a ratio of 1:1 must be used.

The above recommendation should be followed to determine ratio on pieces where close tolerances exist on length and shoulder dimension, but when production is of major importance and tolerance and quality are of a secondary nature, a 1:1 ratio should be used, regardless of length. When using a 1:1 ratio, a minimum amount of rise and fall are necessary on cam, allowing minimum of unproductive time.



COMPUTATION OF REVOLUTIONS NECESSARY WHEN THREADING

When using pulley 5-11/16" in dia., a ratio of 5:4 exists between 3rd or threading spindle speed and headstock spindle speed, or in other words, the threading spindle turns 5 times to 4 turns of the headstock spindle. Therefore, the ratio between threading speed and headstock spindle speed is 1:4, which equals four turns of the spindle to produce one thread. The number of revolutions of headstock spindle necessary for any given number of threads equals number of threads multiplied by ratio existing between threading speed and headstock spindle speed.

EXAMPLE:

Required threads = 30
 $30 \times 4 = 120$ revolutions of headstock spindle

When using pulley 5/4" in dia., the ratio between spindles is 5:2. Ratio between threading speed and headstock spindle speed is 1:2.5.

EXAMPLE:

Required threads = 30
 $30 \times 2.5 = 75$ revolutions of headstock spindle
 When backing off, the ratio between spindles is 1:2, and the threading off speed is also 1:2.

Number of revolutions necessary to back off equal threads cut multiplied by ratio.

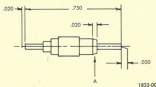
EXAMPLE:

30 numbers of threads cut
 $30 \times 2 = 60$ revolutions for backing off

PRODUCTIVE VERSUS UNPRODUCTIVE GENERATION OR TURNING

A generation or turning should be made an unproductive operation when, if figured at normal feeds, the time necessary (represented by degrees) would be so few as to be impractical for cam maker to develop correct curve on cam profile. The experienced designer by making a slight study of piece can usually tell if such operation will be unproductive or productive, but it may be necessary for the inexperienced designer to rough calculate total revolutions to complete pieces and thereby determine approximately the number of revolutions per degree, from which number of degrees for operation in question can be determined.

In illustration below, the angle A must be determined prior to layout as to whether it will be an unproductive or productive generation.



EXAMPLE: Overall length plus width of cutoff

tool and stop screw allowance, if any, equals complete length to be turned.

$$.750 + .020 + .050 = .820$$

complete turned length

Complete length to be turned divided by average feed to be used equals approximate total of revolutions for turning.

$$.820 \div .0007 = 1.171$$

revolutions for complete length

Radius of bar divided by infeed to be used for plunge cut and cutoff equals number of revolutions necessary for this operation.

$$.187 \div .0003 = 623 \text{ revolutions}$$

Revolutions to complete turnings plus revolutions for infeeds and cutoff equal total revolutions for all productive operations.

$$1.171 + 623 = 1.174$$

total productive revolutions

UNPRODUCTIVE TIME:

Opening of Collet	10'
Closing of Collet	17'
Retract Head (approx.)	20'
Safety (approx.)	20'
Unproductive Tool Movements (approx.)	20'
Total Unproductive Time (approx.)	87'



Unproductive time subtracted from number of degrees in one cycle, equals number of productive degrees.

$$360^\circ - 87^\circ = 273 \text{ productive degrees}$$

Total productive revolutions divided by total productive degrees equals approximate revolutions per degree.

$$1.793 \div 273 = 6.5 \text{ (use 6)}$$

The generation in question has a turning length of .020 and the number of revolutions necessary to

complete turning equals length divided by feed.

$$.020 \div .0007 = 28 \text{ revolutions}$$

Revolutions divided by revolutions per degree equals degrees necessary to complete operation.

$$28 \div 6 = 4.6 \text{ degrees}$$

This would not allow enough surface on profile of cam for cam maker; therefore, operation is made unproductive and 8 and 12 degrees should be used. 10 would be quite satisfactory in this case and time will be entered in unproductive column on layout sheet.

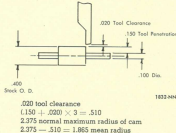
MEAN RADIUS OF ROCKER ARM

On bars 7/32" diameter or less a mean radius of 1.800 is always used, but on bars of 1/4" diameter and over, the mean radius must be calculated and is determined thus: Penetration of the No. 2 tool into bar plus tool clearance multiplied by ratio. This total subtracted from normal maximum cam radius equals mean radius to be used.

EXAMPLE:

In sketch, the stock size is .400, smallest diameter turned .100.

$$\frac{.400 - .100}{2} = .150 \text{ tool penetration.}$$



DRILLING

Drilling on the GORTON Automatic is very little different from the conventional type automatic with the exception that due to work commonly handled on these machines being relatively small, the drills are often delicate; therefore, it is recommended that three times the diameter of the drill should be the maximum advance of drill into work without a recoil to allow for chip removal and that nine to ten times the diameter is approximately the total drill depth practical.

For very fine drilling, the single spindle drilling attachment should be used utilizing the sensitive drilling action. When drilling in such a manner, a drill is advanced into the work by spring action, thus allowing drill to advance only as fast as its cutting edges will permit. If drill becomes dull or encounters an unusually hard spot in the bar, it will merely stop cutting, thereby resulting in quite a saving in drills and scrap.

In Fig. 1, page 39, is shown the cam construction for positive drilling. Here rise on the cam moved the drill into the workpiece to a corresponding depth. This type construction is used on either the single spindle drill or the three spindle attachment.

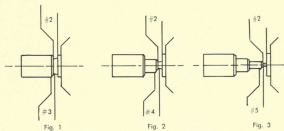
In Fig. 2 is shown cam construction for sensitive drilling. Here fall on the cam allows the spring to advance the drill into the bar and the small rises shown on the cam are for the recoil of drill for chip clearings.

Fig. 3 shows the relative position of the spindles of the three spindle attachment with their normal clearance dimensions.

Fig. 4 shows specifications pertinent to adaptation of miscellaneous tools such as taps, dies, reamers, etc. of the third or threading spindle of the three spindle attachment.

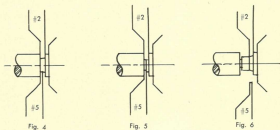


CHAMFERING AND UNDER-CUTTING



Figures 1, 2 and 3 show chamfering back of the shoulder with No. 3, No. 4 and No. 5 frame tools. When turning all diameters back of the shoulder and cutting off with No. 2 rocker tool all three of the frame tools become available for the chamfering operations, provided

they are not utilized in operations on diameters for front of the shoulder. A special form tool can be employed to produce several chamfers at one time. A tool of this kind can be used successfully either in front or back of the shoulder and in some instances on both sides.



1853-17



MULTIPLE UNDERCUTTING OF MULTIPLE DIAMETERS

The method of undercutting a limited number of diameters is very similar to that of chamfering and can be used equally successfully in front or back of the O. D. of the stock.

On those diameters in front, there is, as a general rule, plenty of time to undercut the diameter during the outing of the No. 1 tool to the next diameter TO BE turned. But to undercut the diameter back of the O. D., consideration must be given to the fact that the No. 2 tool must be advanced to

the diameter TO BE undercut before diameter CAN BE undercut. Since after No. 2 tool reaches diameter, the general practice is to allow 2 degrees safety before start of turning operation, in which time undercutting tool would have to complete its operation. Therefore, it is necessary to allow a larger safety which can be determined by feed used and depth of undercut.

See page 40, Fig. 4, 5, and 6.

TURNING AND CUTTING OFF WITH ONE TOOL

Due to similarity of the back turning tool and the cutoff tool, it is not unusual to turn a number of diameters in back of the shoulder and utilize the same tool for parting the piece from the bar.

Caution should be exercised as to the feasibility of this practice. Its use is recommended only when

the material used is of a free machining type with tolerance and finish secondary in importance.

When a tool is used in this manner, all shoulder lengths in back of O. D. are definitely controlled by the throw of cam, thereby detracting from the flexibility of the machine.

MALE CENTERS

The practice of producing pieces with male centers on both ends is very common and is very easily accomplished.

The method of producing centers is best illustrated in Fig. 1, 2, and 3 on page 42.

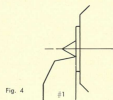
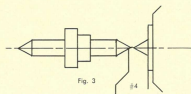
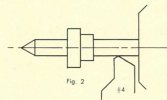
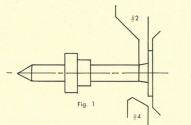
In Fig. 1, page 42, turning of the piece is complete with the exception of the rear center; note the last turning is well over the required length. This is done to spare cutoff tool the necessity of removing any metal except that below diameter to be cut through. It is readily seen that the cutting point of

cutoff tool is considerably out of position; therefore, after outing the No. 2 turning tool, headstock is retracted correct amount to bring the cutting point in line with O. D. of the male center (Fig. 2). The next operation is a generated cutoff in which the stock is advanced proportionately as tool is advanced into the diameter to be parted; in Fig. 3 the tool is shown at completion of the operation and if the correct angle has been ground on cutting side of the cutoff tool, the correct center will be left on face of bar which will be the front center of the next piece as shown in Fig. 4.



Photo 43

MALE CENTERING



1833 011



KNURLING

Three methods most commonly employed are single roll contact knurling, double roll contact knurling, and straddle knurling.

Single roll contact knurling (see Fig. 1, page 44) may be employed where bars or diameter to be knurled are of sufficient size to withstand the necessary pressure to obtain a full knurl.

Double roll contact knurling (see Fig. 2) is preferable to single roll knurling, mainly that rolls of standard widths may be used for all jobs regardless of length—as length is obtained by advancing the bar. Left and right rolls may be used when diamond knurling. Smaller diameters can be knurled as pres-

sure is equalized between the two points of contact. This method is not recommended for bars or diameters under $5/32"$.

Straddle knurling (see Fig. 3) presents the best condition for knurling small diameters and is recommended for bars or diameters from $1/32"$ to $5/32"$. Here, rolls straddle bar and very little pressure is exerted against diameter to be knurled. Length is obtained by advancement of the bar similar to turning operation.

Due to size and shape of double roll tool, it is necessary that allowance be made for tool clearances. (See recommended clearance and tool setting, Fig. 2).

EXTERNAL THREADING OF SMALL PIECES

When threading small screws or pieces which are not of sufficient length to clear tools, the headstock is advanced so that portion to be threaded is well beyond tool to prevent the possibility of die head running into the tool bits, Fig. 1, page 44. Dwell

on feed cam holds headstock in this position during threading period (Fig. 2). The time is computed as explained on page 37. After threading, headstock is retracted correct distance to continue unfinished turnings or cutting off of the piece, Fig. 3, page 44.

LAYOUT FOR SEMI-AUTOMATIC OPERATION

The procedure and calculation for semi-automatic work is very little different than full-automatic operation. With the exception no time allowance is necessary for the opening and closing of the collet as this is done manually. Likewise, no time is necessary for cutting off since the bar must be cut up into pieces of correct blank length. Since the cutoff tool, which normally acts as a stop, is not used, it is necessary to have some means of uniformly positioning the blanks. The most common method is to use a blank tool bit in one of the overhead positions. A lever can then be placed in hole provided on the overhead arm and the tool or stop is operated manually.

EXAMPLE (Plan of Operation):

1. Stop is brought into position.
2. Insert blank.
3. Close collet.
4. Out stop.
5. Start (automatic operation).
6. Turn, etc. (automatic operation).
7. Retraction of head (automatic operation).
8. Stop (automatic operation).
9. Open collet.

NOTE: After the retraction of head, $5'$ to $10'$ should be allowed for operator to throw machine out of automatic control.



Photo 45

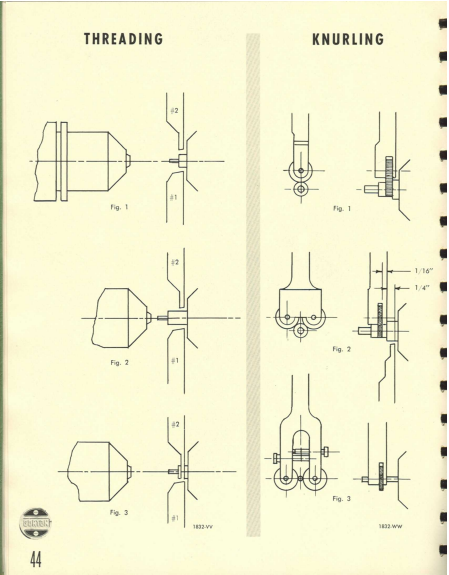
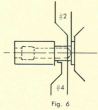


Photo 46

BORING AND SHOULDER UNDER-CUTTING



1822.XX



BORING AND SHOULDER UNDER - CUTTING

In Fig. 1, page 45, is shown a piece with a relief in the drilled hole, and under cut back of shoulder. This is accomplished as shown in Figs. 2 to 8.

In Fig. 2, the boring tool is shown in position prior to the advancement of bar. In Fig. 3, bar has been advanced the correct amount to bring cutting edge of the boring tool in position relative to start of the relief. The following operation will be to advance the boring tool radially into the I. D. of hole to desired depth of the relief. Fig. 4. The headstock then advances bar the correct distance to procure the desired length of relief. Fig. 5, after

which tool is withdrawn radially the amount necessary to clear I. D. of drilled hole. The headstock is then retracted to permit withdrawal of the boring tool to its normal position and completes the operation.

In Fig. 6, the turning on rear diameter has been completed and the under cutting tool has been advanced to the turned diameter. In Fig. 7, the headstock is retracted productively producing the under cut as shown. In Fig. 8, the headstock is advanced to its position for cutoff and the retraction of under cutting tool.

MULTIPLE CHAMFERING

While this method is occasionally employed, it is not recommended, due to the inflexibility of the set-up. Caution should be exercised in using this method.

Clean, accurate chamfering may be accomplished on a limited number of shoulders with a single point tool, eliminating complicated and expensive form tools.

The method employed for such operations is very similar to that of turning a number of diameters with one tool, except the tool is withdrawn to its neutral position after each operation. This method is most successful for those shoulders in back of O. D. including the O. D. Here the No. 2 tool is advanced radially into the stock to diameter to be turned, which allows sufficient time for the chamfer tool to chamfer corner formed by the plunge cut of the No. 2 tool, and withdraw before the bar is advanced for turning.

When using the 3rd, 4th, and 5th tools, it is advisable to use a 2.5:1 ratio, because of a slight error which is present in the linkage and changing from a radial to straight line motion. This allows

the operator to increase or decrease the throw to proportion chamfers correctly.

Use of this method for the shoulders in front of O. D. is commonly employed but not recommended due to the many complications that are developed.

One of the chief objections is that the chamfer tool must be plunged in stock, through to the diameter TO BE turned and actually forms the chamfered shoulder prior to the turning of the shoulder by the No. 1 tool. Full cutting pressure is not applied to the tool until it bites into the bar at the finish of the chamfer. The degree of inaccuracy depends upon the setting of the tool in relation to the center of the bar and is more evident on the less machinable metals. If the tool is set too far away, the result will be an incomplete chamfer.

Since the out of No. 1 tool is unproductive, there is, as a general rule, not enough time for bringing in the chamfering tool as an overlapping operation.



DEVELOPMENT OF GENERATOR OR IRREGULAR CONTOUR

When generating a curved or irregular diameter of a part, the advance of the bar is constant, or the same as for a straight turning. Therefore, the development of the section where generation occurs will be on cam relative to tool doing the turning.

This development can be constructed only after the layout has been completed, then the first step is to lay out the section of part to be generated to a scale in proportion to its size anywhere from 5:1 to 50:1.

The total scale length represents the total number of degrees for completion of generation as figured in layout.

The total scale height represents total travel through which tool must pass to develop desired curve.

To plot curve on cam, it is necessary to scale drawing to find travel of tool per degree, or every several degrees, and so on. Throughout operation this should be charted and be multiplied by ratio of rocker arm.

EXAMPLE LAYOUT

This section to be generated is drawn to scale and is represented by A-A1-AF in Illustration, page 48. The curve C2 is drawn in; the radius of this curve is the sum of radii A and B. Radius B has been determined prior to layout, but to find location of its center at start of generation, it is necessary to draw line G from the center point J so that it intersects corner of the part or very beginning of the curve to be generated. The point of intersection with curve C2 is the starting point from which all calculation or scaling is dependent.

The scale length E represents total degrees in which the generation must be completed. This length will also represent rise needed on the lead cam for this operation.

It is assumed that 36 degrees is the time in which operation must be complete and E is divided into an equal and proportionate number of spaces. 19 spaces are used in Illustration, page 49, each space representing 2 degrees.

To chart the tool travel and calculate rise on rocker cam from here, it is necessary to scale from F1 to the point of intersection of the curve C2 and the division lines.

Degrees	Scale Dim.	Tool Travel	Rise on Cam
0° — 2°	.250	.250 ÷ 40 = .0062	.0062 × 3 = .0186
2° — 4°	.430 — .250 = 180	.180 ÷ 40 = .0045	.0045 × 3 = .0135
4° — 6°	.597 — .430 = 167	.167 ÷ 40 = .004	.004 × 3 = .012
6° — 8°	.755 — .597 = 158	.158 ÷ 40 = .0039	.0039 × 3 = .0117

Etc.

NOTE: If a sharp pointed tool is used, F2 represents the scale length or total degrees and HI the scale height or tool travel.



Photo 50

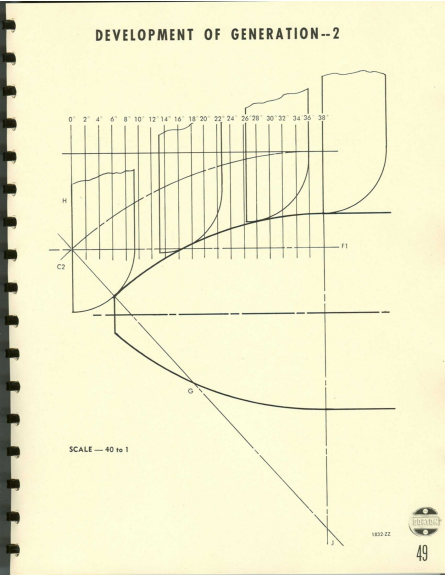


Photo 51

TYPICAL FULL AUTOMATIC WORK



Fig. 1



Fig. 2



Fig. 3

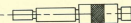


Fig. 4



Fig. 5



Fig. 6



Fig. 7



Fig. 8



Fig. 9



Fig. 10



Fig. 11

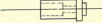


Fig. 12

1832 AAA



Photo 52

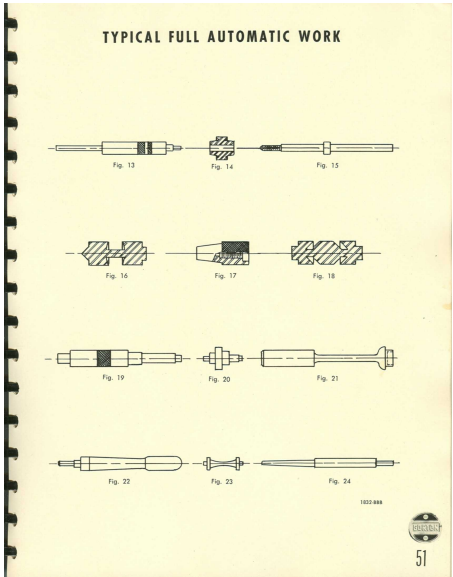


Photo 53

TYPICAL SEMI-AUTOMATIC WORK

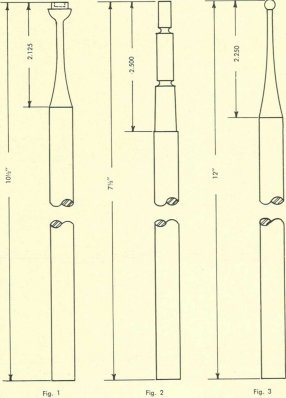


Photo 54

VARIOUS APPLICATION OF TOOLS



Fig. 1



Fig. 2



Fig. 3



Fig. 4



Fig. 5



Fig. 6



Fig. 7



Fig. 8



Fig. 9



Fig. 10



Fig. 11



Fig. 12



Fig. 13



Fig. 14

1832 DOD



CAM LAYOUT

In proceeding with the cam layout, it is well to give quite some thought to which end of the pieces to run forward to gain maximum advantage of the stop screws. Also, taking into consideration any unusual operation which might tend to eliminate use of the stop screws entirely, such as unusual depth of cutting to be made with the number 2 tool, extreme weight or generation at end of the piece, recess in shoulder, etc. When this has been determined, sketch of the part showing all dimensions to be machined in automatic operation should be drawn in the space provided on layout with front of the piece to left of sheet. This is also a common

practice, and it is readily seen that when standing in front of, or working side of the machine, sketch would be in accord with the part coming off the automatic.

All necessary tool data, such as the part number, material, bar size, surface speed, R.P.M., etc., should be entered in its proper place on layout sheet.

It is a good idea, at least for the designer who has not had too much experience, to make a work sheet layout of operations to be performed by each tool before dimensioning tools and showing their relative positions on layout sheet. (Example page 55).

LAYOUT SHEET

The layout sheet upon its completion will contain all information necessary for drawing in of the feed and rocker arm cam, also all productive or unproductive time, if not overlapping, of the third, fourth, and fifth cam, drill cam, threading time and the indexing of the 3-spindle head.

It will also contain information such as width of tools, R.P.M. of spindle, pieces per minute, etc., also sequence of operation.

The form sheets used on pages 72 and 73 are suggested forms and can be purchased from the dealer or manufacturer at a nominal cost.

Prior to the calculation on the sample layout, the end to be run forward has been determined, also the width of the cut-off tool, stop screw allowances,

mean diameter of the rocker cam, and tool clearance.

Width of cut-off tool055
Ratio of feed mechanism	2:1
Ratio of 3rd, 4th, & 5th	2:1
Stop screw allowance on feed mechanism020
Stop screw allowance on rocker005
Mean radius of rocker	2.090
Tool Clearance048

Piece is drawn in and correctly dimensioned.

Tools are drawn in showing relative position and necessary dimensions.

The last, but most important, is a complete plan of the operations in their order of sequence and is entered in the column so titled.



Photo 56

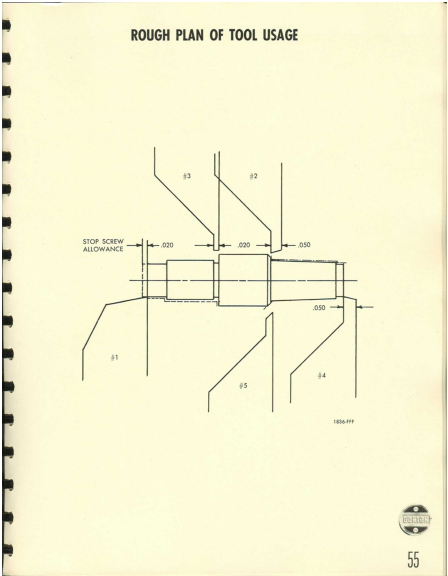
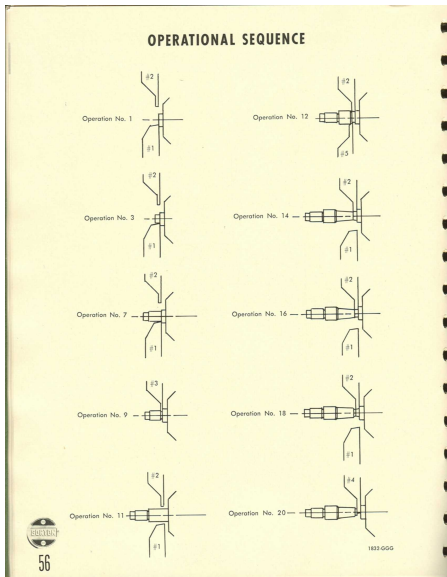


Photo 57



CALCULATION

OPERATION 1 ROCKER CAM

In sample layout on page 66, the first operation is "OUT 4 IN 1." No. 4 is the cutoff tool and its retraction is overlapping the IN operation of the number 1 which is the front turning tool. The unproductive time for this operation is based on the amount of fall necessary to bring tool from a neutral position to the first diameter of piece to be machined plus stop screw allowance.

The radial difference between the diameter of the bar and its smallest turned diameter plus the tool clearance and any stop screw allowance multiplied by ratio (which is permanently 3:1) equals total fall necessary on the rocker cam.

Example:

$$\left(\frac{.250 - .156}{2} + .005 + .048 \right) \times 3 = .300 \text{ fall on cam.}$$

Subtracting fall from the mean cam radius, the radius of cam at completion of the operation is obtained.

$$2.090 - .300 = 1.790 \text{ radius.}$$

The table on page 79 shows 9 degrees would be necessary to fall from a mean radius of 2.090 to 1.790.

The number of unproductive degrees necessary, radius at completion of fall, and diameter to be turned, are entered in their respective columns.

OPERATION 2.

This operation is a safety and intended to allow for the radius on the follower at the base of fall also easier cam alignment.

OPERATION 3 FEED CAM

In turning the first step on a piece, there must be taken into consideration any allowance that has been made for the feed mechanism stop screw. This allowance is chiefly intended to eliminate dependency upon the width of cutoff tool to hold overall length of the piece and gives the operator the privilege of using tools of optional width. In sample layout, .020" has been allowed for the stop screw dimension.

To obtain the revolutions of bar, the length to be turned plus any stop screw allowance divided by feed, equals number of revolutions for this particular operation.

$$(.093 + .020) \div .0009 = 121 \text{ revolutions.}$$

To obtain radius at completion of the rise on the feed cam, the minimum radius must be known. The sum of the length of piece, width of cutoff tool and any stop screw allowance, multiplied by the ratio of feed mechanism, equals total throw necessary on the feed cam. Total throw subtracted from the known maximum radius of the cam, equals the minimum cam radius.

$$1.940 + .055 + .020 \times 2 = 2.030 \text{ total throw.}$$

$$\text{Maximum cam dia.} = 3.156".$$

$$3.156 - 2.030 = 1.126 \text{ minimum radius.}$$

Rise on the feed cam for this operation equals the length of turning plus the stop screw allowance multiplied by the ratio being used.

$$(.093 + .020) \times 2 = .226 \text{ rise on cam.}$$



Radius at completion of rise equals minimum radius plus rise.

$1.126 + .226 = 1.352$ radius of cam at completion of rise.

Feed, revolution of bar, and radius of feed cam at completion of rise, are entered in their respective columns.

OPERATION 4.

"Safety." Here a safety is allowed and mainly intended to give the tool a chance to clean up and assure a square shoulder. Two degrees are allowed and entered under column, "Unproductive Time."

OPERATION 5 ROCKER CAM

"Out 1." This is an unproductive operation and time is based on amount of rise necessary for retraction of the tool. Rise equals the radial difference between the diameter turned and the diameter to be turned, plus stop screw allowance multiplied by ratio.

$$\left(\frac{.187 - .156}{2} + .005 \right) \times 3 = .0615 \text{ rise on cam.}$$

Radius at completion of rise equals rise plus radius on cam preceding this operation.

$.0615 + 1.790 = 1.8515$ radius at completion of rise.

(The table on page 78, shows 3 degrees are necessary.)

OPERATION 6.

"Safety." Importance is chiefly in assuring a sharp corner on piece.

OPERATION 7 FEED CAM

"Turn No. 1." As in the third operation, feed to be used is established. Then proceed to calculate number of revolutions of bar needed, rise on cam, and radius on feed cam at completion of rise.

The number of revolutions equals length to be turned divided by feed.

$$.247 \div .0009 = 271 \text{ revolutions.}$$

Rise on cam equals length to be turned multiplied by ratio used on feed mechanism.

$$.247 \times 2 = .494.$$

Radius at completion of rise equals rise on cam plus radius of cam preceding this operation.

$$.494 + 1.352 = 1.846.$$

OPERATION 8.

"Safety," purely for cam fabrication.

OPERATION 9 ROCKER CAM

"Out No. 1. In No. 3." Here No. 1 tool is withdrawn to its neutral position.

Rise on cam equals radial difference between last turned diameter and diameter of bar plus tool clearance multiplied by ratio.

$$\left(\frac{.250 - .187}{2} + .048 \right) \times 3 = .2385 \text{ rise.}$$

The radius on cam at completion of rise equals rise plus radius of cam preceding this operation.



Photo 60

and, if figuring up to this point is correct, will be mean radius of cam.

$.2385 + 1.8515 = 2.090$ radius of cam at completion of rise and mean radius of rocker arm cam.

Six degrees are necessary to rise from a radius of 1.8515 to radius of 2.090. (Table on page 78).

OPERATION 9 (Cont'd) . . . 3RD CAM

At completion of turning in Operation 7, the No. 3 tool should have been advanced to within approximately .003 of diameter to be undercut. Although the cutting of this diameter is productive, the feeds, rise on cam, and cam radii have no place on layout sheet, as the unproductive time necessary to Out No. 1 completely overlaps this operation. Actual calculation is done at "drawing in" of cams.

OPERATION 10 3RD CAM

"Out No. 3." The out of this tool not being overlapping, it is necessary to find number of degrees required to retract tool from diameter of undercut to a safety margin beyond diameter of bar.

Fall will equal radial difference between diameter of bar and diameter of undercut plus safety margin multiplied by ratio used.

$$\left(\frac{.250 - .167}{2} + .010 \right) \times 2 = .103 \text{ fall.}$$

When tool is at minimum diameter of undercut, cam will be at its known maximum radius 2.375 and table on page 85 shows two degrees necessary to fall from a radius of 2.375 to a radius of 2.272.

OPERATION 11 FEED CAM

"Advanced Head." As no metal is being removed from bar in this operation, it is unproductive, and time is based on amount of rise needed to advance stock correct distance to attain shoulder width specified on drawing.

The cutting edges of the No. 1 and No. 2 tool are directly in line with each other, which means the clearance edge of the No. 2 tool is in proportion to its width behind cutting edge of the No. 1 tool. Therefore, the stock must be advanced the length specified plus the width of No. 2 tool. This multiplied by the ratio equals rise on cam.

$$(.250 + .055) \times 2 = .610 \text{ rise on cam.}$$

Rise plus radius on cam preceding this operation equals radius at completion of rise.

$$.610 + 1.846 = 2.456 \text{ radius at completion of rise.}$$

Sixteen degrees (16°) are needed to rise from a radius of 1.846 to a radius of 2.456. (Table on page 81).

OPERATION 11 (Cont'd) . ROCKER ARM

During the advance of headstock, the No. 2 tool is brought from its neutral position to cutting position on bar, allowing small safety margin between edge of tool and outside diameter of bar.

Rise necessary equals tool clearance minus safety margin multiplied by ratio.

$$(.048 - .005) \times 3 = .129 \text{ rise on cam.}$$
$$(.129 + 2.090 = 2.219 \text{ radius at completion of rise.})$$



To rise from mean cam radius of 2.090 to 2.219 will take approximately 4 degrees, but as this operation is completely overlapped by 16 degrees necessary to advance head, it is well to take advantage of some of this time and at "drawing in" of cam 7 degrees are allowed for rise and a 4 degree dwell following and preceding rise.

OPERATION 12. ROCKER CAM

"In No. 2, Chamfer No. 5." Here the No. 2 tool is fed radially from its position just outside bar into diameter to be turned which, in this case, is a taper. Since metal will be removed from bar in so doing, it is a productive operation.

The number of revolutions of bar equals radial difference between diameter of bar and diameter to be turned plus safety margin divided by feed.

$$\left(\frac{.250 - .190}{2} + .005 \right) \div .004 = 80 \text{ revolutions.}$$

Rise on cam equals radial difference between diameter of bar and diameter to be turned plus safety margin multiplied by ratio.

$$\left(\frac{.250 - .190}{2} + .005 \right) \times 3 = .105 \text{ rise on cam.}$$

Radius of cam at completion of rise equals rise plus radius of cam preceding this operation.

$$.105 + 2.219 = 2.324 \text{ radius at completion of rise.}$$

The chamfering with No. 5 tool is overlapping and will be dealt with only at "drawing in" of cams.

OPERATION 13.

"Safety." It is most important that a safety margin, or factor allowance be made before the following operation, for "infeed" of the No. 2 tool must start exactly at the same time bar begins its forward motion or error at beginning of taper would result.

The safety allows set up man to adjust, to a limited extent, for any fractional errors of a degree that are common in cam fabrication.

OPERATION 14.

"Turn No. 2." In turning this taper, the No. 2 tool must be advanced the radial difference between largest diameter and smallest diameter of taper at which time the head advances the stock the correct length to be turned.

OPERATION 14 (Cont'd) . . . FEED CAM

After establishing feed, the number of revolutions necessary for this operation is found. Revolutions equal length to be turned divided by feed.

$$.317 \div .006 = 531 \text{ revolutions.}$$

Rise on feed cam equals length to be turned multiplied by ratio.

$$.317 \times 2 = .634 \text{ rise.}$$

Radius on feed cam at completion of rise equals rise plus radius preceding this operation.

$$.634 + 2.456 = 3.090 \text{ radius on feed cam at completion of rise.}$$



ROCKER CAM.

The rise on rocker arm cam equals radial difference between large and small diameter of taper multiplied by ratio.

$$\left(\frac{.190 - .180}{2}\right) \times 3 = .15 \text{ rise.}$$

Radius on rocker arm cam at completion of rise equals rise plus radius on cam in preceding operation.

.015 + 2.324 = 2.339 radius on rocker arm cam at completion on rise.

OPERATION 15.

"Safety."

OPERATION 16.

"In No. 2." Here tool must be advanced through stock to diameter to be turned.

Feed used is established.

Radial difference between small end of taper and diameter to be turned divided by feed equals revolutions needed.

$$\left(\frac{.180 - .156}{2}\right) \div .004 = 30 \text{ revolutions.}$$

Rise on cam equals difference between end of taper and diameter to be turned multiplied by ratio.

$$\left(\frac{.180 - .156}{2}\right) \times 3 = .036 \text{ rise.}$$

Radius on cam at completion of operation equals rise plus radius on cam preceding this operation.

.036 + 2.339 = 2.375 radius on rocker arm cam at completion of rise.

OPERATION 17.

"Safety."

OPERATION 18 FEED CAM

"Turn No. 2." As in preceding operation, feed is established and then necessary calculating is completed.

Revolutions equal length to be turned divided by feed.

$$.003 \div .0009 = 36 \text{ revolutions.}$$

Rise on cam equals length to be turned multiplied by ratio.

$$.033 \times 2 = .066 \text{ rise.}$$

Radius of cam at completion of operation equals rise plus radius of cam preceding this operation.

$$.066 + 3.090 = 3.156.$$

OPERATION 19.

"Out No. 2. In No. 4." No. 4 is cutoff tool and should be within few thousandths of diameter where parting of cutoff will be made by the time No. 2 is withdrawn to its neutral position.

OPERATION 19 (Cont'd) ROCKER ARM

Fall on rocker arm cam equals radial difference between diameter turned and stock diameter plus tool clearance multiplied by ratio.

$$\left(\frac{.250 - .156}{2} + .048\right) \times 3 = .285 \text{ fall on cam.}$$

Radius at completion of fall is mean radius of cam and must check.

$$2.375 - .285 = 2.090 \text{ mean radius.}$$

Radius of 2.375 from which fall start is known from Operation 16 and table on page 79 shows 7 degrees are necessary.

OPERATION 20 4TH CAM

"Cut off." The only figures in relation to "cutoff" that are needed at this time are feed to be used and number of revolutions of bar necessary.



To part piece from bar, it is only necessary for tool to travel one-half the diameter of last turned diameter, but to assure complete facing on the end of stock, tool must travel, in addition, one and one-half times the distance facing point is behind the cutting point of tool. The distance between these two points depends upon width of tool and clearance angle used.

Revolutions of bar necessary will equal one-half diameter of last turned diameter plus one and a half times distance facing point is behind cutting point divided by feed.

$$\frac{.156}{2} + (1.5 \times .021) \div .0052 = 190 \text{ revolutions.}$$

OPERATION 21.

"Open Collet." 10 degrees are allowed for this operation and is self explanatory.

OPERATION 22. FEED CAM

"Recoil Head". During the period of cutting off and opening collet, the headstock is at the extreme forward travel and feed cam is at the largest or maximum radius, and now must be recoiled to starting position and minimum radius on feed cam.

This fall is equal to total throw of cam and is known from Operation 3, but for checking purposes it is well to sum up all the rise on feed cam and this sum should equal figure in Operation 3.

$$.236 + .494 + .610 + .634 + .066 = 2.030 \text{ fall on cam.}$$

The number of degrees necessary to fall to minimum radius, or 1.126, from a radius of 3.156 is 31 degrees. (Table on page 81).

OPERATION 23.

"Close Collet." This is the last operation and complete cycle. 17 degrees are allowed for this operation.

PRODUCTIVE DEGREES.

The next step is proceeding with calculating number of PROductive degrees necessary for each PROductive operation. Before this can be accomplished, the sums of PROductive revolutions and UNPRO-

ductive degrees must be known. Therefore, figures in respective columns are added and total entered at proper place.

The total of UNPROductive degrees subtracted from total degree of circle, or one cycle of camshaft equals total number of PROductive degrees.

$$360^\circ - 114^\circ = 256^\circ \text{ (PROductive degrees).}$$

The total number of PROductive revolutions divided by total of PROductive degrees equals revolutions per degree.

$$1289 \div 246^\circ = 5.16 \text{ Revolutions per degree (Use 5.2).}$$

The number of PROductive degrees for each PROductive operation equals number of revolutions divided by revolutions in one degree.

$$\text{Operation 3: } 121 \div 5.2 = 23 \text{ degrees.}$$

$$\text{Operation 7: } 271 \div 5.2 = 53 \text{ degrees}$$

ETC.

PROGRESSIVE DEGREES

The progressive degrees are the sum of unproductive and productive operations in sequence and in accord with plan of operations.

These sums are derived by adding the number of degrees for an operation, productive or unproductive to the sum of the preceding operations.

TOTAL REVOLUTIONS FOR ONE PIECE

This is the total number of turns of spindle or bar to complete piece and equals one complete cycle of camshaft. Revolutions equal revolutions per degree multiplied by 360 degrees.

$$360 \times 5.2 = 1872 \text{ total number of revolutions.}$$

PIECES PER MINUTE

Self explanatory and equals spindle R.P.M. divided by revolutions for one piece.

$$1734 \div 1872 = .9 \text{ pieces per minute.}$$

GROSS HOURLY PRODUCTION

Self explanatory and equals number of pieces per minute multiplied by 60 minutes.

$$.9 \times 60 = 54 \text{ Gross hourly production.}$$



"DRAWING IN" OF THE CAMS

It is common procedure to draw each cam to scale on individual sheets which, for convenience, are inscribed with a circle graduated in degrees. These sheets can be purchased from the manufacturer at a nominal cost.

The templates necessary can be traced from the illustrations on pages 75 to 78 and will be found quite satisfactory. Templates of a plastic material are available and can be purchased in sets and are desirable where considerable layout is to be done.

The first cam "drawn in" is the "Rocker Arm" cam and the template necessary is T-5348, as illustrated on page 75.

ROCKER CAM

The template is placed over sheet, so that center of template is directly over center of inscribed circle and a thumb tack or pin is placed through small hole provided. The template can now be moved freely to any graduation of the circle and division lines are now drawn, moving in counter clockwise direction. As the faces, or layout side, of the 3rd, 4th, 5th and rocker arm cam are toward the front of machine, or the attachment end, these faces, when camshaft is in motion, would travel in a clockwise direction. Layout will be counter clockwise. The face of the feed cam, being toward the back of the machine, would therefore, be traveling counter clockwise and layout will be clockwise. Usually the first division line is the 360 or 0 graduation and the template is so positioned that curve marked "Rocker Arm Division" passes through this point of circle and a line is drawn from graduation to a point somewhat lower than minimum cam diameter. Then following in order, a division line is drawn for each operation on layout sheet in accord with column headed "progressive degrees."

Thus: 0-8-11-34-36-39, etc.

Having completed the laying out of division lines, the radii or dwells are drawn in. By referring to layout sheet, it is seen that the first dwell on the rocker arm cam will be through the second, third and fourth

operations and extends from 9 degree division line to the 36 degree division, and radius of dwell is 1.790.

The second dwell is sixth, seventh and eighth operation and from 39 degrees to 96 degrees. The radius is 1.8515 and so on.

9-36"
39-96"
See Note: 102-109"
See Note: 113-120"
137-139"
242-244"
250-259"
265-360"

NOTE: The cut of the No. 1 tool is complete at 102 degrees, and the following operations are out of No. 3 and the unproductive advance of head. The total number of degrees for these two operations is 18. During this period, the No. 2 tool is brought from its neutral position .048 from bar to .005 of the bar and is the unproductive "in". Table shows 3 degrees necessary, but since plenty of time is available, 4 degrees are used. The 14 degrees left are divided into two equal dwells, one preceding the 4 degree rise and one following.

Thus: 102 to 109 — 113 to 120.

The unproductive falls and rises are then drawn in, using corresponding curves of the template and in accord with time allotted in column headed "Unproductive degrees."

2.090 — 1.790 Fall 8"
1.790 — 1.8515 Rise 3"
1.8515 — 2.090 Fall 6"
2.090 — 2.219 Rise 4"
2.375 — 2.090 Fall 7"

To complete the rocker arm cam, the productive rises are drawn in. These rises are usually faked in since there is no point in developing the curve, as this is done at time of fabrication. Where constant rise cam miller or grinder is used, curve is taken care of automatically.



FEED CAM

The only difference in "drawing in" of feed cam is the template and the fact that division line will proceed in a clockwise order. Temp. T-5353 is used for this cam. See Page 75.

3RD CAM

Drawing in of the third cam is somewhat different, as calculating was not completed on layout and must be done at this time. The template for unproductive rise, fall and division line is No. T-5351 on page 77.

The first step is to find amount of productive time and amount of productive rise on cam.

Rise equals radial difference of diameter to be undercut and diameter of undercut plus small safety margin multiplied by ratio.

$$\left(\frac{.187 - .167}{2} + .003\right) \times 2 = .026 \text{ rise.}$$

Productive time equals radial difference between diameter to be undercut and diameter of undercut plus safety margin divided by feed.

$$\left(\frac{.187 - .167}{2} + .003\right) \div .0004 = .31 \text{ revolutions.}$$

Revolutions divided by revolutions per degree equal productive time.

$$.31 \div 5.2 = 5.9' \text{ (Use } 6' \text{) Productive time.}$$

The maximum radius of any frame tool cam being 2.375, the start of rise will be from radius of 2.349.

$$2.375 - .026 = 2.349.$$

As this rise will start at the out of the No. 1 tool, it therefore, will start at 96 degrees and finish at 102 degrees. The unproductive rise and fall are then drawn from these points to minimum radius of cam and complete the third cam.

4TH CAM

The only necessary calculating on the No. 4 cam is the amount of rise, as productive time and amount of dwell are shown on layout.

Rise equals one-half of last turned diameter plus

1.5 times distance facing point is behind cutting point of the cutoff tool, multiplied by ratio.

$$\left(\frac{.156}{2} \div .021\right) \times 2 = .198 \text{ rise.}$$

This rise is completed at maximum radius of cam which means it must start at a radius of 2.177.

$$2.375 - .198 = 2.177 \text{ radius.}$$

Calculated time for this rise is 37 degrees, starting at 265 degrees and ending at 302 degrees.

As cutoff tool acts as a stop for bar, it must remain in this position during recoil of head and closing of collet. Therefore, a dwell will follow completion of rise, or from 302 degrees to 360 degrees.

The unproductive rise is then drawn from start of productive rise to base or minimum radius of cam and the unproductive fall is drawn from end of dwell to base, or minimum radius of cam, completing drawing.

5TH CAM

The fifth cam, being completely overlapping in its operation, has not been dealt with during the layout. The chamfering operation done by this tool should be made just slightly behind the "IN" of the No. 2 tool. The first step is to find the amount of productive rise and productive degrees. Rise on cam equals depth of cut times ratio.

$$.010 \times 2 = .020 \text{ rise on cam.}$$

Productive revolutions equal depth of cut divided by feed.

$$.010 \div .0004 = 25 \text{ revolutions}$$

Revolutions divided by revolutions per degree equal productive degrees.

$$25 \text{ Revolutions} \div 5.2 = 4.8' \text{ (5' are used).}$$

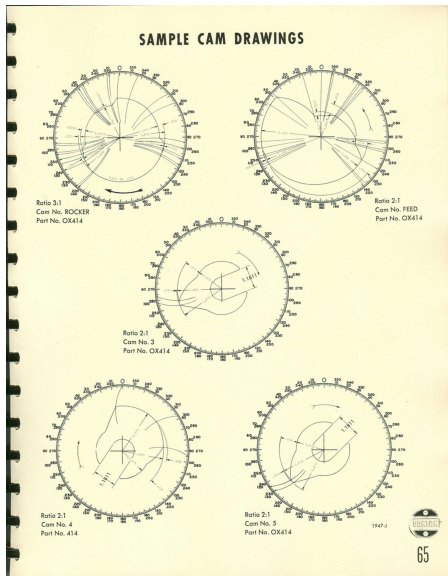
As the No. 2 tool starts infeed at 120 degrees, it is well to start No. 5 in approximately 4 degrees later, or at 124 degrees which means rise should be complete at 129 degrees. The radius of cam at start of rise is 2.355.

$$2.375 - .020 = 2.355.$$

To complete the cam, the unproductive fall and rise are drawn from the 124 degrees and 129 degrees points to base, or minimum radius of cam.



Photo 66



CAM LAYOUT WHEN DRILLING AND THREADING

Here again it is important that the designer is well acquainted with the characteristics of the attachment and primary uses of its three work spindles.

The work spindles are always referred to in their

consecutive order, starting in the back or camshaft side of machine and moving toward front or work side of machine; thus, first, second, and third position.

No. 1 SPINDLE OR FIRST POSITION

This spindle is a stationary spindle, and no provision is made for rotating it at any time. It is most commonly employed for spot centering, but where centering is accomplished from one of the machine tools proper, it may be used as a drill spindle.

Certain conditions make it impractical to drill from this spindle, mainly, the size of drill in re-

lation to R.P.M. of headstock spindle. This in turn, is governed by diameter of bar and recommended surface speed of material. It is readily seen that to drill a .032" hole in a piece being machined from a 3/8" diameter steel bar, the R.P.M. of headstock spindle would be too low for recommended surface speed of the drill.

No. 2 SPINDLE OR SECOND POSITION

This spindle, in all respects, is exactly like the No. 1 spindle with the exception that it can be revolved. As the rotation is opposite that of main spindle and

is well suited for fine drills, high drilling speed can be obtained. The drill speed equals R.P.M. of drill spindle plus R.P.M. of the main spindle.

No. 3 SPINDLE OR THIRD POSITION

The third spindle is a live spindle and its primary purpose is threading. This spindle revolves in the same direction as headstock spindle. To run on a tap or die, speed is increased over that of headstock spindle and to back off, speed is dropped to idling speed which is considerably lower than headstock speed. This method of threading is not uncommon

and similar principles are used on many conventional type automatics.

The indexing from one position to another is controlled by two flat cams. Two cams are used to add flexibility and also serves to allow for very accurate centering of each spindle independently, regardless of any inaccuracies in cam fabrication.



PROCEDURE AND CALCULATION

In this layout, the part is identical with original layout on page 66, except drilled and lepped holes have been added.

OPERATION 1.

Out 4, In 1, Advance spot center.

This operation, in all respects, is the same as operation No. 1 on original layout, with the exception that spot center is advanced unproductively to within a small margin of the face of the bar. The figures will remain the same as on original layout.

OPERATION 2.

Spot Center.

The only calculation necessary at this time is to determine number of revolutions of the bar necessary which is equal to depth of center plus safety margin divided by feed.

$$(.090 + .005) \div .0011 = 60 \text{ revolutions.}$$

OPERATION 3.

Safety.

To allow the spot center to recoil before advance of the stock in the following operation.

OPERATION 4.

Turn No. 1.

Same as Operation 3 on original layout with the exception that during this time the spindle must be recoiled to normal position and start to index head to drilling position; as these are overlapping operations and unproductive, no calculation is necessary at this time.

OPERATION 5.

Safety.

Same as No. 4 on original layout.

OPERATION 6.

Out No. 1.

Same as No. 5 on original layout.

OPERATION 7.

Safety.

Same as No. 6 on original layout.

OPERATION 8.

Turn No. 1.

Same as Operation 7 on original, except that drill spindle should start its unproductive advance and be in drilling position at completion of the "Out of No. 3."

OPERATION 9.

Safety.

Same as Operation 8 on original.

OPERATION 10.

Out 1, In No. 3.

Same as Operation 9 on original.

OPERATION 11.

Out No. 3.

Same as Operation 10 on original.

OPERATION 12.

Drill No. 2 Spindle.

Here drill is advanced productively into the piece and is figured accordingly. The only calculation necessary at this time is to find number of revolutions required. Also unproductive time necessary for recoil of drill to clear chips. This is usually necessary after drill has penetrated to a depth of three times its diameter.

Revolutions equal depth of drilled hole divided by feed.

$$.312 \div .001 = 312 \text{ revolutions.}$$

As a .070 drill is being used and depth desired is considerably over three times its diameter, it is, therefore, necessary to allow for one recoil of drill to clear the chips.

The heights on drill cams are obtained by taking known maximum height and subtracting total throw necessary to drill required depth.

$$2.250 - .317 = 1.933 \text{ minimum work height on cam.}$$

The recoil of drill will start at minimum work height of cam, plus three times the diameter of the drill.

$$1.933 + (3 \times .070) = 2.143.$$



Photo 70

From chart on page 84 it is readily seen that to fall from a height of 2.143 to 1.933 on the drill cam, it requires two (2°) degrees, and to rise unproductively the same amount, five (5°) degrees. To allow for complete withdrawal of drill, it is necessary to allow approximately three (3°) degrees at base of fall, due to radius of follower. The sum of 10° necessary for these operations should be entered in column unproductive time.

OPERATION 13.

Advance Head.

Same as Operation 11 on original.

OPERATION 14.

In 2, In 5.

Same as Operation 12 on original.

OPERATION 15.

Safety.

Same as Operation 13 on original.

OPERATION 16.

Turn No. 2.

Same as Operation 14 on original.

OPERATION 17.

Thread.

During the operations, 13, 14, 15, and 16, the recoil of drill spindle, indexing of head and unproductive advance of tap should be completed and the start of this operation is to advance tap into work; the correct lead will be figured at the "drawing in" of the cams. At this time, it is only necessary to figure number of revolutions needed for threading.

As a 4:1 ratio exists between headstock spindle and threading speed, number of revolutions equal number of threads desired multiplied by ratio.

$$16 \times 4 = 64 \text{ actual revolutions.}$$

To allow a slight safety factor it is well to add approximately 25% to this figure.

$$64 \div 16 = 80 \text{ revolutions required.}$$

OPERATION 18.

In 2.

Same as 16.

OPERATION 19.

Safety.

Same as 17.

OPERATION 20.

Turn 2.

Same as 18.

OPERATION 21.

Out 2, In 4.

Same as 19.

OPERATION 22.

Cutoff 4.

Same as 20.

OPERATION 23.

Open Collet.

Same as 21.

OPERATION 24.

Recoil Head.

Same as 22.

OPERATION 25.

Close Collet.

Same as 23.

The procedure of calculating from here is exactly as on original layout; total unproductive degrees, number of revolutions of the bar, revolutions per degree and determine number of degrees for each productive operation, etc.



"DRAWING IN" OF CAMS

Since the cams for feeding and turning are very much the same as cams in original layout, only the cams for 3 spindle attachment are illustrated.

The first cam "drawn in" is the outer index cam. Using template T6321, a division line is drawn at zero degrees which is in accord with layout and at the completion of closing of collet. As it is known that spot center is advancing into cutting position, and attachment head must remain in a fixed position, this means a dwell on index cam. The radius of this dwell is known and is 2.7/16". The head will remain in this position during the advance of spindle, actual cutting with spot center, and recoil of spindle. The 2.7/16" radius, therefore, will extend through

- 0° - 9° Advance of spindle known from layout.
- 9° - 17° Actual cutting spot center known from layout.
- 17° - 25° Degree necessary for recoil known from table, page 81, and a division line is drawn at 25° mark.

The next operation to be performed by attachment is from the drill spindle, or No. 2 position. Therefore, it is necessary to index head accordingly.

The number 2 spindle is governed by the inner index cam and necessitates a fall on outer cam from 2.7/16" in radius to 1" radius. The chart shows 14° are necessary and the correct curve is traced from the template.

As no work will be performed by the outer index cam until indexing of head back to starting position it will remain at this radius up to the 307 graduation. The 307 point or start of indexing is determined by number of degrees necessary for unproductive advance of spot center plus time required to index head from 3rd position to number 1 position.

Chart shows time required for indexing head 35°.

The unproductive advance of spot center will necessitate a rise on lobe from minimum height to start of work height of 1.685 and table shows 22 degrees required.

35° + 22° = 57°
or 360° start of operations

360° - 57° = 303° Calculated start of indexing but to insure clearance 10° are allowed as a safety factor.

Rise is then drawn in using template . . . bringing radius of cam up to 2.7/16" at the 328° point. This radius will continue to 360° or 0° mark and completes cam.

The next cam drawn in is the inner index cam. A starting point of 19° is used and cam is at its maximum radius of 2.5/8". This radius must continue through and include recoil of drill spindle. It is known from layout that drilling is complete at 144° and chart shows 6° are needed for recoil, therefore: 144° + 6° = 150° at end of recoil of spindle and end at 2.7/16" radius.

The last operation to be performed by the three-spindle attachment is from the threading spindle or third position which necessitates a fall on inner index cam from its maximum radius of 2.7/16" to a minimum radius of 1.7/16". Chart shows 11° are needed and correct fall curve is traced from template . . .

150° + 11° = 161° at completion of fall.

The radius of 1.7/16" continues through 360 or zero, to approximately 14° (time necessary for rise from minimum radius to maximum radius) or point where change over from outer cam to inner cam takes place; by looking at drawing of outer cam, it is readily seen that this takes place at 19°.

19° - 22° = 7° or 353° start of index.



Photo 72

Template is used to draw in rise bringing cam up $2\frac{1}{4}$ " radius and complete drawing.

The lobes controlling lateral motion of No. 1 and No. 2 spindles, being a bell cam segment, are laid out flat or in a straight line.

The production time for spotting is known from layout and is from 9" through to 17". Division lines are drawn at these points using template T5352. The No. 1 spindle finishes its stroke at 17" and cam is at its maximum height of $1\frac{3}{4}$ ". Depth of spot being .065" throw or rise on cam will be identical, as a 1:1 ratio exists. Therefore, height of cam at start of rise will be:

$$1.750 - .065 = 1.685 \text{ height of cam at start of rise.}$$

Unproductive rise and fall are traced from template . . . so that they intersect division line at respective heights of 1.685 and 1.750 completing spotting lobe.

Drawing in of the drill lobe is the same as spotting lobe with the exception of recoil provided for clearing chips. It is known from layout that drill spindle completes its stroke at 144, and cam is, therefore, at its maximum height of $1\frac{3}{4}$ ". The height of cam at starting point of productive rise will be maximum height of cam minus desired depth of hole.

$$1.750 - .312 = 1.438 \text{ height at start of rise.}$$

As drill must be recoiled when it has penetrated bar to a depth equal three times its diameter, it is necessary to find degree line where recoil will start.

$$.070 \times 3 = .210 \text{ depth of drill at point of recoil.}$$

$$.210 \div .001 \text{ (lead per rev.)} = 210 \text{ revolutions.}$$

$$.210 \div .7 \text{ (No. of revolutions per degree)} = 30 \text{ degrees.}$$

$$91^\circ + 30^\circ = 121^\circ \text{ point where recoil starts.}$$

A fall line is drawn here to a height of cam slightly below starting height of rise and chart shows 2" necessary. To allow for radius of follower, a 3" dwell is allowed before starting unproductive rise.

The unproductive rise, as traced from template and chart, shows 3" necessary. This unproductive rise should stop a few thousandths under point from which recoil started. This is done as a safety factor and is to make sure drill will not be advanced unproductively into base of hole.

Draw in unproductive rise, and fall, as on spotting lobe, and drawing is complete.

To calculate the lead angle of the threading cam, the number of actual degrees must be known. In layout, 11" are allotted for threading, but this includes an additional 25% of the time allowed as a safety factor.

Actual number of revolutions are known from page 69. The actual number of degrees equals revolutions divided by revolutions per degree.

$$64 \div 7.2 = 9 \text{ actual degrees necessary for 16 threads.}$$

Total number of threads divided by actual number of degrees, equals threads per degree.

$$16 \div 9 = 1.8 \text{ threads per degree.}$$

The number of lead threads (usually from 2 to 4, 3 are used here) divided by threads per degree equals total degrees for lead angle.

$$3 \div 1.8 = 1.6 \text{ total degrees (Use 2').}$$

Rise of lead angle equals pitch of thread multiplied by number of lead threads.

$$.015 \times 3 = .045 \text{ rise of lead angle.}$$



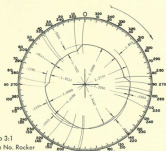
SEQUENCE OF OPERATIONS		TIME	REV. OF SPINDLE	REV. OF TABLE	FEEDS AT END OF FEED OR REV. FEEDS PER MIN.	LENGTH OF FEEDS TURNED	NUMBER OF FEEDS OR FEEDS PER MIN. PER MIN.	RECTANGULAR SEQUENCES	CIRCULAR SEQUENCES	PROG. AND DEPTH
1	Out No. 4 in No. 1				1.750		0	0	0	0
2	Start Center	0011	00							17
3	Safety									19
4	Turn No. 1 (0.025 on shop)	0009	121	134		315	1.340	17	2	26
5	Safety									28
6	Out No. 1				1.0315					31
7	Safety									33
8	Turn No. 1	0009	271	107		347	1.844	30	2	81
9	Safety									83
10	Out No. 1 in No. 3				2.095					89
11	Out No. 3									91
12	End	0013					2.450	43	15	144
13	Advance Head in No. 2				2.210					149
14	In No. 2 in No. 5	0004	90		2.324					153
15	Safety									175
16	Turn No. 2	0006	031	Super	2.039	017	0.090	25	2	248
17	Thread		80							250
18	In No. 2	0004	30		2.375					263
19	Safety									265
20	Turn No. 3	0009	54	136		003	3.150	5	2	270
21	Out No. 2 in No. 4				2.090					276
22	Cut Off No. 4	0003	190							282
23	Open Collar									312
24	Revol Head (0.025 on shop)						1.126			343
25	Close Collar									357
26										
27										
28										
29										
30										
31										
32										
33										
34										
35										
36										
TOTAL			1721		7.24 Rev. Per Minute			338	122	

SE. GORTON MACHINE CO.
MACHINE, WIS., U.S.A.

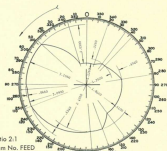


Photo 74

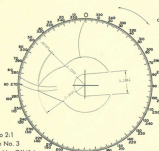
SAMPLE CAM DRAWINGS



Ratio 3:1
Cam No. Rocker
Part No. OX414

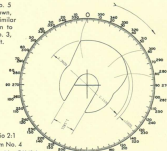


Ratio 2:1
Cam No. FEED
Part No. OX414

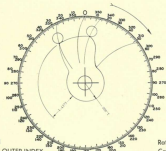


Ratio 2:1
Cam No. 3
Part No. OX414

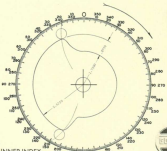
Cam No. 5
Not Shown,
as it is similar
in design to
Cam No. 3,
or left.



Ratio 2:1
Cam No. 4
Part No. OX414



Ratio 1:1
Cam No. OUTER INDEX
Part No. OX414



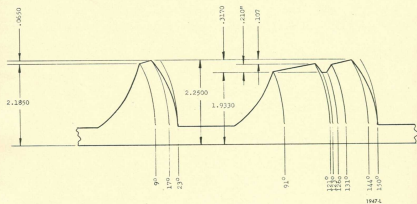
Ratio 1:1
Cam No. INNER INDEX
Part No. OX414



1947 E

Photo 75

CENTERING & DRILLING CAM



THREADING CAM

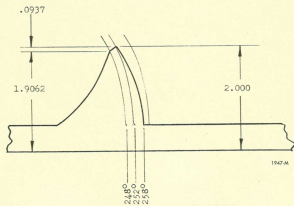


Photo 76

TEMPLATES

The templates on the following pages are full scale drawings and may be traced for producing satisfactory templates for use in cam designing or cam fabrication.

The curves rise and fall are developed curves and produce a desirable contact angle for the toe or riding piece to give working member a smooth motion and eliminate abnormal wear on profile of cam. Corresponding table on following pages.

The division lines correspond to the motion of the riding piece on the various arms or slides and it is essential that correct division lines are used in laying out the cams to obtain accurate coordination of the operations.

Templates of either transparent material for use in designing or steel for use in cam fabrication can be purchased from the maker at a nominal cost.

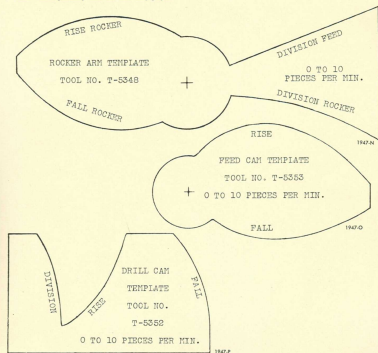


Photo 77

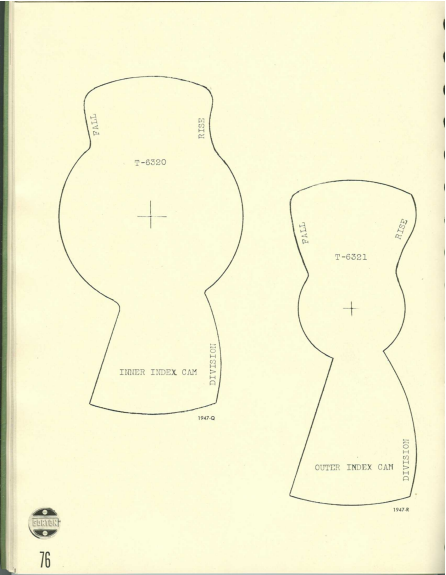
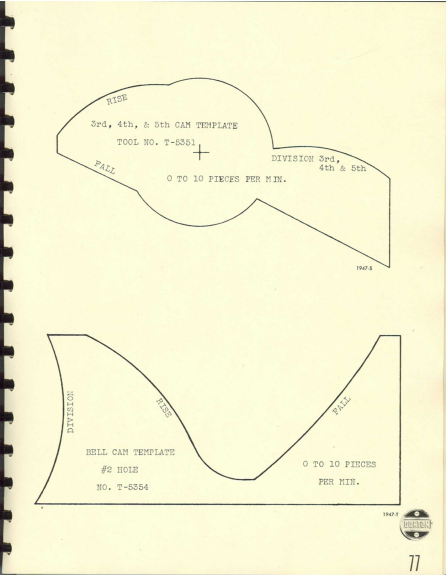


Photo 78



DEGREES NECESSARY FOR RISE ON ROCKING ARM CAM FOR A PRODUCTION OF 0 TO 10 PIECES PER MINUTE

		RADIUS AT COMPLETION OF RISE											
		1.125	1.25	1.375	1.50	1.625	1.75	1.875	2.00	2.125	2.250	2.375	2.50
RADIUS AT START OF RISE	1.00	6	12	17	22	26	30	33	36	39	42	45	48
	1.125	6	11	16	20	24	27	30	33	36	39	42	
	1.25			5	10	14	18	21	24	27	30	33	36
	1.375				5	9	13	16	19	22	25	28	31
	1.50					4	8	11	14	17	20	23	26
	1.625						4	7	10	13	16	19	22
	1.75							3	6	9	12	15	18
	1.875								3	6	9	12	15
	2.00									3	6	9	12
	2.125										3	6	9
2.250											3	6	
2.375												3	



Photo 80

DEGREES NECESSARY FOR FALL ON ROCKING ARM CAM FOR A PRODUCTION OF 0 TO 10 PIECES PER MINUTE

		RADIUS AT START OF FALL											
		1.125	1.250	1.375	1.500	1.625	1.750	1.875	2.000	2.125	2.250	2.375	2.500
RADIUS AT COMPLETION OF FALL	1.000	6	11	16	20	24	27	30	33	36	39	42	45
	1.125		5	10	14	18	21	24	27	30	33	36	39
	1.250			5	9	13	16	19	22	25	28	31	34
	1.375				4	8	11	14	17	20	23	26	29
	1.500					4	7	10	13	16	19	22	25
	1.625						3	6	9	12	15	18	21
	1.750							3	6	9	12	15	18
	1.875								3	6	9	12	15
	2.000									3	6	9	12
	2.125										3	6	9
	2.250											3	6
	2.375												3



Photo 81

DEGREES NECESSARY FOR FALL ON THE FEED CAM FOR A PRODUCTION OF 0 TO 10 PIECES PER MINUTE

		RADIUS AT START OF FALL																	
		1.125	1.250	1.375	1.500	1.625	1.750	1.875	2.000	2.125	2.250	2.375	2.500	2.625	2.750	3.000	3.125	3.250	
RADIUS AT COMPLETION OF FALL	1.000	9	5	7	9	11	13	15	17	19	21	23	25	27	29	31	33	35	37
	1.125		2	4	6	8	10	12	14	16	18	20	22	24	26	28	30	32	34
	1.250			2	4	6	8	10	12	14	16	18	20	22	24	26	28	30	32
	1.375				2	4	6	8	10	12	14	16	18	20	22	24	26	28	30
	1.500					2	4	6	8	10	12	14	16	18	20	22	24	26	28
	1.625						2	4	6	8	10	12	14	16	18	20	22	24	26
	1.750							2	4	6	8	10	12	14	16	18	20	22	24
	1.875								2	4	6	8	10	12	14	16	18	20	22
	2.000									2	4	6	8	10	12	14	16	18	20
	2.125										2	4	6	8	10	12	14	16	18
	2.250											2	4	6	8	10	12	14	16
	2.375												2	4	6	8	10	12	14
	2.500													2	4	6	8	10	12
	2.625														2	4	6	8	10
	2.750															2	4	6	8
	2.875																2	4	6
3.000																	2	4	
3.125																			2



Photo 82

DEGREES NECESSARY FOR RISE ON THE FEED CAM FOR A PRODUCTION OF 0 TO 10 PIECES PER MINUTE

		RADIUS AT COMPLETION OF RISE																	
		1.125	1.250	1.375	1.500	1.625	1.750	1.875	2.000	2.125	2.250	2.375	2.500	2.625	2.750	2.875	3.000	3.125	3.250
RADIUS AT START OF RISE	1.000	4	8	12	15	18	21	24	27	30	34	38	41	44	47	50	53	56	59
	1.125		4	8	11	14	17	20	23	26	30	34	37	40	43	46	49	52	55
	1.250			4	7	10	13	16	19	22	26	30	33	36	39	42	45	48	51
	1.375				3	6	9	12	15	18	22	26	29	32	35	38	41	44	47
	1.500					3	6	9	12	15	19	23	26	29	32	35	38	41	44
	1.625						3	6	9	12	16	20	23	26	29	32	35	38	41
	1.750							3	6	9	13	17	20	23	26	29	32	35	38
	1.875								3	6	10	14	17	20	23	26	29	32	35
	2.000									3	7	11	14	17	20	23	26	29	32
	2.125										4	8	11	14	17	20	23	26	29
	2.250											4	7	10	13	16	19	22	25
	2.375												3	6	9	12	15	18	21
	2.500													3	6	9	12	15	18
	2.625														3	6	9	12	15
	2.750															3	6	9	12
	2.875																3	6	9
3.000																	3	6	
3.125																		3	

RISE AND FALLS ON INDEX CAMS

RISE:			FALL:		
From Position	To Position	Degrees Necessary	From Position	To Position	Degrees Necessary
3	2	21°	1	3	21°
2	1	23°	1	2	16°
2	1	20°	2	3	15°



DEGREES NECESSARY FOR FALL ON THE BELL CAM (No. 2 HOLE) FOR A PRODUCTION OF 0 TO 10 PIECES PER MINUTE

HEIGHT AT START OF FALL											
		1.750	1.875	2.000	2.125	2.250	2.375	2.500	2.625	2.750	3.000
500	3										
550	6										
600	9										
650	3										
700	6										
750	9										
800	3										
850	6										
900	9										
950	3										
1000	6										
1050	9										
1100	3										
1150	6										
1200	9										
1250	3										
1300	6										
1350	9										
1400	3										
1450	6										
1500	9										
1550	3										
1600	6										
1650	9										
1700	3										
1750	6										
1800	9										
1850	3										
1900	6										
1950	9										
2000	3										
2050	6										
2100	9										
2150	3										
2200	6										
2250	9										
2300	3										
2350	6										
2400	9										
2450	3										
2500	6										
2550	9										
2600	3										
2650	6										
2700	9										
2750	3										
2800	6										
2850	9										
2900	3										
2950	6										
3000	9										

HEIGHT AT COMPLETION OF FALL



DEGREES NECESSARY FOR RISE ON THE DRILL CAM FOR A PRODUCTION OF 0 TO 10 PIECES PER MINUTE

		HEIGHT AT COMPLETION OF RISE															
		.625	.750	.875	1.000	1.125	1.250	1.375	1.500	1.625	1.750	1.875	2.000	2.125	2.250		
HEIGHT AT START OF RISE	.500	5	9	13	16	18	21	23	26	28	30	32	35	38	40		
	.625		4	8	11	13	16	18	21	23	25	27	30	35	37		
	.750			4	7	9	12	14	17	19	21	23	26	29	31		
	.875				3	5	8	10	13	15	17	19	22	25	27		
	1.000					2	5	7	10	12	14	16	19	22	24		
	1.125						3	5	8	10	12	14	17	20	22		
	1.250							2	5	7	9	11	14	17	19		
	1.375								3	5	7	9	12	15	17		
	1.500									2	4	6	9	12	14		
	1.625										2	4	7	10	12		
	1.750											2	5	8	10		
	1.875												3	6	8		
	2.000													3	5		
	2.125														2		

DEGREES NECESSARY FOR FALL ON DRILL CAM FOR A PRODUCTION OF 0 TO 10 PIECES PER MINUTE

2.250	to	1.877	2 Degrees
2.250	to	1.500	4 Degrees
2.250	to	.500	6 Degrees



DEGREES NECESSARY FOR FALL ON 3rd, 4th, AND 5th CAM FOR A PRODUCTION OF 0 TO 10 PIECES PER MINUTE

		RADIUS AT START OF FALL							
		1.500	1.625	1.750	1.875	2.000	2.125	2.250	2.375
RADIUS AT COMPLETION OF FALL	1.375	7	13	17	21	24	26	27	28
	1.500		6	10	14	17	19	20	21
	1.625			4	8	11	13	14	15
	1.750				4	7	9	10	11
	1.875					3	5	6	7
	2.000						2	3	4
	2.125							1	2
	2.250								1
	2.375								

DEGREES NECESSARY FOR RISE ON 3rd, 4th, AND 5th CAM FOR A PRODUCTION OF 0 TO 10 PIECES PER MINUTE

		RADIUS AT COMPLETION OF RISE							
		1.500	1.625	1.750	1.875	2.000	2.125	2.250	2.375
RADIUS AT START OF RISE	1.375	6	13	20	26	31	36	41	47
	1.500		7	14	20	25	30	35	41
	1.625			7	13	18	23	28	34
	1.750				6	11	16	21	27
	1.875					5	10	15	21
	2.000						5	10	16
	2.125							5	11
	2.250								6
	2.375								



Photo 87

CLEARANCE ALLOWANCE BETWEEN ROCKER ARM TOOLS AND STOCK

Stock Dia.	Clearance Per Side	Stock Dia.	Clearance Per Side
0 to 1/32	.155	3/16 to 7/32	.060
1/32 to 1/16	.142	7/32 to 1/4	.048
1/16 to 3/32	.125	1/4 to 9/32	.030
3/32 to 1/8	.110	9/32 to 5/16	.025
1/8 to 5/32	.095	5/16 to 3/8	.022
5/32 to 3/16	.075	3/8 to 7/16	.020

WIDTH OF CUT OFF BITS FOR VARIOUS STOCK DIAMETERS AND MATERIALS

Stock Dia.	MATERIAL	MATERIAL
	Brass Aluminum Mild Steel	Drill Rod Stainless Steel Hard Bronze Monel Metal
0 to 1/16	.030	.040
1/16 to 3/32	.035	.045
3/32 to 5/32	.040	.050
5/32 to 3/16	.045	.050
3/16 to 1/4	.050	.055
1/4 to 9/32	.055	.060
9/32 to 5/16	.060	.060
5/16 to 3/8	.070	.070
3/8 to 7/16	.080	.080



CAM DIMENSIONS AND LEVER RATIOS

Name of Cam	Type of Cam	FLAT CAM			BELL CAM			E Bore	D Width	C Max. Throw	B Min. Dia.	A Max. Dia.	F Max. Ht.	H Min. Ht.	Ratio of Lever
		E	A	B	C	D	F								
Feed	Flat	6.312	2.125	2.125	2.125	.312	.7876							1:1, 2:1 3:1	
Feed	Bell	4.750	4.750	2.750	2.750	.312	.7876	3.250	.500					1:1	
Rocker Arm	Flat	4.750	2.125	1.312	1.312	.312	.7876							3:1	
3rd. 4th. & 5th	Flat	4.750	2.750	1.000	1.000	.312	1.1811							2:1, 3:1	
Drill	Bell	4.750	4.750	1.750	1.750	.312	.7876	2.250	.500					1:1	
Inner Index	Flat	5.250	3.500	1.687	1.687	.312	.7876							1:1	
Outer Index	Flat	4.675	2.000	1.000	1.000	.312	.7876							1:1	

FOR MEAN DIAMETER OF ROCKER CAM SEE PAGE 38

1002-008



SETUP OF MACHINE AND ATTACHMENTS

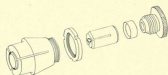
The setting up of the Gorton Automatic is not complicated if a few simple rules are observed and a common procedure followed. This step by step setup is to acquaint foremen and setup men with general procedure, adjustments, placing of cams, tools, etc.

Before setting up a new job, the machine should be thoroughly cleaned and lubricated; in the case of used machines check all moving parts for any wear; mainly gibs of the slide tools, and headstock. The chucking sleeve, chucking sleeve tube and equalizing spacer should be removed and cleaned. To remove dirt and sludge from inside of spindles, a bottle brush will answer the purpose quite satisfactorily. Replace equalizer, tube and correct chucking sleeve, then proceed with setup of job.

The part to be set up is the same as used in the sample layout on page 72, and the first step is to determine correct chucking sleeve to be used. As stock size is $\frac{1}{4}$ ", the collet will be in the category of large collets, or No. 27J. Therefore, large chucking sleeve will be used. After positioning the chucking sleeve, insert collet, making sure retaining spring is behind it. Then screw spindle nose cap into place and tighten with the spanner wrench provided.

NOTE: LEFT HAND THREAD.

A stationary bushing is used for this particular job. The stock diameter being $\frac{1}{4}$ ", bushing type GT is

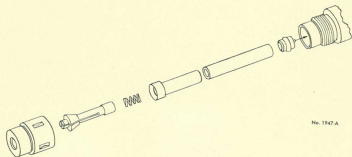


No. 1947 B

used. The bushing holder is held securely in tool frame by large nut provided. Place $\frac{1}{4}$ " bushing in bore of holder, followed by spacer and adjusting nut.

Before proceeding further, a bar of stock is placed in machine, so that adjustment of the collet and guide bushing may be completed.

To place bar in machine, loosen knurled nut and the feed tube is pulled toward the operator. Place the pointed end of bar into countersunk end of tip of push rod. Shove bar into feed tube until its end clears headstock spindle. Push feed tube back into its normal position and lock. The bar can now be brought forward through the spindle into the collet and guide bushing.



No. 1947 A



Tension is applied to collet by the large nut. To adjust tension, release the lock screw and nut is then turned right hand to increase tension and left hand to decrease tension on the bar. When properly adjusted, there should be enough tension to necessitate a good firm, but not forcible, pull on lever used to open and close collet manually. Of course, this depends a good deal on size of bar and type of material.

The adjusting of the guide bushing is very simple. To bring bushing to correct fit on bar, draw up on the nut in rear of holder. When guide bushing is properly adjusted, stock should rotate freely, without any noticeable amount of drag. The type of material regulates to a large extent the fit of bushing and a little experimenting is sometimes necessary for the inexperienced operator.

The next step is the positioning of the cams, and the first cam to be placed on the machine is the feed cam. For a starting point, it is common practice to use the opening of the collet. Close the collet manually and rotate the camshaft, using hand wheel until opening dog rests against pin. This is the beginning of the opening of collet and from lay-out sheet, it is known that opening starts at completion of cutoff, or the 302 degree graduation. Cam is



No. 1947 C

placed on camshaft and positioned so that follower of feed mechanism will contact cam at the 302 degree graduation. Replace the correct spacer and nut and lock cam in place.

Since this is maximum radius of cam, and headstock has been advanced to its farthest position, it is well to check amount of clearance existing between the spindle nose and the adjusting nut of the bushing support. The minimum clearance should not be less than $\frac{1}{4}$ ". To set head to desired clearance, loosen nuts holding Tee Block, head may now be advanced or retarded, leaving correct clearance. Slide Tee Block back against shoe of feed mechanism and lock in place (shoe should be at approximate ratio to be used, as any great change will definitely affect stroke of spindle, and therefore, clearance.)

Turn the camshaft by means of handwheel until head is completely retracted. Check if weights are exerting enough tension on headstock to assure total retraction of head. To decrease or increase tension, weights may be removed or added accordingly.

Close the collect manually, then turn camshaft until follower of feed mechanism is resting directly opposite 0 graduation. From the layout, it is known that this should complete the closing of the collet. To set the closing dog, loosen the lock screw and retard or advance dog until the roller is directly opposite pin of the closing mechanism and lock in this position.

The next cam to be placed on the machine is the rocker arm cam. Since marking on rocker arm cam and feed cam are always relative to each other, their respective followers must be at a common graduation. Since feed cam is at 0 degree, the rocker arm cam is placed likewise, and is locked in this position.



Photo 91



In placing of the third cam, it is seen that the only markings on this cam are at 94° and 99°. Therefore, the camshaft is rotated until followers of feed and rocker cam are at either of these graduations on their respective cams and the third cam is positioned accordingly and locked in place.

The 4th and 5th cams are positioned in the same manner as No. 3 cam, and completes placing of cams necessary for turning, chamfering, cutting off, etc.

The cams for operating the three spindle head will be dealt with separately, or after completing setup of machine proper. This is done mainly to simplify explanation of setup.



The next step is placing of tools and turning blank. Rotate the camshaft manually until followers of feed mechanism and rocker arm are at 343 degree graduation or where headstock is completely retracted. Position stock so that approximately $.020''$ to $.030''$ of bar protrudes through guide bushing and close the collet manually. Turn camshaft to 9 degree graduation which completes "in" of No. 1 tool. The rocker arm follower is now resting on the minimum diameter of its respective cam. Make certain stop screw is backed off, causing no interference. The tool is then placed in No. 1 holder, so that cutting point is as close as can be visually approximated to first diameter to be turned. Hold the tool firmly against the back of the tool block and lock in position.

Start the machine, using a low spindle speed. Turn on coolant and, with hand wheel make lateral cut of approximately $1/16''$. Turn off coolant. Loosen gib lock nut. Using micrometer nut the No. 1 tool is run in past center of bar. Center tool in its relation to center of the bar. Then back out to diameter to be turned and again lock in position. Turn on coolant, and still operating manually, continue to turn first two steps on piece through to "in" of No. 2, or 120 degrees. Place No. 2 tool in tool block, so that its cutting point is within a few thousandths of bar. Hold tool firmly against back of block and lock in place. Place pin provided into rocker arm and slowly bear down on it, bringing tool into the bar and cutting off portion turned. If tool is correctly centered, end of bar will be smooth and free of any burr. If a tear is left, the tool must be correctly adjusted.

Operating manually, turn the taper and the $.156''$ diameter, stopping at the completion of "Out 2" or the 265 degree graduation. The next operation is "cut off No. 4". Tool is placed in 4th position with cutting point a few thousandths from $.156$ diameter and is locked in place. Operating with hand feed, make cutoff. If any tear is left on end of bar, center the tool. Continue to 0 graduation, completing cycle.



Set camshaft and headstock spindle at proportionate low speeds, turn on coolant, and operating automatically, run off trial piece.

The first and only dimension to be checked on first, or trial piece, will be the .247" length. This is to determine if ratio of feed mechanism is correctly set, as this dimension is determined solely by throw of cam. To make adjustment, shoe is moved away from axis of lever to lengthen dimension and toward axis to shorten dimension. Another trial piece is then run off, assuming .247" dimension is correct, the .250" length, .940" or overall length and .033" length are checked.

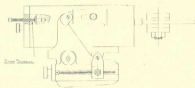
The .250" dimension is governed by the No. 1 and No. 2 tool and correction may be made by moving either tool. To adjust, loosen centering screw and tool block is then moved accordingly using micrometer nut.

The .033" dimension is governed by the cutoff tool or its setting in relation to the No. 1 and No. 2 tools. To lengthen dimension, tool must be moved toward tool frame and away from tool frame to shorten.

The overall length, or .940" dimension, is controlled with stop screw of feed mechanism. If specified width tool has been used, piece should check .020 long which is taken up with stop screw.

When all lateral dimensions check with drawing, proceed with turned diameters.

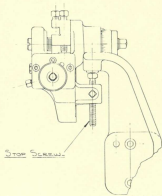
The first diameters to be checked are the .187" and the back .156" dimensions. Tools are adjusted accordingly and another trial piece is run off. When the .187" dimension is correct the .156" front dimension should be considerably undersize which is corrected with the rocker arm stop screw. If .156" back diameter is correct, the taper should check also, as this dimension is controlled by throw of cam.



No. 1947-F

To set third or undercutting tool, stop machine at completion of turning the .187" diameter. Tool is then set in holder with its cutting edge a few thousandths away from .187" diameter and is locked in place. The No. 5 or chamfer tool is set in a similar manner just after the "in" of the No. 2 tool. With the setting of these two tools, blank is complete. The next step will be setting up of three-spindle head for spot centering, drilling, and threading of the piece.

The attachment is mounted against scraped surface of machine proper and held securely by the four screws provided. See Fig. 1 on page 84. A simple method of centering the attachment is to place a dowel pin in guide bushing. A small indicator is placed in middle spindle in such a manner



No. 1947-G



ner that it can be rotated around dowel pin. In this manner, positive alignment is assured. Shoulder studs, with jack screws, should be in position to accomplish the alignment. After mounting the attachment, the necessary cams for its operation are placed on camshaft.

Prior to placing of the cams, remove stock from guide bushing and collet. Then turn camshaft to 9 degree graduation. Remove nut and all spacers up to the rocker arm cam. Mount cams necessary for operating attachment with correct spacing, as shown on page 14. The spotting lobe shows a 9 degree graduation. The cam is so positioned that the follower of work lever rests at this point. The outer index cam also has 9 degree graduation and is correctly positioned. These are the only cams positioned at this point. Lock cams firmly in place and turn to the 81 degree graduation on rocker arm cam which is start of drilling operation.

Corresponding markings are found on inner index cam which is positioned accordingly. Lock cams in position and turn to the 243 degree graduation. From layout, it is known that this is start of threading operation. There are no markings on the thread cam, but since, at this point, die should start on work, follower should rest at the very start of lead angle of thread cam and is so positioned. It is necessary that clutch has been engaged and thread spindle should be running at threading speed. This should take place at least 10 degrees prior to threading and starting cam is so positioned. This completes placing of the cams and now they are locked firmly in position.

Turn camshaft to the opening of collet. Stock is now brought back into bushing. Operate machine manually and turn to 9 degrees start of spotting operation. Place proper collet and spot center in No. 1 spindle. Spot center is brought forward to within a few thousandths of end of bar and is tightened firmly in collet. Spotting tool is now centered in its relation to center of bar. Operating manually, make spotting operation and continue through to start of drilling operation.

Place correct collet and drill in middle spindle. Position and center in similar manner as spot centering tool. Operate manually through drilling operation. Then run off balance of piece, using power feed. Run another trial piece and check for concentricity and depth of hole. To regulate depth, unlock set screw and adjust.

To set tap, operating manually, turn camshaft until follower is resting on very beginning of lead angle of thread cam. Tap is placed in third position and correctly centered in relation to hole to be tapped. Tap is then brought toward work, using adjustment screw pounded until it is within approximately .010" of bar. Unlock set screw and set collar approximately $\frac{1}{8}$ " away from ball contacting point of lever. This governs depth of tap. $\frac{1}{8}$ " setting is used to insure tap will not bottom on first piece and break. Back camshaft up about 10 degrees, then using power feed, run trial piece. If tap did not have enough lead to start, adjustment is made and another trial piece is run. Check depth of thread which should be considerably short. Adjustment is made and completes piece and machine is ready for operation or first piece inspection.



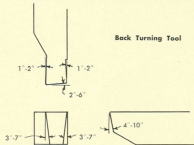
TOOL GRINDING CLEARANCES

Grinding of tools for the automatic is little different than tools for common bench or engine lathes. The clearance grinds shown in illustrations at right are merely suggestions to give the new operator a general idea as to clearance angles commonly used. Since the nature or machinability of the material to be turned is the chief factor in determining tool clearance, a little experimenting is sometimes necessary to arrive at the best grind for a specific material.

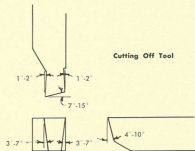
Top rake is very seldom found beneficial nor is its use recommended. The purpose of top rake on a tool is to minimize chip friction where cuts are fairly heavy. Since on the Gorton Automatic very light feed with high spindle speeds are used, little benefit would be afforded.

Clearances shown are for high speed steel tool bits only. When employing carbide tool bits side clearances should be reduced approximately 50%.

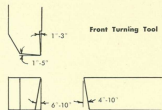
Back Turning Tool



Cutting Off Tool



Front Turning Tool



1832-BRR



Photo 95

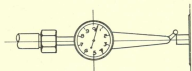


FIG. 1
Showing dial indicator mounted in middle spindle of the 3-
spindle attachment for centering of attachment.

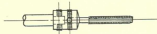


FIG. 2
Adaptation of tap to threading spindle.

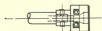


FIG. 3
Adaptation of Geometric Die head to threading spindle.

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COMMON TROUBLES

Error in the form of a front or back taper on what should be a cylindrical turning will sometimes be encountered. Fundamentally there are only three conditions responsible for an error of this kind.

1. Tool Setting
2. Poor Tooling
3. Machine Error

TOOL SETTING

Back taper may be caused by tool being set slightly below center of bar. A purely theoretical explanation why this condition results is:

When the tool makes its initial plunge, its cutting edges are at a disadvantage and, therefore, the tool point springs below center of the bar. As it continues cutting, it will, over a period of revolutions of bar, right itself to a limited extent causing turning to become smaller proportionately. Since feed is constant, result is usually a tapered surface and not irregular or wavy.

By the same token it would seem that front taper would be caused by tool being above center of bar,

but this is not true. Raising the tool point slightly above center of bar reduces tool spring, the cutting load remains uniform and no taper results. On completion of cut the tool springs up and away from center of bar, eliminating the possibility of taper from tool spring.

Tools set too far away from bushing also result in a taper. This can usually be distinguished by the fact that finish will be poor and surface may be wavy. Theory of this is somewhat on the same line as tools being below center. Only here the bar springs away from tool when initial cut is made and rights itself irregularly, resulting in a poor finish.

POOR TOOLING OR CAMMING

Front or back taper may be caused by dwell or radius of rocker cam, not being concentric with bore. Reworking of cam is necessary and the only means of correcting error.

A large bore in rocker cam resulting in a loose fit on the camshaft will naturally throw dwell or radii of cam eccentric to center of camshaft and the most practical correction is to have a new cam made.

Accelerated rise on feed cam may cause taper on the less machinable metals. An accelerated rise on feed cam would increase feed or width of cut per revolutions, causing greater cutting pressure

which in turn creates more tool spring.

Guide bushing bell mouthed in front will have same effect as tools set too far away from bushing.

Guide bushing bell mouthed in back may cause tapered condition. This theory is somewhat different than any previous explanation. Here when tool makes its initial bite into work, bar is bowed and straightens up during turning which usually results in a front taper.

Taper may also be caused by incorrect chamfering, see page 46.



MACHINE ERROR

Machine error is not very likely as machines are precision built and thoroughly and competently checked before leaving factory. But if tapered condition persists after all other possibilities of taper

have been eliminated, it is well to check the headstock, fit of rocker, concentricity of camshaft, and alignment of spindle with bore of tool frame.

VARIATION OF SHOULDER OR LATERAL DIMENSION AND OBLIQUE SHOULDERS

A loose fit of the rocker arm in its relation to its bearing against the tool frame is one of the chief causes of inability to hold or obtain correct shoulder lengths and often results in oblique shoulders. Provision is made for adjustment of the rocker arm through means of nut, (see page 131). The rocker arm when correctly fitted should, if operated by hand with tension spring removed, have a slight tendency to drag, never a binding fit.

Incorrect preload on the radial thrust bearing of the main spindle, or a worn spindle bearing may cause a similar condition. This is readily under-

stood when consideration is given to the fact that during the automatic cycle extremely light cut or maximum heavy cuts may be encountered, thus setting up proportionate pressures against the thrust bearing. Therefore, the preload must be sufficient to withstand the maximum pressures encountered. Otherwise spindle is apt to creep on the heavier cuts returning to a normal position at the completion of cut or relief of extreme pressure which is usually during the out of the tool, thus causing the shoulder to be out of square and length of turning incorrect.

DIAMETER OUT OF ROUND AND INCONSISTENCY OF SIZES

A determining factor in producing perfectly round accurate turnings is the condition of the stock. The stock must be round, and of uniform size throughout its entire length, also uniformity in size of the bars.

For very accurate work, centerless ground stock is recommended, maximum out of roundness .0003", uniformity throughout the length $\pm .0003$ ", uniformity of the bars $\pm .0005$ ".



COOLANTS AND CUTTING OILS

The various oil companies have over a period of years spent vast sums of time and money in their research laboratories to produce the various cutting oils on the market today. These companies have men who specialize in this field. They also have special equipment for testing the cooling qualities, the anti-weld properties, the pressure resistance, the lubricity, etc., of the cutting fluid and its components. Therefore, the recommendation of the supplier should be efficient and superior to any recommendation by the machine builder. Thus, we would rather not recommend any brand or grade of oil, but rather leave the proper selection to the customer's preference.

Many good articles have been published by the various oil suppliers. One of these is the following article on "Cutting Oils," an excerpt from the booklet, "Fluids for Metal Cutting" published by the Socomey-Vacuum Oil Company, Inc. and should be beneficial to the foreman or machine operator in understanding effects of the different components in the cutting fluids and aid them in selecting the correct coolant for specific type materials.

The cutting oils employed today may be classified as corrosive or non-corrosive, depending upon whether they will or will not stain, discolor, or corrode non-ferrous metals, such as copper, brass, bronze, etc. In general, they are made by adding suitable pressure-resisting, anti-welding and lubricity ingredients to straight-mineral oils. The pressure-resisting and anti-welding ingredients may consist of various forms of sulphur, chlorine, etc. The lubricity ingredients comprise one or more of the various animal or vegetable fats. The amount and nature of these ingredients depend upon the magnitude of the cutting pressures and upon cutting qualities of the metal. Cooling properties are obtained by adjusting the viscosity of the oil to the requirements of the job or range of jobs.

Exhaustive laboratory tests, supported by field experience, show the relative values of various forms of sulphur, the usefulness of fatty ingredients, the advantages and disadvantages of chlorine and the

influence of oil viscosity in the machining of various types of metals. Three widely different metals are involved —

- Hot-rolled S.A.E. 1020 bar stock (120 Brinell), which is quite draggy, and is machined with moderate cutting pressures;
- Hardened S.A.E. 3140 bar stock (280 Brinell), which is relatively clean-cutting, and is machined with high cutting pressures;
- Two-inch seamless tubing, of approximately the same composition and hardness as the S.A.E. 1020 bar stock, but having only a thin wall to conduct away generated heat.

It was not the purpose of these tests to select the most suitable cutting fluid for the machining of these materials. On the contrary, the aim was merely to determine the influence of the ingredients used, and the functions which each can be expected to perform.

EFFECT OF SULPHUR

The presence of sulphur in cutting oils provides considerable pressure-resisting characteristics that minimize friction on the lip of the tool, preventing chip seizure and reducing tool vibration. Sulphur also imparts anti-welding characteristics which keep the built-up edge mobile and sloughing the chip.

Cutting oils may contain one or more of four different kinds of sulphur —

- The natural sulphur in the mineral oil;
- The free sulphur that has been added to the mineral oil;
- The sulphur that has been previously combined with fat to form a sulphurized fat base;
- Sulphur compounds (including sulphurized fat bases).

Although the natural sulphur in a mineral oil is more or less inactive, it does have some pressure-resisting and anti-welding characteristics, which are



evidenced by its effectiveness in reducing friction, seizure and welding. The free sulphur that is added to a mineral oil, however, is very active and, therefore, much more effective. The relative effectiveness of natural and free sulphur is shown by the test record in Table III, when the hard high-Speed Steel (S.A.E. 3140) was threaded with a mineral oil low in natural sulphur, a mineral oil high in natural sulphur, and a mineral oil containing 2 1/4% of free added sulphur. The oil high in natural sulphur out-

performed the low sulphur oil, but the mineral oil containing free added sulphur had still greater pressure-resisting characteristics, and outperformed the other two in every respect, although its total sulphur content was very little greater than the mineral oil with high natural sulphur. The quantity of sulphur in a cutting oil, therefore, is not always a criterion of its effectiveness, since the kind of sulphur also influences materially its pressure-resisting and anti-welding characteristics for the work.

TABLE III
EFFECTIVENESS OF NATURAL SULPHUR AND FREE SULPHUR
IN VARIOUS MINERAL OILS OF SAME VISCOSITY

THREADING HARDENED S.A.E. 3140 BAR STOCK (280 B.H.N.)

Type of Oil	%	Sulphur Kind	Tool Life	Finish*	Chaser Wear Per Min.
Mineral Oil Low in Natural Sulphur	0.32	All Natural	2 Min.	0.5	.0510"
Mineral Oil High in Natural Sulphur	2.43	All Natural	54 Min.	1.0	.0023"
Mineral Oil Free From Added Sulphur	2.66	2.40 Natural 2.26 Free	80 Min.	2.3	.0017"

*The figures in the "Finish" column of the various tables in this Chapter are purely arbitrary, and indicate quality of finish — the higher the figure, the better the finish.

Although the addition of free sulphur to a mineral oil imparts considerable pressure-resisting characteristics and high anti-welding properties, the usefulness of such an oil is limited. Its anti-welding characteristics, unless properly adjusted, render it unsuitable for machining clean-cutting stocks, because it would otherwise rob the cutting edges of the tools of the mesager protection afforded by the slight amount of build-up present. Moreover, its lack of lubricity would result in excessive wear in the boundary area at the nose or tip of the cutting

tool. However, it is frequently employed for general-machining operations on fairly draggy stocks.

EFFECT OF SULPHURIZED FAT

The presence of sulphurized fat in a cutting oil imparts high pressure-resisting characteristics and exceptional lubricity. These properties are necessary where cutting pressures are high, and where tool vibration is likely to be excessive, due to the brittle or semi-brittle condition of the metal that is machined and the consequent spasmodic chip flow,



Such conditions are encountered in the machining of clean cutting stocks. In this service, the high pressure-resisting characteristics of the sulphurized fat minimize friction on the lip of the tool, and prevent chip seizure. Moreover, the exceptional lubricity of the cutting oil minimizes frictional heating and wear in the boundary area at the nose or tip of

the tool. The results recorded in Table IV show the superior performance of a mineral oil containing a sulphurized-fat base in the threading of the hard high-Brinell steel (S.A.E. 3140, as compared with the performance of a similar mineral oil to which free active sulphur has been added.

TABLE IV
EFFECTIVENESS OF FREE SULPHUR AND SULPHUR COMBINED WITH FAT IN VARIOUS MINERAL OILS OF SAME VISCOSITY

THREADING HARDENED S.A.E. 3140 BAR STOCK (280 B.H.N.)

Type of Oil	Total Sulphur	Tool Life	Finish	Chaser Wear Per Min.
Mineral Oil Plus Free Added Sulphur	3.16%	80 Min.	2.3	.0017"
Mineral Oil Plus Sulphurized-Fat Base	1.48%	90 Min.	4.3	.0015"

The amount of sulphurized fat for any job or range of jobs depends chiefly on the clean-cutting quality of the stock that is machined, and on the magnitude of the cutting pressures that are encountered.

However, such an oil, unless properly adjusted, is not suitable for the machining of draggy material, due to the deficient activity of the sulphur, as compared with that of free sulphur.

EFFECT OF CHLORINE

The pressure-resisting and anti-welding characteristics of a cutting oil are materially increased by the addition of chlorine. Such an oil is especially adapted, therefore, to the machining of very soft draggy metals, such as low-carbon low-Brinell steels, where the accumulation of build-up is excessive. Oils of this type promote smooth chip flow and keep the built-up edge mobile and sloughing with the chip.

The effectiveness of a sulphurized-chlorinated cutting oil in threading the soft draggy steel (S.A.E. 1020), as compared with a sulphurized-fat blend, is shown in Table V. The superior Anti-welding characteristics of the sulphurized-chlorinated oil prevented an excessive accumulation of build-up and at the same time, maintained adequate protection for the cutting edge of the tool. Moreover, the exceptional pressure-resisting characteristics minimized frictional wear and made it possible to obtain longer tool life.

The mere presence of chlorine, however, does not necessarily assure satisfactory performance, unless the chlorine is of such a form as to impart maximum pressure-resisting and anti-welding characteristics. Obviously, the amount of chlorine depends on the dragginess of the metal.

Oils of this type, unless properly adjusted, are unsuitable for machining clean-cutting stocks, such



TABLE V
EFFECTIVENESS OF SULPHURIZED FAT AND SULPHURIZED-CHLORINATED BASES IN MINERAL OILS OF SAME VISCOSITY

THREADING HOT-ROLLED S.A.E. BAR STOCK (122 B.H.N.)

Type of Oil	Tool Life	Finish	Chaser Wear Per Min.
Mineral Oil Plus Sulphurized-Fat Base	20 Min.	1.0	.0004"
Mineral Oil Plus Sulphurized-Chlorinated Base	86 Min.	3.9	.0003"

as hard high-Brinell steels, since their exceptional anti-weld properties rob the cutting edges of the tools of the protection that would otherwise be afforded by the small accumulation of build-up that would be present.

EFFECT OF VISCOSITY

The viscosity of a suitable cutting oil depends upon a proper balance between fluidity and adhesiveness—the former characteristic being required for adequate cooling, and the latter for the purpose of enabling the cutting oil to cling to the cutting tools at the speeds encountered. For example, high cutting speeds such as are employed in the cutting of the weaker non-ferrous metals, require

light-bodied quick-acting oils, in order to secure rapid penetration into the cutting areas, and to assure adequate fluidity for the removal of heat. Conversely, low cutting speeds, such as are employed in the machining of the stronger high-Brinell steels, require the use of heavy-bodied oils that will cling strongly to the tools during low-speed cutting. The viscosity of cutting oils, therefore, ranges from very light-bodied oils for high-speed light-duty cutting, to very heavy-bodied products for low-speed heavy-duty jobs.

The body of a cutting oil for any particular job or range of jobs is determined also by the volume of oil applied and by the method of applying it. Generally speaking, heavier oils are required for intermittent application than for flood application.

TABLE VI
EFFECT OF DIFFERENCE IN VISCOSITY OF OTHERWISE IDENTICAL CUTTING OILS

THREADING HARDENED S.A.E. 3140 BAR STOCK (280 B.H.N.)

Viscosity	Tool Life	Finish	Chaser Wear Per Min.
129"	60 Min.	1.0	.0025"
169"	80 Min.	1.5	.0017"
205"	18 Min.	1.5	.0065"



The various types of corrosive cutting oils that are employed for the machining of ferrous metals, are as follows:

- (a) Sulphurized Mineral Oils (i.e. mineral oils containing free added sulphur)—these products have considerable pressure-resisting characteristics and high anti-welding properties, but are deficient in lubricity.
- (b) Mineral Oils plus Corrosive Sulphurized-Fat Base—These products have high pressure-resisting characteristics and exceptional lubricity, but are somewhat deficient in anti-welding properties.
- (c) Sulphurized Mineral Oils plus Corrosive Sulphurized-Fat Base—These products have high pressure-resisting and anti-welding characteristics and exceptional lubricity properties.
- (d) Sulphurized Mineral Oils plus Chlorinated Base—These products have exceptional pressure-resisting and anti-welding characteristics, and may have exceptional lubricity if fat is present.
- (e) Mineral Oils plus Sulphurized Chlorinated Base—These products have moderately high pressure-resisting and anti-welding characteristics, and may have exceptional lubricity if fat is present.
- (f) Sulphurized Mineral Oils plus Sulphurized Chlorinated Base—these products have exceptional pressure-resisting, anti-welding and lubricity characteristics.

Each group differs from the other group in pressure-resisting, anti-welding and lubricity properties. The choice between them depends on the cutting pressure encountered and the cutting qualities of the metal or metals that are machined.

Obviously, the use of corrosive oils should be avoided in machining non-ferrous metals. Even in the cutting of ferrous metals, the use of corrosive oils on certain makes of automatic results in the corrosion of bronze gibs and slides, and renders the machines inoperative. Moreover, regardless of the type of metal, if the machined parts require a plating process after the machining operation, the presence of corrosive materials interferes with this subsequent process. Also, in many plants, it is customary to copper-plate certain portions of machined parts, in order to concentrate a subsequent carburizing process on the other portions. The use of corrosive oils affects this copper plate and renders their use undesirable. Other conditions, also, may exist, such as high humidity, the presence of moisture in the cutting-oil system, the stacking of machined parts in storage, where the use of corrosive oils is either detrimental or undesirable.

NON-CORROSIVE CUTTING OILS

Straight-mineral oils, straight-lard oils and mineral-lard oils are non-corrosive. They are good lubricants, but are deficient in the pressure-resisting and anti-welding properties necessary for long tool life and superior finish at high cutting speed. In production shops, therefore, where non-corrosive oils are required, straight-mineral oils containing a non-corrosive sulphurized fat are commonly employed. Such oils may even have appreciable amounts of chlorine. These non-corrosive cutting oils will not stain or discolor non-ferrous metals at ordinary temperatures. Even when such metals are immersed for three hours in heated oil, at a temperature of 210 degrees F., there will be only slight discoloration.¹ These non-corrosive blends possess considerable pressure-resisting, anti-welding and lubricity characteristics and are used for the machining of ferrous metals when the use of corrosive oil is either detrimental or undesirable.

Moreover, for any given metal, there is an optimum (i.e. best) viscosity for any specific cutting operation. At this optimum viscosity, maximum tool life is obtained. If lighter-bodied or heavier-bodied oils are employed, tool life is materially shortened, as shown by the test recorded in Table VI. This test involved the threading of the hard high-Britnell steel (S.A.E. 3140). The oils used were of the same type, containing the same percentage and the same kind of base, and varying only in the viscosity of the cut-back. It must not be assumed, however, that the optimum viscosity determined by this test will also be the optimum viscosity for other machining operations or for other metals.

For practical purposes, the viscosity of a cutting oil must be suited usually to a range of jobs and to a range of machining operations. Such viscosity is frequently a compromise between several optimum viscosities.

Even non-corrosive oils containing only non-corrosive sulphurized fat may stain tool holders, chucks, etc., when such parts are permitted to remain idle and coated with the oils for extended periods of time, particularly on surfaces that lie in contact with other metallic surfaces.

CORROSIVE CUTTING OILS

Corrosive cutting oils contain either sulphur or chlorine, or both, in extremely active form. Such oils will discolor or darken a polished copper strip that is immersed in the oil for a period of three hours while the oil is held at a temperature of 210 degrees F. However, they do not ordinarily stain or discolor ferrous metals. Corrosive cutting oils, therefore, are commonly used for the machining of wrought iron and steels. Such oils have high pressure-resisting and anti-welding characteristics and are, therefore, more effective than non-corrosive oils, in which the sulphur or chlorine, or both, are less active. The results recorded in Table VII show the relative effectiveness of corrosive and non-corrosive cutting oils of the same type, in the threading of the hard high-Britnell steel (S.A.E. 3140).



TABLE VII

EFFECTIVENESS OF CORROSIVE AND NON-CORROSIVE CUTTING OILS
HAVING SAME VISCOSITY AND SAME PERCENTAGE OF BASE

THREADING HARDENED S.A.E. 3140 BAR STOCK (280 B.H.N.)

Type of Oil	Tool Life	Finish	Chaser Wear Per Min.
Corrosive	60 Min.	1.5	.0025"
Non-Corrosive	40 Min.	1.0	.0030"

The lubricity of these two cutting oils was practically identical. The results, therefore, indicate nothing more than their relative pressure-resisting characteristics, and their consequent effectiveness in minimizing friction on the lip of the tool, chip seizure and lip wear. The performance of the corrosive oil was obviously superior to the performance of the non-corrosive blend in every respect—tool life, finish and tool wear.*

*Where corrosive cutting oils are used for the machining of ferrous metals, some deterioration may occur during extended periods of damp or humid weather, especially in the case of tool-holders or chucks when the various parts remain in intimate contact for long periods of time.

S V SULTRAN CUTTING OILS

For general machining operations have exceptional anti-pressure-resisting and lubricity properties. Their anti-welding characteristics are adjusted to make them all-purpose oils, such as are effective in the machining of both draggy and clean-cutting stocks. Their use on lathes, turret lathes, semi-automatics, automatics, gear hobblers, gear shapers, etc., makes it possible to maintain increased feeds and cutting speeds without the development of excessive tool wear and vibration. Their proper selection and application result in high machine production, long tool life, superior finish and low tool expense.

MATERIALS

On the following pages are listed various materials and numbers as designated by the A.I.S.I. also corresponding S.A.E. number, also showing recommended surface speeds, relative machinability and recommended cutting feed table as of chart on page . . .

It will be noted on material of less than approximately 50% machinability no cutting feed table has been applied. This does not necessarily mean that all the material in this category cannot be handled

S V VACMUL CUTTING OILS

Should be used for the general machining of all non-ferrous metals, such as copper, brass, bronze, etc. The benefits derived from their use on lathes, turret lathes, semi-automatics, automatics, gear hobblers, gear shapers, etc., are reflected in high production, long tool life, satisfactory finish and low tool cost. The S V Vacmul Cutting Oils should be recommended, instead of the S V Sultran Cutting Oils, for the machining of ferrous metals wherever conditions require non-corrosive products.

S V VACSUL CUTTING OILS

For general machining operations are like the S V Sultran Cutting Oils for general machining except that they have greater pressure-resisting, anti-welding and lubricity characteristics. Their anti-welding characteristics are adjusted to make them all-purpose oils for the machining of both draggy and clean-cutting stocks. Their use on lathes, turret lathes, semi-automatics, automatics, gear hobblers, gear shapers, etc., makes it possible to maintain maximum feeds and cutting speeds, maximum production, maximum tool life, superior finish and minimum tool expense.

on the automatic, but it is questionable as to quality and finish that may be acquired. Likewise feeds necessary to gain maximum efficiency and quality products.

On material of this nature, the turning and plunge cutting feed should be proportionately light, thus allowing the operator maximum flexibility in establishing practical feeds necessary to obtain best results. It is a common and recommended practice for cam layout purposes that the plunge cutting feed should be approximately 1/3 that of the turning feed.



STEELS

NICKEL STEELS

A.I.S.I. Number	Present S.A.E. Number	Surface Feet Per Min.	% Relative Speed Based on A.I.S.I. B1112 (S.A.E. 1112) As 100%	For Tool Feeds See Table Page 110
A2217	2217	110	66	E
A2330	2330	90	54	*
A2330 Annealed	2330	115	70	D
A2335	—	85	51	*
A2335 Annealed	—	115	70	D
A2340 Annealed	2340	95	57	*
A2345 Annealed	2345	85	51	*
E2512 Annealed	—	85	51	*
A2515 Annealed	2515	85	51	*
E2517 Annealed	—	85	51	*

MANGANESE STEELS

(Manganese in Alloy Range)

A.I.S.I. Number	Present S.A.E. Number	Surface Feet Per Min.	% Relative Speed Based on A.I.S.I. B1112 (S.A.E. 1112) As 100%	For Tool Feeds See Table Page 110
A1320	1320	95	57	*
A1330 Annealed	1330	100	60	E
A1335 Annealed	1335	100	60	E
A1340 Annealed	1340	95	57	*

CHROMIUM STEELS

A.I.S.I. Number	Present S.A.E. Number	Surface Feet Per Min.	% Relative Speed Based on A.I.S.I. B1112 (S.A.E. 1112) As 100%	For Tool Feeds See Table Page 110
A5045 Annealed	—	115	70	D
A5120	5120	125	76	D
A5130	—	95	57	*
A5140 Annealed	5140	115	70	D
A5145 Annealed	—	110	66	E
A5150 Annealed	5150	105	64	E
A5152 Annealed	—	105	64	E
E5205 Annealed	—	70	42	*
E5205B Annealed	—	65	40	*
E5205B Annealed	—	65	40	*
E52100 Annealed	52100	65	40	*
E52101 Annealed	—	65	40	*
E52107 Annealed	—	65	40	*



CHROMIUM STEELS (Continued)

A.I.S.I. Number	Present S.A.E. Number	Surface Feet Per Min.	^{1/2} Relative Speed Based on A.I.S.I. B1112 (S.A.E. 1112) As 100%	For Tool Feeds See Table Page 110
E4132 Annealed	—	120	72	D
A4134 Annealed	—	120	72	D
E4135 Annealed	—	115	70	D
A4137 Annealed	4137	115	70	D
E4137 Annealed	—	110	66	E
A4140 Annealed	4140	110	66	E
A4142 Annealed	—	110	66	E
A4143 Annealed	—	115	70	D
A4145 Annealed	4145	105	64	E
A4150 Annealed	4150	100	60	E
E4150 Annealed	—	100	60	E
A4317 Annealed	—	100	60	E
A4320 Annealed	4320	100	60	E
A4337 Annealed	—	95	57	*
E4337 Annealed	—	90	54	*
A4340 Annealed	4340	95	57	F
E4342 Annealed	—	85	51	*
A4608	—	110	66	E
A4615	4615	110	66	E
E4617	—	105	64	E
A4620	4620	110	66	E
E4620	—	105	64	E
A4621	—	110	66	E
A4640 Annealed	4660	110	66	E
E4640 Annealed	—	85	51	*
A4645 Annealed	—	85	51	*
A4815 Annealed	4815	85	51	*
A4820 Annealed	4820	80	49	*

NICKEL-CHROMIUM STEELS

A.I.S.I. Number	Present S.A.E. Number	Surface Feet Per Min.	^{1/2} Relative Speed Based on A.I.S.I. B1112 (S.A.E. 1112) As 100%	For Tool Feeds See Table Page 110
A3045 Annealed	—	110	66	E
A3115	3115	110	66	E
A3120	3120	110	66	E
A3130	3130	95	57	*
A3130 Annealed	3130	120	72	D
A3135	3135	90	54	*
A3135 Annealed	3135	115	70	D
A3140	3140	85	51	*
A3140 Annealed	3140	110	66	E
A3141	3141	80	49	*
A3141 Annealed	3141	105	64	E
A3145 Annealed	3145	105	64	E
A3150 Annealed	3150	100	60	F
A3240 Annealed	3240	95	57	*
E3310 Annealed	3310	85	51	*
E3216 Annealed	—	80	49	*



FREE CUTTING STEELS

A.I.S.I. Number	Present S.A.E. Number	Surface Feet Per Min.	% Relative Speed Based on A.I.S.I. 8112 (S.A.E. 1112) As 100%	For Tool Feeds See Table Page 110
B1008	—	130	78	D
B1011	—	130	78	D
B1110	—	140	85	C
B1111	1111	155	94	B
B1112 (Union Free Cut)	1112	165	100	B
B1113 (Union Supercut)	1113	225	136	A
C1109	—	135	81	C
C1110	—	135	81	C
C1112	—	135	81	C
C1113	—	135	81	C
C1115	1115	125	81	C
C1116	—	155	94	B
C1117	1117	140	85	C
C1118	1118	140	85	C
C1120	—	135	81	C
C1121	—	135	81	C
C1122	—	135	81	C
C1132	1132	125	76	D
C1137	1137	120	72	D
C1141	1141	115	70	D
C1144	—	120	72	D
C1217	—	140	85	C

MOLYBDENUM STEELS

A.I.S.I. Number	Present S.A.E. Number	Surface Feet Per Min.	% Relative Speed Based on A.I.S.I. 8112 (S.A.E. 1112) As 100%	For Tool Feeds See Table Page 110
A4023	4023	130	78	D
A4024	—	130	78	D
A4027	4027	110	66	E
A4028	—	120	72	D
A4032 Annealed	4032	125	76	D
A4037 Annealed	4037	120	72	D
A4042 Annealed	4042	115	70	D
A4047 Annealed	4047	110	66	E
A4063 Annealed	4063	85	51	*
A4068 Annealed	4068	80	49	*
A4119	4119	125	76	D
A4120	—	125	76	D
A4125	4125	125	76	D
A4130 Annealed	4130	120	72	D



Photo 108

CARBON STEELS

A.S.I. Number	Present S.A.E. Number	Surface Feet Per Min.	% Relative Speed Based on A.S.I. 1112 (S.A.E. 1112) As 100%	For Tool Feeds See Table Page 110
C1006	—	110	66	E
C1008	1008	110	66	E
C1010 (Light Feeds)	1010	120	—	—
C1012 (Light Feeds)	—	120	—	—
C1014	—	120	72	D
C1015	1015	120	72	D
C1016	1016	130	78	D
C1017	—	120	72	D
C1018	—	130	78	D
C1019	—	130	78	D
C1020	1020	120	72	D
C1021	—	120	72	D
C1022	1022	130	78	D
C1023	—	125	76	D
C1024	1024	120	72	D
C1025	1025	120	72	D
C1028	—	120	72	D
C1029 (Light Feeds)	—	120	—	—
C1030	1030	115	70	D
C1033	—	115	70	D
C1035	1035	115	70	D
C1036	1036	105	64	E
C1040	1040	105	64	E
C1042	—	105	64	E
C1043	—	105	64	E
C1045	1045	95	57	E
C1045 Annealed	1045	120	72	*
C1050	1050	70	54	*
C1050 Annealed	1050	115	70	D
C1052 Annealed	1052	85	51	*
C1055 Annealed	1055	85	51	*
C1060 Annealed	1060	85	51	*
C1061 Annealed	—	85	51	*
C1064 Annealed	—	80	49	*
C1065 Annealed	1066	80	49	*
C1068 Annealed	—	80	49	*
C1070 Annealed	1070	80	49	*
C1074 Annealed	—	75	45	*
C1078 Annealed	—	75	45	*
C1089 Annealed	1080	70	42	*
C1085 Annealed	1085	70	42	*
C1088 Annealed	—	70	42	*
C1095 Annealed	1095	70	42	*



CHROMIUM-VANADIUM STEELS

A.I.S.I. Number	Present S.A.E. Number	Surface Feet Per Min.	% Relative Speed Based on A.I.S.I. B1112 (S.A.E. 1112) As 100%	For Tool Feeds See Table Page 110
A6120	---	95	57	*
A6145 Annealed	---	110	66	E
D6150 Annealed	---	95	57	*
A6152 Annealed	---	100	60	E

NATIONAL EMERGENCY STEELS

A.I.S.I. Number	Surface Feet Per Minute	% Relative Speed Based on A.I.S.I. B1112 (S.A.E. 1112) As 100%	For Tool Feeds See Table Page 110
NE9024	110	66	E
NE9124	95	57	*
NE9223 Annealed	105	64	E
NE9245 Annealed	110	66	E
NE9339 Annealed	95	57	*
NE9442 Annealed	85	51	*
NE9447 Annealed	80	49	*
NE947 Annealed	75	45	*
NE9620	110	66	E
NE9630 Annealed	120	72	D
NE9724	120	72	D
NE9729 Annealed	110	66	E
NE9745 Annealed	105	64	E
NE9749 Annealed	100	60	E
NE9819	110	66	E
NE9849 Annealed	85	51	*

SILICON-MANGANESE STEELS

A.I.S.I. Number	Present S.A.E. Number	Surface Feet Per Min.	% Relative Speed Based on A.I.S.I. B1112 (S.A.E. 1112) As 100%	For Tool Feeds See Table Page 110
A9255 Annealed	---	90	54	*
A9260 Annealed	S260	85	51	*
A9262 Annealed	---	85	51	*
A9263 Annealed	---	85	51	*

STAINLESS STEELS

Symbol	A.I.S.I. Type	Surface Ft. Per Min.	For Tool Feeds See Table Page 110
Enduro 18-8 Annealed and cold drawn	302	75	*
Enduro AA Annealed and cold drawn	420	90	E
Enduro S1 Annealed and cold drawn	410	90	E
Enduro 18-8 FM	300	100	D
Enduro FC	416	150	D



GROUP 1

FREE CUTTING NON-FERROUS ALLOYS

Name	Machinability Rating
Leaded Copper—846	80
Leaded Commercial Bronze—202	80
Hardware Bronze—267	90
Leaded Red Brass 80%—205	90
Leaded Brass—211	90
Butt Brass—229	70
Free Cutting Yellow Brass—271	100
Forging Brass—250	80
Extruded Architectural Bronze—280	80
Leaded Naval Brass—612	80
Special Free Cutting Phosphor Bronze—610	90
Extruded Leaded Nickel Silver 10%—823	80

RECOMMENDED SPEEDS AND FEEDS USING HIGH SPEED STEEL TOOLS

Surface speed, from 300 to 700 feet per minute
 Feed .002" to .003"
 Cutoff .001" to .0015"

NOTE: The machinability rating as given above is based on free cutting yellow brass as being 100% machinable.

GROUP 2

READILY MACHINABLE NON-FERROUS ALLOYS

Name	Machinability Rating
Red Brass 85%—24	30
Red Brass 80%—32	30
Yellow Brass—61	40
Muntz Metal—66	40
Naval Brass—452	30
Tobin Bronze	30
Manganese Bronze—937	30
Leaded Phosphor Bronze 5%—979 (Grade B)	50
Leaded Nickel Silver 12%—796	50
Leaded Nickel Silver 18%—789	50
Evedur—1010	30
Evedur—1012 (leaded)	60
Evedur—1013	30

RECOMMENDED SPEEDS AND FEEDS USING HIGH SPEED STEEL TOOLS

Surface speed, from 150 to 300 feet per minute
 Feed .001" to .002"
 Cutoff .0005" to .001"

NOTE: The machinability rating as given above is based on free cutting yellow brass as being 100% machinable.

GROUP 3

NON-FERROUS ALLOYS WITH MACHINABILITY RATING OF 20

Name	Name
Electrolytic Tough Pitch Copper	Nickel Silver—18%—723
Deoxidized Copper—939	Ambrac—850
Commercial Bronze 90%—14	Super Nickel—701
Phosphor Bronze 4%—303 (Grade A)	Ambralay—901
Phosphor Bronze 5%—351 (Grade A)	Ambralay—928
Phosphor Bronze 8%—353 (Grade C)	Ambralay—917
Phosphor Bronze 10%—354 (Grade D)	Avialite—815
Phosphor Bronze—314	Beryllium Copper—175 (To-beat treated)
Phosphor Bronze—316	Chromium Copper—999
Nickel Silver 18%—719	

RECOMMENDED SPEEDS AND FEEDS USING HIGH SPEED STEEL TOOLS

Surface speed, from 95 to 165 feet per minute
 Feed .0007" to .0015"
 Cutoff .0005" to .0008"

NOTE: The machinability rating as given above is based on free cutting yellow brass as being 100% machinable.



Photo 111

BARS .1875 TO .375			A	B	C	D	E
Tool Name	Drill Size	Width or Depth of Cut	Feed Per Rev.	Feed Per Rev.	Feed Per Rev.	Feed Per Rev.	Feed Per Rev.
TURNING		.032	.003	.002	.0015	.001	.0009
		.045	.002	.0017	.0012	.0008	.0007
		.062	.0015	.0012	.001	.0006	.0006
		.093	.0012	.001	.0008	.0005	.0005
		.125	.001	.0007	.0006	.0004	.0004
TWIST DRILL		.062	.001	.0008	.0007	.0007	.0007
		.093	.0012	.0011	.001	.0008	.0008
		.125	.0014	.0013	.0012	.001	.001
		.156	.0016	.0014	.0014	.0012	.0012
		.187	.0018	.0015	.0015	.0014	.0014
CUT OFF		.050	.0007	.0005	.0004	.0004	.0003
		.055	.0009	.0005	.0004	.0004	.0003
		.060	.001	.0006	.0006	.0005	.0003
		.065	.0011	.0007	.0006	.0005	.0004
		.070	.0012	.0008	.0007	.0006	.0005
		.075	.0013	.0008	.0007	.0006	.0005

BARS .062 TO .1875			A	B	C	D	E
Tool Name	Drill Size	Width or Depth of Cut	Feed Per Rev.	Feed Per Rev.	Feed Per Rev.	Feed Per Rev.	Feed Per Rev.
TURNING		.010	.002	.0015	.001	.0009	.0007
		.020	.0018	.0013	.0008	.0007	.0006
		.040	.0015	.0012	.0006	.0006	.0005
		.080	.0012	.001	.0005	.0005	.0004
TWIST DRILL		.032	.0005	.0004	.0003	.0003	.0003
		.045	.0006	.0005	.0004	.0004	.0003
		.060	.0008	.0007	.0006	.0006	.0004
		.093	.001	.0009	.0008	.0008	.0005
		.125	.0012	.001	.0009	.0009	.0006
CUT OFF		.030	.0004	.0004	.0004	.0004	.0002
		.035	.0005	.0004	.0004	.0004	.0002
		.040	.0006	.0005	.0005	.0004	.0003
		.045	.0007	.0005	.0005	.0005	.0003
		.050	.0007	.0006	.0006	.0005	.0004

NOTE: The speeds and feeds in the table have been compiled partly from field experience, recommendation of the supplier or manufacturer of the raw material and composition or analysis thereof. Therefore, this table is in the greater part of a hypothetical nature, but should be of quite sufficient value and a basis in designing of the cams or calculation of the layout sheet.



COMMERCIAL STAINLESS STEELS AND CORRESPONDING A.I.S.I. TYPE NUMBER

Manufacturer's Designation	A.I.S.I. Type No.	Manufacturer's Designation	A.I.S.I. Type No.
Allegheny Ludlum Steel Corporation		USS 19 12	317
Allegheny Metal 18 8	302	USS 27	446
Allegheny Metal 18 8-8Z	303	USS 12	410
Allegheny Metal 20-10 S	308	USS 17	430
Allegheny Metal 25 12	309	USS 18-8S	304
Allegheny Metal 25 20	310	USS 22	416
Allegheny Metal 18 8 M	316	USS 4 6 Cr-Mo	501 Mo
Allegheny Metal 18 8 C	347	USS 4 6 CR T	301 T
Allegheny Metal 28	446	USS 18 8 Mo	316
Allegheny Metal 12	410	USS 16 6	301
Allegheny Metal 12 2	414	USS 18 8S Mo	316
Allegheny Metal 12 2Z	416	USS 18 8T	321
Allegheny Metal L-12	429	USS 19 8S	306
Allegheny Metal 17	430	USS 20 10S	308
Allegheny Metal 17 2Z	430 T	USS 35 20	310
Allegheny Metal 16 1	431	USS 12 2M	416
Allegheny Metal B-17	440	USS 18 8 FM	303
Allegheny Metal M-17	440	USS 19 8T1	321
Allegheny Metal 46	502	USS 25 12T1	309 T
American Rolling Mill Company		USS 4 6 Cr	501
Armco 16 6	301	USS 19 8	305
Armco 17 7	310 X	USS 20 10	307
Armco 18 8	302-304	USS 18 8T1	320
Armco 19 8	303-306	Carpenter Steel Company	
Armco 25-12	308	Carpenter No. 1	410
Armco 27	446	Carpenter No. 2	420
Armco 17	430	Carpenter No. 4	302
Armco B A	434 A	Carpenter No. 5	410 T
Armco 15	405	Carpenter No. 6	430
Armco 13	410	Carpenter No. 8	303
Bethlehem Steel Company		Crucible Steel Company of America	
Bethadur No. 1	410	Resistal No. 3	309
Bethadur No. 2	302	Resistal KA-2M	316
Bethadur No. 3	305	Resistal KA-2S	304
Bethlehem Stainless Type A	430	Resistal KA-2M	317
Bethadur No. 4	430	Resistal No. 2C	302
Bethadur No. 6	430	Resistal No. 2C	309
Bethadur No. 8	440	Resistal No. 7	310
Bethadur No. 7	440	Resistal FM 19 8	303
Bethadur No. 2A	302	Resistal KA-2S Mo	316
Bethadur No. 2B	305	Resistal KA-2ST Special 19-8	321
Bethadur No. 2C	305	Resistal Stainless B-100	440
Bethadur No. 2D	307	Resistal Stainless Iron 17	450
Bethadur No. 2E	307	Resistal Stainless Iron 12	430
Carnegie-Illinois Steel Corporation		Resistal Stainless Steel Grade A	420
United States Steel Corporation		Resistal Stainless Steel Grade B	440
USS 18 8	302	Resistal Stainless Iron FM 2	416
USS 18 8 Stabilized	303	Resistal EA-2 ST	321
USS 25 12	309	Resistal Stainless Iron 27	446
		Resistal Stainless Steel Grade B-100	440

Continued (over)



APPROXIMATE STOCK REQUIRED TO MAKE 1000 PIECES

Allowance has been made for bar end loss

Length of Finished Piece Cut Off - Inches	Number of Feet 1000 Pieces	Length of Finished Cut Off - Inches	Number of Feet 1000 Pieces	Length of Finished Cut Off - Inches	Number of Feet 1000 Pieces	Length of Finished Cut Off - Inches	Number of Feet 1000 Pieces	Length of Finished Cut Off - Inches	Number of Feet 1000 Pieces	Length of Finished Cut Off - Inches	Number of Feet 1000 Pieces
1/64	...	41/64	53.7	1-17/64	106.3	1-57/64	160.1	2-33/64	211.4	3-9/64	265.7
1/32	...	21/32	55.0	1-9/32	107.9	1-29/32	162.0	2-17/32	214.4	3-5/32	268.7
3/64	...	43/64	56.3	1-19/64	109.2	1-59/64	161.9	2-35/64	214.8	3-11/64	267.0
1/16	5.2	11/16	57.6	1-5/16	110.2	1-15/16	162.2	2-9/16	214.9	3-3/16	267.2
5/64	6.5	45/64	58.9	1-21/64	111.3	1-41/64	164.6	2-37/64	218.1	3-13/64	271.9
3/32	7.8	23/32	60.2	1-11/32	113.0	1-31/32	167.9	2-19/32	218.3	3-7/32	272.6
7/64	9.1	47/64	61.5	1-23/64	114.1	1-43/64	168.0	2-39/64	218.6	3-15/64	272.9
1/8	10.5	3/4	62.8	1-3/8	115.5	2	169.0	2-5/8	222.1	3-1/4	273.0
9/64	11.8	49/64	64.1	1-25/64	116.8	2-1/64	169.3	2-41/64	222.3	3-17/64	276.9
5/32	13.1	25/32	65.4	1-13/32	117.7	2-1/32	171.2	2-21/32	222.6	3-9/32	278.6
11/64	14.4	51/64	66.7	1-27/64	119.0	2-3/64	171.4	2-43/64	224.9	3-19/64	279.1
3/16	15.7	13/16	68.0	1-7/16	121.1	2-1/8	173.8	2-11/16	226.5	3-5/16	279.3
13/64	17.0	53/64	69.3	1-29/64	122.5	2-5/64	174.0	2-45/64	227.5	3-21/64	279.5
7/32	18.3	27/32	70.7	1-15/32	123.8	2-3/32	176.4	2-23/32	227.8	3-11/32	283.8
15/64	19.6	55/64	72.0	1-31/64	125.1	2-7/64	176.6	2-47/64	230.2	3-23/64	284.1
1/4	20.9	7/8	73.3	1-1/2	126.4	2-1/8	178.9	2-3/4	230.8	3-5/8	285.4
17/64	22.2	57/64	74.6	1-33/64	127.7	2-9/64	179.2	2-49/64	232.8	3-25/64	285.7
9/32	23.6	29/32	75.9	1-17/32	129.0	2-5/32	181.5	2-25/32	234.9	3-13/32	285.9
19/64	24.9	59/64	77.2	1-35/64	130.3	2-11/64	181.8	2-51/64	235.6	3-27/64	286.4
5/16	26.2	15/16	78.5	1-9/16	131.8	2-3/16	184.3	2-13/16	235.9	3-7/16	282.3
21/64	27.5	61/64	79.8	1-37/64	132.0	2-13/64	184.6	2-53/64	239.8	3-29/64	282.7
11/32	28.8	31/32	81.8	1-19/32	133.4	2-7/32	187.5	2-27/32	242.0	3-15/32	282.9
23/64	30.1	63/64	82.4	1-39/64	135.1	2-15/64	187.8	2-55/64	242.2	3-31/64	285.7
3/8	31.4	1	83.8	1-5/8	136.4	2-1/4	190.3	2-7/8	244.0	3-1/2	297.1
25/64	32.7	1-1/64	85.3	1-41/64	137.7	2-17/64	190.8	2-57/64	244.4	3-33/64	298.4
13/32	34.0	1-1/32	86.6	1-21/32	139.6	2-9/32	193.3	2-29/32	244.8	3-17/32	299.7
27/64	35.3	1-3/64	87.9	1-43/64	140.9	2-19/64	193.7	2-59/64	245.2	3-35/64	300.4
7/16	36.7	1-1/16	89.2	1-11/16	141.2	2-5/16	193.9	2-15/16	249.8	3-9/16	300.6
29/64	38.0	1-5/64	90.5	1-45/64	143.0	2-21/64	196.2	2-61/64	250.1	3-37/64	303.8
15/32	39.3	1-3/32	91.8	1-23/32	144.8	2-11/32	196.8	2-31/32	250.4	3-19/32	304.9
31/64	40.6	1-7/64	93.2	1-47/64	146.2	2-23/64	198.0	2-63/64	251.2	3-39/64	307.2
1/2	41.9	1-1/8	94.5	1-3/4	146.9	2-3/8	201.0	3	252.5	3-5/8	307.6
33/64	43.2	1-9/64	95.9	1-49/64	148.2	2-25/64	201.3	2-1/4	255.1	3-41/64	307.9
17/32	44.5	1-5/32	97.1	1-25/32	150.2	2-13/32	203.0	3-1/32	255.4	3-21/32	308.0
35/64	45.8	1-11/64	98.4	1-51/64	151.5	2-27/64	203.9	3-3/64	256.5	3-43/64	308.4
9/16	47.1	1-3/16	99.2	1-13/16	152.2	2-7/16	205.9	3-1/16	259.9	3-11/16	311.6
37/64	48.4	1-13/64	101.0	1-53/64	154.1	2-29/64	206.7	3-5/64	260.9	3-45/64	315.1
19/32	49.7	1-7/32	102.6	1-27/32	155.4	2-15/32	207.3	3-3/32	261.0	3-23/32	315.6
39/64	51.0	1-15/64	103.6	1-55/64	156.1	2-31/64	210.5	3-7/64	261.2	3-47/64	315.8
5/8	52.4	1-1/4	105.2	1-7/8	158.4	2-1/2	211.3	3-1/8	263.3	3-3/4	316.0

Based on 12" O" bars.



R. P. M. OF SPINDLE TO CORRESPONDING SURFACE FEET

SURFACE FEET

Dia.	5	10	20	30	40	50	60	70	80
1 16	305	610	1220	1830	2440	3050	3660	4270	4880
5 64	245	490	980	1470	1960	2450	2940	3430	3920
3 32	203	406	812	1218	1625	2031	2436	2842	3250
7 64	174	348	696	1044	1392	1740	2088	2536	2784
1 8	152	304	608	912	1216	1520	1824	2128	2432
9 64	137	274	548	822	1096	1370	1644	1918	2192
5 32	132	264	528	792	1056	1320	1584	1908	2152
11 64	112	224	448	672	896	1120	1344	1568	1812
3 16	101	202	404	606	808	1010	1212	1416	1616
13 64	94	188	376	564	752	940	1128	1316	1504
7 32	89	178	356	534	712	890	1068	1246	1424
15 64	81	162	324	486	648	810	972	1134	1296
1 4	79	158	316	474	632	790	948	1106	1264
17 64	71	142	284	426	568	710	852	994	1136
9 32	67	134	268	402	536	670	804	938	1072
19 64	64	128	256	384	512	640	768	896	1024
5 16	61	122	244	366	488	610	732	854	976
21 64	58	116	232	348	464	580	696	812	928
11 32	55	110	220	330	440	550	660	770	880
23 64	53	106	212	318	424	530	636	742	848
3 8	52	104	208	312	416	520	624	728	832
25 64	49	98	196	294	392	490	588	686	784
13 32	47	94	188	282	376	470	564	658	752
27 64	45	90	180	270	360	450	540	630	720
7 16	43	86	172	258	344	430	516	602	688



R. P. M. OF SPINDLE TO CORRESPONDING SURFACE FEET (Continued)

SURFACE FEET

90	100	200	300	400	500	600	700	800
5,490	6,100	12,200						
4,410	4,890	9,780	14,670					
3,656	4,060	8,120	12,180					
3,132	3,380	7,960	10,540					
2,736	3,040	6,080	9,120	12,160				
2,466	2,740	5,480	8,220	10,960				
2,196	2,440	4,880	7,320	9,760	12,200			
2,036	2,240	4,480	6,720	8,960	11,200			
1,818	2,020	4,040	6,060	8,080	10,100			
1,692	1,840	3,680	5,520	7,360	9,200	11,040		
1,602	1,780	3,560	5,340	7,120	8,900	10,680		
1,458	1,620	3,240	4,860	6,280	8,100	9,720	11,340	12,560
1,364	1,520	3,040	4,560	6,080	7,800	9,120	10,640	12,160
1,278	1,420	2,840	4,260	5,680	7,100	8,520	9,940	11,360
1,206	1,340	2,680	4,020	5,360	6,700	8,040	9,380	10,720
1,152	1,280	2,560	3,840	5,120	6,400	7,680	8,960	10,240
1,098	1,220	2,440	3,660	4,880	6,100	7,320	8,540	9,760
1,044	1,160	2,320	3,480	4,640	5,800	6,960	8,120	9,280
990	1,100	2,200	3,300	4,400	5,500	6,600	7,700	8,800
954	1,060	2,120	3,180	4,240	5,300	6,360	7,420	8,480
936	1,040	2,080	3,120	4,160	5,200	6,240	7,280	8,320
882	980	1,960	2,940	3,920	4,900	5,880	6,860	7,840
846	940	1,880	2,820	3,760	4,700	5,640	6,580	7,520
810	900	1,800	2,700	3,600	4,500	5,400	6,300	7,200
774	860	1,720	2,580	3,440	4,300	5,160	6,020	6,880



THREAD ELEMENTS AND TAP DRILL SIZES

AMERICAN NATIONAL FINE STANDARD THREAD (Formerly S.A.E. Standard)

Size	DIAMETERS (Inch)					THREAD DATA					TAP DRILL	
	Threads Per Inch	Major Diameter -Inches	Pitch Diameter -Inches	Minor Diameter -Inches	Minor Equivalent of Major Diameter -mm	Pitch -Inches	Depth of Thread -Inches	Basic Width of Flut -Inches	Basic Area of Thread -Inches	Tap Drill To Produce Agreed TPI, Full Thread	Desired Equipment Tap Drill -Inches	
0	80	0.060	0.0519	0.0438	1.524	0.01250	0.00912	0.00156	0.0015	1.20 mm	.0472	
1	72	.073	.0640	.0550	1.854	.01389	.00992	.00174	.0014		.53	
2	64	.086	.0759	.0657	2.184	.01562	.01015	.00185	.0014	1.80 mm	.0708	
3	56	.099	.0874	.0758	2.515	.01786	.01160	.00223	.00145	45	.0820	
4	48	.112	.0985	.0849	2.845	.02083	.01353	.00260	.0017	2.30 mm	.0905	
5	44	.125	.1102	.0955	3.176	.02273	.01476	.00284	.0018	2.60 mm	.1024	
6	40	.138	.1218	.1055	3.505	.02500	.01624	.00312	.0019	33	.1130	
8	36	.164	.1460	.1279	4.166	.02778	.01804	.00347	.0021	29	.1360	
10	32	.190	.1697	.1494	4.826	.03125	.02030	.00391	.0025	21	.1590	
12	28	.216	.1929	.1696	5.486	.03571	.02230	.00446	.0026	4.60 mm	.1811	
1/8	28	.2500	.2260	.2036	6.350	.03971	.02330		.0028	5.50 mm	.2165	
1/4	24	.3125	.2854	.2584	7.838	.04167	.02706	.00521	.0024	1	.2720	
1/4	24	.3750	.3479	.3209	9.525	.04167	.02706	.00521	.0029	8.50 mm	.3346	
3/8	20	.4375	.4050	.3725	11.113	.05000	.03248	.00625	.0030	9.90 mm	.3998	
1/2	20	.5000	.4675	.4350	12.700	.05000	.03248	.00625	.0036	11.5 mm	.4527	
1/2	18	.5625	.5264	.4903	14.288	.05556	.03608	.00694	.0038	13.0 mm	.5118	

THREAD ELEMENTS AND TAP DRILL SIZES

AMERICAN NATIONAL COARSE STANDARD THREAD (Formerly U. S. Standard)

Size	DIAMETERS (Inch)					THREAD DATA					TAP DRILL	
	Threads Per Inch	Major Diameter -Inches	Pitch Diameter -Inches	Minor Diameter -Inches	Minor Equivalent of Major Diameter -mm	Pitch -Inches	Depth of Thread -Inches	Basic Width of Flut -Inches	Basic Area of Thread -Inches	Tap Drill To Produce Agreed TPI, Full Thread	Desired Equipment Tap Drill -Inches	
1	64	0.073	0.0629	0.0527	1.854	0.01562	0.01015	0.00195	0.0022	1.45 mm	.0571	
2	56	.086	.0744	.0628	2.184	.01786	.01160	.00223	.0021	1.75 mm	.0689	
3	48	.099	.0855	.0719	2.515	.02083	.01353	.00260	.0011	2.00 mm	.0787	
4	40	.112	.0968	.0795	2.845	.02500	.01624	.00312	.0010	2.20 mm	.0866	
5	40	.125	.1088	.0925	3.175	.02500	.01624	.00312	.0067	39	.0905	
6	32	.138	.1177	.0974	3.505	.03125	.02030	.00391	.0075	36	.1065	
8	32	.164	.1437	.1234	4.166	.03125	.02030	.00391	.0120	3.40 mm	.1339	
10	24	.190	.1629	.1359	4.826	.04167	.02706	.00521	.0145	25	.1485	
12	24	.216	.1889	.1619	5.486	.04167	.02706	.00521	.0206	4.40 mm	.1732	
1/4	20	.2500	.2175	.1850	6.350	.05000	.03248	.00625	.0269	7	.2010	
1/4	18	.3125	.2764	.2403	7.938	.05556	.03608	.00694	.0454	F	.2570	
3/8	16	.3750	.3344	.2938	9.525	.06250	.04058	.00781	.0678	1/2	.3125	
1/2	14	.4375	.3911	.3447	11.113	.07143	.04620	.00883	.0503	U	.3680	
1/2	13	.5000	.4500	.4001	12.700	.07692	.04996	.00962	.1257	27.64	.4219	
3/4	12	.5625	.5084	.4542	14.288	.08333	.05413	.01042	.1620	31.64	.4844	
1	11	.6250	.5660	.5069	15.875	.09091	.05905	.01136	.2018	17.32	.5312	
1 1/8	10	.7500	.6850	.6201	18.050	.10000	.06495	.01250	.3020	18.5 mm	.6486	
1 1/4	9	.8750	.8028	.7307	22.225	.11111	.07217	.01389	.4193	49.64	.7656	



Photo 118

DECIMAL EQUIVALENT OF THE NUMBERS OF TWIST DRILL AND STEEL WIRE GAGE

No.	Size of No. in Decimals	No.	Size of No. in Decimals	No.	Size of No. in Decimals	No.	Size of No. in Decimals	No.	Size of No. in Decimals
1	.2280	17	.1730	33	.1130	49	.0730	65	.0350
2	.2210	18	.1685	34	.1110	50	.0700	66	.0330
3	.2130	19	.1660	35	.1100	51	.0670	67	.0320
4	.2090	20	.1610	36	.1065	52	.0635	68	.0310
5	.2055	21	.1590	37	.1040	53	.0595	69	.0292
6	.2040	22	.1570	38	.1015	54	.0550	70	.0280
7	.2010	23	.1540	39	.0995	55	.0520	71	.0260
8	.1990	24	.1520	40	.0980	56	.0465	72	.0250
9	.1960	25	.1495	41	.0960	57	.0430	73	.0240
10	.1935	26	.1470	42	.0935	58	.0420	74	.0225
11	.1910	27	.1440	43	.0950	59	.0410	75	.0210
12	.1890	28	.1405	44	.0860	60	.0400	76	.0200
13	.1850	29	.1360	45	.0820	61	.0390	77	.0180
14	.1820	30	.1285	46	.0810	62	.0380	78	.0160
15	.1800	31	.1200	47	.0785	63	.0370	79	.0145
16	.1770	32	.1160	48	.0760	64	.0360	80	.0135



Photo 119

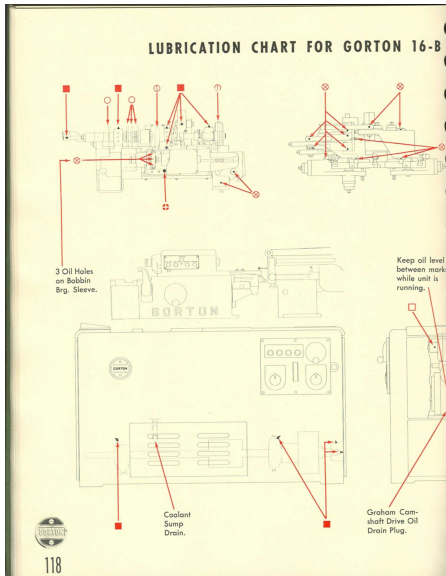
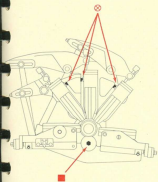


Photo 120

PRECISION AUTOMATIC SCREW MACHINE



SCHEDULE FOR LUBRICATION

- | | |
|---|---|
| ↔ | Keep reservoir filled. Use Vacuoline Oil 1409. Close air vent to stop oil feed. |
| ■ | Grease once a week. |
| ⊙ | Oil twice a day. Use light machine oil. |
| ○ | Oil cam profiles. Use S. A. E. No. 40 oil once a day. |
| ● | *Oil twice a week. |
| ◆ | Keep oil level up to plug. Use S. A. E. No. 40 oil. |
| ◻ | *Grease every 1000 hours. Do not over-grease. |
| ⊕ | Keep reservoir filled. Use S. A. E. No. 40 oil. |

*Consult spindle drive motor instruction card wired on drive motor for recommended motor bearing grease and oil.

Special Oil Filler Plug
See Graham Instruction
Book in Folder.

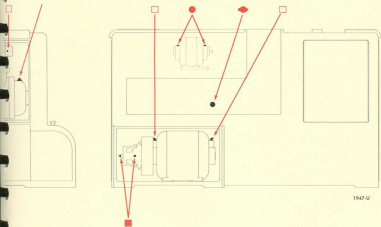


Photo 122

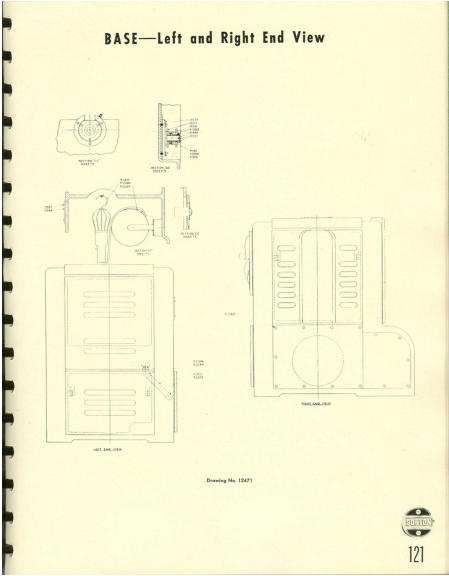
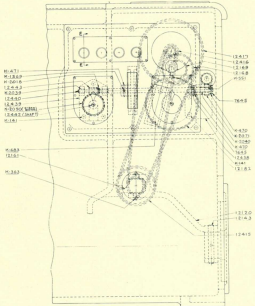


Photo 123

CONTROLS AND



RIGHT FRONT VIEW Drawing No. 12471



Photo 125

COOLANT PUMP, CHAINS, CUTLER-HAMMER PANEL AND MAIN DISCONNECT SWITCH

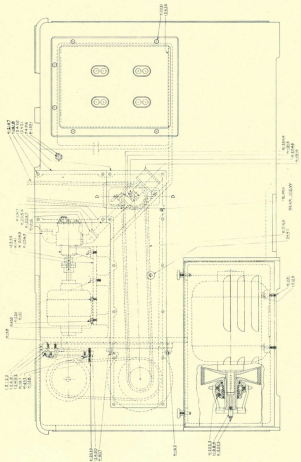


Photo 127

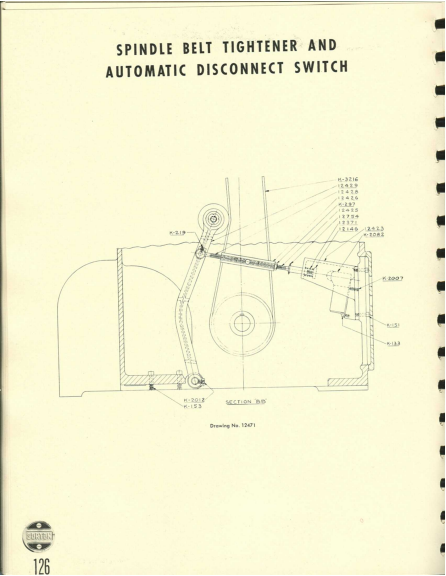


Photo 128

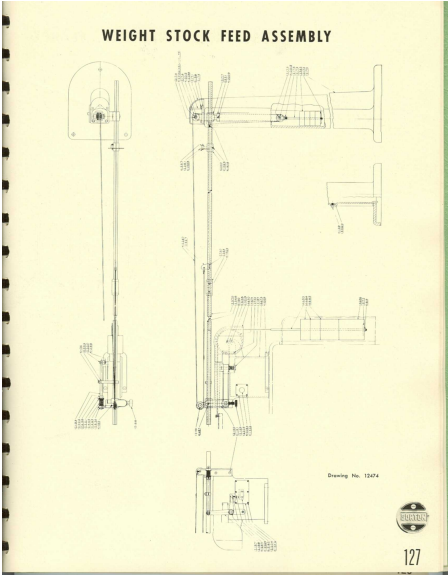


Photo 135

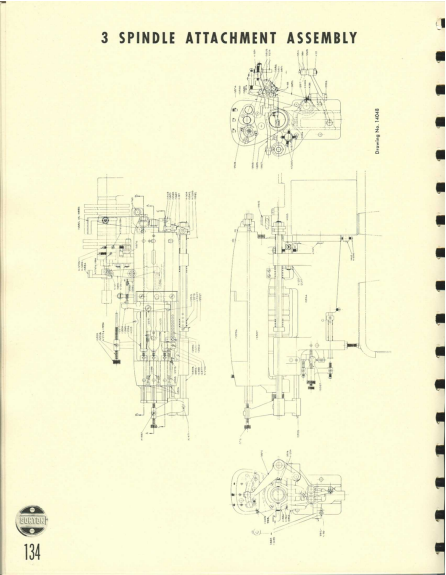
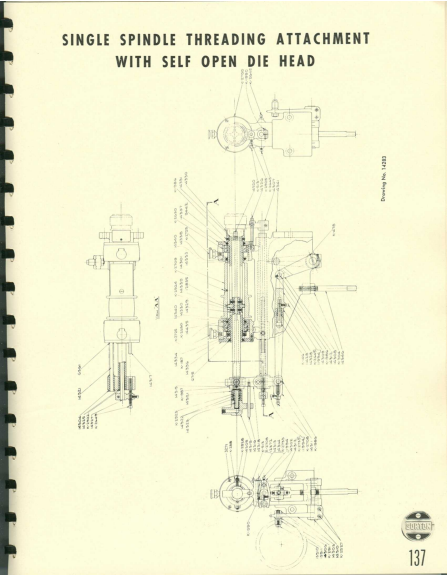


Photo 138



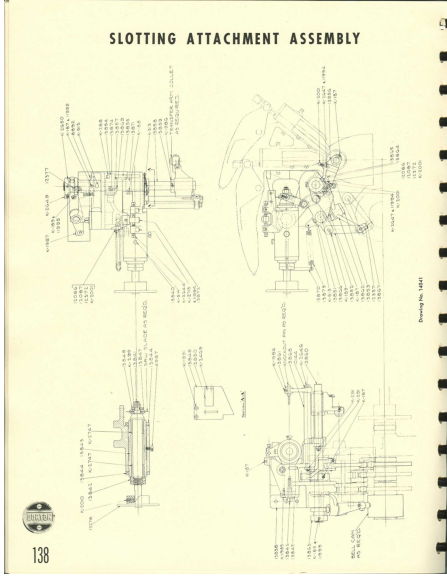


Photo 141

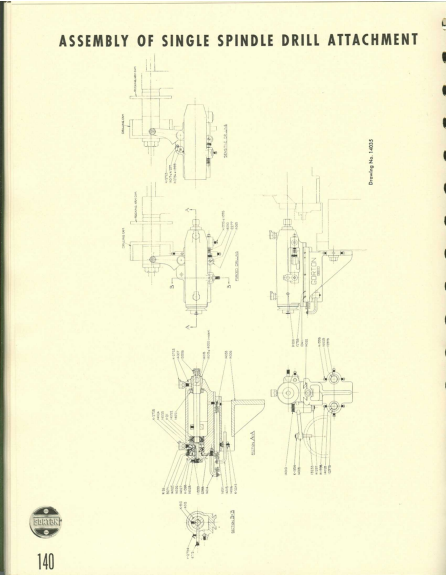


Photo 142

PARTS LIST FOR GORTON AUTOMATIC MODEL 16-B

Part No.	Name	Part No.	Name
3271	3/16" x 32 x 17/32" Filler Head Screw	12122	Camshaft Drive Mounting Plate
3336	5/16" x 24 Hexagon Nut	12126	Coolant Pump Cover
7645	Collar	12127	Sump Pan
7887	Screws for Medallion	12130	Camshaft Drive Outboard Bearing Housing
8951	Collar	12131	Attachment Hole Cover
9219	Collar	12137	Main Drive Shaft
9339	Lump Bracket Clamp	12139	No. 1-2 Radial Micrometer Screw
9445	Oil Hole Lever Plug	12141	Reeves Drive Inset Spacer
9496	Light Stud	12142	Control Screw Link Pin
9507	Spring	12143	Control Screw Lever Pivot Pin
9614	Dowel Pin for Bronze Bearing	12146	Belt Tightener Screw End
9923	3/16" x 1" Straight Pin	12148	Belt Tightener Yoke Shaft
9963	Plug	12157	Belt Tightener Yoke Bracket
11587	Plug	12160	Control Screw
11984	Groove Fitting Adapter	12161	Control Screw Nut
11994	1/8" Dia. x 1/32" Brass Disc	12162	Control Screw Indicator Sprocket
11995	3/16" Dia. x 1/16" Brass Disc	12164	Control Screw Cover Plate
12069	No. 3 Tool Frame	12166	Handwheel
12071	No. 3 Tool Slide Bracket	12167	Dial Indicator
12072	No. 4-5 Tool Slide Bracket	12168	Spindle Control Intermediate Gear—Large
12074	Tool Holder	12169	Spindle Control Intermediate Gear—Small
12075	Vertical Micrometer Screw	12172	Stack Feed Weight Rod Plate
12076	Vertical Micrometer Dial	12182	Spindle Control Dial Gear
12077	Micrometer Dial Indicator	12184	Spindle Indicator Shaft Collar
12078	Centering Block Stock	12187	Stack Support Lever Stud
12079	Centering Block	12189	Lever Stud Lock Screw
12090	Tool Slide Gb	12205	Pilot Light Mounting Plate
12081	Horizontal Micrometer Screw	12210	Control Wheel Knob Stud
12082	Horizontal Micrometer Dial	12211	Camshaft Drive Sprocket Core
12083	Tool Slide Cap	12212	Camshaft Drive Intermediate Sprocket—Outer
12084	Micrometer Dial Indicator	12216	Camshaft Drive Intermediate Sprocket—Inner
12085	Micrometer Screw Collar	12217	Camshaft Drive Upper Sprocket
12086	Clamping Sleeve—Nut	12218	Camshaft Drive Intermediate Sprocket Bushing
12087	Clamping Sleeve—Short	12219	Camshaft Drive Intermediate Sprocket Bracket
12088	Clamping Sleeve—Long	12220	Camshaft Drive Intermediate Sprocket Collar
12089	Rocker Arm Pin	12221	No. 1 Tool Holder
12090	No. 4 Rocker Pin	12222	No. 2 Tool Holder
12091	No. 5 Rocker Pin	12223	No. 1-2 Axial Micrometer Screw Stud
12095	No. 5 Rocker Arm	12232	Adjustable Bushing Sleeve Screw
12096	No. 4 Rocker Arm	12234	Coolant Pump Plate
12097	No. 3 Rocker Arm	12235	Slotted Attachment Hole Cover
12098	Rocker Arm Link Pin	12243	Collar Closing Bearing Stud
12099	No. 3 Rocker Pin	12247	Cam Spacer—2 M. M.
12100	No. 3 Link	12248	Cam Spacer—8 M. M.
12101	No. 4 Link	12249	Cam Spacer—10 M. M.
12102	No. 5 Link	12250	Cam Spacer—25 M. M.
12103	Link Indicator	12251	Cam Spacer—50 M. M.
12104	Link Stud Washer	12253	Camshaft Drive Outboard Bearing
12105	Link Stud	12254	Camshaft Rear Drive Shaft
12110	Main Shaft Bearing Housing—Left	12255	Camshaft Drive Warm Bearing Nut
12111	Main Shaft Bearing Retainer—Left	12259	Camshaft Drive Warm Bearing Retainer
12112	Main Shaft Bearing Housing—Right	12263	Camshaft Shift Rod Fork
12113	Main Shaft Bearing Retainer—Right	12264	Camshaft Shift Rod
12114	Lower Left End Cover	12280	Spindle Shoe Holder Screw
12116	Upper End Cover—Right or Left	12281	Spindle Shoe Holder
12117	Lower Right End Cover	12289	Spindle Shoe
12119	Control Screw Link	12290	No. 3 Tool Frame Dowel Pin
12120	Control Screw Lever		
12121	Front Cover		



PARTS LIST FOR GORTON AUTOMATIC MODEL 16-B—Continued

Part No.	Name	Part No.	Name
12296	Chuckling Sleeve—Large Collet	12426	Belt Tightener Turnbuckle
12297	Chuckling Sleeve—Small Collet	12427	Belt Tightener Eye Bolt Stud
12298	Chuckling Sleeve Drive Key	12428	Belt Tightener Eye Bolt
12299	Toggle Equalizer	12429	Idler Pulley Yoke
12300	Chuckling Sleeve Tube	12430	Cable Clips
12301	Chuckling Sleeve Spring—Large Collet	12431	Spindle Motor Cover Knob
12302	Chuckling Sleeve Spring—Small Collet	12432	Spindle Motor Cover Knob Stud
12303	Toggle Key	12433	Spindle Motor Cover
12304	Bobbin Bearing Housing	12435	Reset Box Plate
12305	Bobbin Bearing Housing Cover	12436	Radial Micrometer Screw Spring
12306	Bobbin Bearing Pivot Pin	12437	Sump Pan Screen Knob
12307	Bobbin Bearing Sleeve	12438	Camshaft Control Spiral Gear Bracket
12308	Bobbin Bearing Sleeve Nut	12439	Camshaft Control Worm Gear Bracket
12309	Toggle Holder Finger	12440	Camshaft Control Worm Shaft
12310	Toggle Holder Hinge Pin	12441	Camshaft Control Spiral Gear Shaft
12311	Toggle Holder	12442	Camshaft Control Indicator Shaft
12312	Toggle Holder Sleeve	12443	Camshaft Dial Indicator Shaft Worm
12313	Toggle Adjusting Nut	12444	Spindle Indicator Shaft
12316	Front Spindle Bearing Lock Nut	12445	Camshaft Control Wheel
12320	Rear Spindle Bearing Spacer	12446	Spindle Control Intermediate Gear Stud
12321	Rear Spindle Bearing Oil Reservoir	12447	Handwheel Knob Extension
12322	Spindle Pulley	12448	Spindle Indicator Shaft Spacer
12323	Spindle Pulley Nut	12450	Spindle Speed Plate
12324	Bobbin Bearing Lever	12452	Idler Pulley
12325	Bobbin Bearing Bell Crank Lever	12453	Idler Pulley Shaft
12326	Bobbin Bearing Bell Crank Lever Stud	12457	Vertical Micrometer Set Screw
12327	Bobbin Bearing Lever Shaft	12460	Spindle Fullback Spring Stud
12332	Collet Opening Counterweight Lever Stud	12468	Attachment Cover Gasket
12333	Collet Opening Counterweight Lever Pin	12483	Coolant Line Nozzle
12334	Collet Opening Counterweight Lever Pin End	12490	Camshaft Motor Hinge Plate
12335	Collet Opening Pin	12491	Camshaft Motor Hinge
12336	No. 1—2 Axial Micrometer Screw Collar	12492	Camshaft Motor Hinge Rod
12337	Rocker Arm Roller	12493	Worm Gear Bearing Nut
12338	Rocker Arm Roller Pin	12494	Camshaft Speed Plate
12339	Counter Cam Roller	12589	3/16" x 5/8" Straight Pin
12341	Tool Slide Spring	12394	1/8" x 7/16" Straight Pin
12342	Tool Slide Spring Pin	12607	1/4" x 1-1/4" Drill Rod
12347	No. 1 Tool Slide Gib	12608	3/16" x 1-1/4" Drill Rod
12348	No. 2 Tool Slide Gib	12612	5/16" x 1" Drill Rod
12349	Main Drive Shaft Collar	12636	3/16" x 7/8" Drill Rod Pin
12356	No. 1—2 T Adjusting Bracket Stop Stud	12668	5/16" x 13/16" Drill Rod Pin
12357	No. 1—2 Tool Rocker Spring Anchor—Lower	12732	Pin in Guard 1/4" x 1-3/8" Drill Rod Pin
12371	Belt Tightener Spring	12751	2785" x 7/32" Straight Pin
12372	1/4" Washer (3-4-5 Clamp Screws)	12752	1/8" x 9/16" Straight Pin
12376	No. 1—2 Tool Rocker Spring	12753	1/8" x 7/8" Straight Pin
12378	Sump Pan Screen	12754	1/8" x 1" Straight Pin
12399	Spindle Nose—Small Collet	12757	1/4" x 1-1/8" Straight Pin
12400	Spindle Control Intermediate Gear Shim	12767	1/8" x 15/16" Straight Pin
12415	Control Screw Sprocket—Large	12866	5/16" x 1-1/4" Straight Pin
12416	Spindle Control Shaft Sprocket	12924	5/32" x 3/8" Straight Pin
12417	Spindle Control Idler Shaft Sprocket	13129	3/8" Hardened Washer
12418	Spindle Control Idler Shaft	13200	Adjustable Bushing Screw—Large
12419	Spindle Control Shaft	13201	Adjustable Bushing Sleeve—Large
12420	Camshaft Chain Cover—Vertical	13203	Adjustable Bushing Sleeve Spacer—Large
12421	Camshaft Chain Cover	13204	Adjustable Bushing Sleeve—Small
12422	Spindle Control Shaft Outboard Bearing	13205	Adjustable Bushing Sleeve Spacer—Small
12423	Belt Tightener Switch Bracket	13206	Spindle Control Handwheel Spacer Stud
12424	Camshaft Drive Cover	13238	No. 1—2 Rocker Toe
12425	Belt Tightener Screw	13239	No. 1—2 Rocker Toe Holder



PARTS LIST FOR GORTON AUTOMATIC MODEL 16-B—Continued

Part No.	Name	Part No.	Name
13444	Rear Bearing Felt Retainer	14660	Counterweight Bracket
13786	Rear Bearing Felt Retainer—Front	14661	Front Spindle Bearing Retainer
13787	Rear Bearing Felt Retainer—Rear	14662	Front Spindle Bearing Retainer—Felt
13796	Adjustable Oil Wiper	14663	Front Spindle Bearing Retainer—Inner
13797	Spindle Housing Oil Drip Tube	14664	Spindle Housing Felt—Front
14247	Belt Tightener Yoke Bracket Bushing	14665	No. 3—4 Cam Spacer
14350	Revs Drive Outer Spacer	14666	Camshaft
14600	Tool Frame	14667	Control Plate
14601	Spindle Housing Glb	14668	Warm Gear Cover—Left Side
14602	Collet Closing Roller Lever	14669	Warm Gear Cover—Right Side
14603	Collet Opening Toe Lever	14670	Weight Pulley Guard
14604	No. 3 Adjusting Screw	14671	Weight Pulley
14605	No. 4 Adjusting Screw	14672	Counter Bracket
14606	No. 5 Adjusting Screw	14673	Stock Support
14607	Bed	14674	Upper End Cover Right
14608	Camshaft Outboard Bearing Bracket	14675	Rocker Adjusting Stud
14609	Spindle Housing	14676	Rocker Stud Bushing
14610	Spindle Housing Glb	14677	No. 1 & 2 Tool Rocker Pivot Pin
14611	Collet Opening Lever	14678	Needle Oil Valve
14612	No. 1—2 Tool Rocker	14679	Adjusting Bushing Holder—Large
14613	No. 3 Cam Adapter	14681	1" Dia. x 3" 8" Long Pin
14614	Slotted Attachment Cam Adapter	14682	Spindle Drive Pulley
14615	Cam Locker Spacer	14683	Spindle Housing Plate
14616	Camshaft Bearing	14684	Spindle Housing Weight—5 lb.
14617	Slotted Attachment Cam	14685	Spindle Housing Weight—10 lb.
14618	Cam Locking Collar	14686	Spindle Housing Weight—15 lb.
14619	Tool Rocker	14692	Spindle Return Weight Bad
14620	Tool Rocker	14693	Spindle Return Weight Cable Assembly
14621	No. 4 Tool Rocker	15251	Tool Rocker Spring Anchor
14622	No. 3 Tool Slide	15255	Terminal Box Cover
14623	No. 4 and No. 3 Tool Slide	15257	Washer for 15163 Assembly
14624	Camshaft Shim	K-31	Patric M-204-W-I-D-8
14627	Camshaft Thrust Washer	K-34	Ball Bearing
14628	4 and 5 Cam Adapter	K-66	New Departure No. 77038-ZT
14629	Hand Control Shaft Clutch	K-76	1" 4" Dia. Steel Ball
14630	Camshaft Hand Control Shaft	K-85	Ball Bearing Nut
14631	Camshaft Drive Worm Clutch—Front	K-91	Ball Bearing Nut Washer—W-03
14632	Camshaft Drive Worm Clutch—Rear	K-92	Ball Bearing Lock Washer—W-07
14633	Rear Drive Shaft	K-133	1" 4" 20 x 1" 2" Socket Head Cap Screw
14634	Warm Gear Cover	K-136	1" 4" 20 x 7" 8" Socket Head Cap Screw
14635	Camshaft Hot Gear	K-137	1" 4" 20 x 1" 2" Socket Head Cap Screw
14636	Camshaft Drive Worm Shaft	K-138	1" 4" 20 x 1-1/4" Socket Head Cap Screw
14637	Spindle Housing Cover	K-141	5/16" 18 x 3/4" Socket Head Cap Screw
14638	Spindle	K-142	5/16" 18 x 4" Socket Head Cap Screw
14639	Control Panel Cover	K-144	5/16" 24 x 1" Socket Head Cap Screw
14640	No. 1 Tool Slide	K-145	5/16" 18 x 1" 8" Socket Head Cap Screw
14641	No. 2 Tool Slide	K-146	5/16" 18 x 1-1/4" Socket Head Cap Screw
14642	Collet Opening Lever Shaft	K-148	5/16" 18 x 1" 2" Socket Head Cap Screw
14643	Collet Opening Counterweight Lever	K-149	5/16" 18 x 2" Socket Head Cap Screw
14644	Counter Cam	K-151	3" 8" 16 x 1" 2" Socket Head Cap Screw
14645	Camshaft Outboard Bearing Bushing	K-152	3" 8" 16 x 1-1/4" Socket Head Cap Screw
14646	Spindle Housing Stop Bracket	K-153	3" 8" 16 x 1-1/2" Socket Cap Screw
14648	Tool Slide Glb. Screw—Short	K-161	1" 2" 12 x 1-1/2" Socket Head Cap Screw
14649	Tool Slide Glb. Screw	K-175	7/16" 14 x 1-1/4" Socket Head Cap Screw
14650	Camshaft Nut	K-187	1" 4" 20 x 1/4" Flat Point Socket Set Scr.
14651	Base	K-189	1" 4" 20 x 5/16" Flat Point Socket Set Scr.
14656	Bearing Cap	K-190	1" 4" 20 x 3/8" Cup Point Socket Set Scr.
14657	No. 1 & 2 Adjusting Bracket	K-191	1" 4" 20 x 1/2" Cup Point Socket Set Scr.
14659	Air Vent Screw	K-192	1" 4" 20 x 5/8" Flat Point Socket Set Scr.



PARTS LIST FOR GORTON AUTOMATIC MODEL 16-B—Continued

Part No.	Name	Part No.	Name
K-197	5/16"-18 x 1/2" Cup Point Socket Set Scr.	K-1920	No. 00 x 1" Taper Pin
K-198	5/16"-18 x 1/2" Cup Point Socket Set Scr.	K-1982	No. 4-40 x 1/4" Round Head Machine Screw
K-199	5/16"-18 x 1/2" Cup Point Socket Set Scr.	K-1984	No. 8-32 x 3/16" Socket Head Cap Screw
K-219	3/8"-16 x 1/2" Flat Point Socket Set Scr.	K-1985	No. 8-32 x 1/2" Socket Head Cap Screw
K-256	5/16"-18 x 1/2 Hexagon Head Cap Scr.	K-1986	No. 10-32 x 1/4" Flat Point Socket Set Screw
K-258	5/16"-18 x 3/4" Hexagon Head Cap Scr.	K-1988	No. 10-32 x 3/8" Flat Point Socket Set Screw
K-260	5/16"-18 x 1" Hexagon Head Cap Scr.	K-1994	1/4"-20 x 3/8" Flat Point Square Head Set Screw
K-271	3/8"-16 x 1/3, 4" Hexagon Head Cap Scr.	K-1995	1/4"-20 x 1/2" Flat Point Square Head Cap Screw
K-287	10-32 Hexagon Machine Nut	K-1998	1/4"-20 x 3/4" Flat Point Square Head Set Screw
K-291	5/16"-18 Hexagon Nut	K-1999	1/4"-20 x 7/8" Square Head Set Screw
K-297	3/8"-14 Hexagon Nut	K-2000	1/4"-20 x 1" Cup Point Square Head Set Screw
K-363	No. 10-32 x 1/2" Flat Head Machine Scr.	K-2001	1/4"-20 x 1-1/2" Flat Point Square Head Set Screw
K-373	No. 10-32 x 3/8" Filler Head Mach. Scr.	K-2002	1/4"-20 x 1-1/2" Oval Point Square Head Set Screw
K-377	1/4"-20 x 1/2" Round Head Mach. Scr.	K-2004	1/4"-20 x 3/8" Cup Point Square Head Set Screw
K-394	No. 10-32 x 3/8" Round Head Mach. Scr.	K-2006	1/4"-20 x 1/2" Flat Point Socket Set Screw
K-395	No. 10-32 x 1/2" Round Head Mach. Scr.	K-2007	1/4"-20 x 5/8" Cup Point Socket Set Screw
K-405	1/8" Square Head Pipe Plug	K-2009	5/16"-18 x 5/16" Flat Point Socket Set Screw
K-406	1/8" Socket Pipe Plug	K-2010	5/16"-18 x 3/4" Flat Point Socket Set Screw
K-411	3/8" Socket Pipe Plug	K-2012	3/8"-16 x 5/8" Flat Point Socket Set Screw
K-423	No. 6-5 1/6" Round Head Drive Pin	K-2013	5/8"-11 x 5/8" Flat Point Socket Set Screw
K-441	5/16" Spring Lock Washer	K-2018	No. 000 x 1" Taper Pin
K-453	5/16" Standard Rough Washer	K-2020	No. 00 x 1-1/2" Taper Pin
K-454	3/8" Rough Washer	K-2021	No. 2 Woodruff Key
K-461	No. 0 x 1" Taper Pin	K-2021	No. 6 Woodruff Key
K-462	No. 00 x 3/4" Taper Pin	K-2023	Grease Fitting—Alenite No. 1610
K-470	No. 2 x 1" Taper Pin	K-2025	Grease Fitting—Alenite No. 1925
K-471	No. 2 x 1-1/2" Taper Pin	K-2026	5/16"-18 x 7/8" Socket Head Cap Screw
K-479	No. 3 x 1-1/2" Taper Pin	K-2027	5/16"-18 x 1-3/4" Socket Cap Screw
K-480	No. 3 x 1-3/4" Taper Pin	K-2029	3/8"-16 x 7/8" Socket Head Cap Screw
K-486	No. 4 x 1-5/8" Taper Pin	K-2032	5/16"-18 x 1-1/4" Flat Point Square Head Set Screw
K-487	No. 4 x 1-3/4" Taper Pin	K-2033	5/16"-18 x 1" Flat Point Square Head Set Screw
K-488	No. 4 x 2" Taper Pin	K-2034	Oilite Bushing A-1130
K-513	Gits Oiler No. 301	K-2035	Oilite Bushing A-1108-1
K-514	Gits Oiler No. 302	K-2036	Oilite Bushing A-1054-5
K-547	No. 95 Dayton Block Bolalite Ball	K-2038	Oilite Bushing A-604-1
K-551	No. 3 Woodruff Key (1/8" x 1 1/2")	K-2039	Oilite Bushing A-411
K-553	No. 5 Woodruff Key (1/8" x 5/8")	K-2040	Oilite Bushing A-604-4
K-555	No. 9 Woodruff Key (3/16" x 3/4")	K-2042	No. 1920 Alenite Grease Fitting
K-556	No. 11 Woodruff Key (3/16" x 7/8")	K-2043	No. 6575 Alenite Grease Gun
K-664	5/16"-18 x 1" Thumb Wing Head Screw	K-2044	1/2" Swing Valve
K-666	"Gorton" Medcillon	K-2046	3/4" Male-1/2" Female Reducer
K-683	3/8"-16 x 5/8" Socket Head Cap Screw	K-2047	1/2"-45° Elbow
K-685	Lamp	K-2049	3/4" Street Elbow
K-810	1/4"-20 x 3/8" Full Dog Point Set Screw	K-2050	1/2" Male-3/8" Female Reducer
K-819	5/16"-18 Hexagon Nut	K-2052	3/8" Tee
K-1346	No. 6-32 x 3/8" Round Head Machine Screw	K-2058	3/8" Nipple—7" Long
K-1350	Ball Bearing Nut N-07	K-2059	No. 216 Garlock Closure
K-1351	9/16"-12 Hexagon Half Nut	K-2060	No. 149 Garlock Closure
K-1354	5/16"-18 Hexagon Half Nut	K-2061	No. 390 Garlock Closure
K-1517	1/4"-20 Hexagon Half Nut		
K-1569	1/4"-20 x 3/8" Flat Point Socket Set Screw		
K-1618	1/4"-20 x 5/8" Socket Head Cap Screw		
K-1762	No. 8-32 x 3/4" Round Head Machine Screw		
K-1763	No. 8-32 Hexagon Full Nut		
K-1888	7/16"-14 x 1" Hexagon Head Cap Screw		
K-1915	3/4" Close Couple		
K-1916	Coolant Oil Strainer		
K-1917	1/2" Nipple—6" Long		
K-1918	5/16"-24 x 5/8" Oven Hard Machine Screw		



PARTS LIST FOR GORTON AUTOMATIC MODEL 16-B—Continued

Part No.	Name	Part No.	Name
K-2062	No. 151 Garlock Closure	K-3228	Nipple
K-2063	No. 546 Garlock Closure	K-3229	Tree
K-2067	1-1/4" Dia. Expansion Plug	K-3230	Plug
K-2068	1/4"-20 x 1/4" Flat Point Slotted Set Screw	K-3234	Coolant Pump
K-2069	1/4"-20 x 3/8" Filler Head Machine Screw—Low Clear	K-3232	Gasket
K-2070	Q-1344 Boston Gear	K-3239	5/16"-24 x 7/8" Slotted Half Hog Point Set Screw
K-2071	H-1620-R Boston Spiral Gear	K-3260	1-1/8" Dia. x 1-1/16" Glass Disc
K-2079	"Stop" Pushbutton Element	K-3308	No. 10-32 x 1/4" Oval Head Machine Screw
K-2080	"Run" Pushbutton Element	K-3613	Security Lock Nut
K-2081	No. 2 Position Throw Over Switch		
K-2082	Limit Switch		
K-2083	Pilot Light		
K-2088	Graham Transmission		
K-2162	BZE-RQ Micro-Switch		
K-2163	No. 5-D-1 Production Counter	12224	Adjustable Bushing Sleeve Screw
K-2164	No. 1532 Asplaton Portable Buellie	12216	Front Spindle Bearing Lock Nut
K-2165	No. 4000-C Reeves Motor Drive	12924	5/32" dia. x 3/8" Straight Pin
K-2166	No. 66 Diamond Chain—126 Pitches, 1/2" Pitch—With Connection Link	12200	Adjustable Bushing Screw—Large
K-2167	No. 66 Diamond Chain—98 Pitches, 1/2" Pitch—With Connection Link	12001	Adjustable Bushing Sleeve—Large
K-2168	No. 82 Diamond Chain—124 Pitches, 3/8" Pitch—With Connection Link	12203	Adjustable Bushing Sleeve Spacer—Large
K-2169	No. 82 Diamond Chain—109 Pitches, 3/8" Pitch—With Connection Link and Off-Set Link	12204	Adjustable Bushing Sleeve—Small
K-2172	No. 2480—10-3" Dia. Gates Belt	12205	Adjustable Bushing Sleeve—Small
K-2173	VSA—4-0" Dia. Gates Pulley	14679	Adjustable Bushing Holder
K-2174	VSA—5-0" Dia. Gates Pulley		
K-2179	Fafnir No. 207-L Bearing	6507	3/8" Washer
K-2180	Fafnir No. 1-607 Bearing, 1-13/16" Bore	11587	Oil Hole Plug
K-2181	No. 203-LL Fafnir Bearing	11990	3/16" x 1-7/8" Drill Rod Pin
K-2182	No. 2205 Fafnir Bearing	11994	1/8" x 1/32" Brass Disc
K-2183	No. 206-LL Fafnir Bearing	11999	1/8" x 1/16" Brass Disc
K-2184	No. 1-6308 Fafnir Bearing	12356	No. 1 & 2 Tool Adj. Brkt. Stop Stud
K-2260	Shield & Socket	12377	No. 1 & 2 Tool Rocker Spring Anchor—Upper
K-2291	Can of Oil for Graham Transmission	25589	3/16" x 5/8" Pin
K-2292	No. 1025 Williams End Wrench	12874	1/8" x 5/8" Drill Rod Pin
K-2294	721-A Billings & Spencer End Wrench	13233	Attachment Spindle Advance Toe
K-2313	3/8"-11 x 5/8" Flat Point Socket Set Screw	13749	Trip Lever Rear Spring
K-2332	5/16"-24 x 3/8" Flat Point Socket Set Screw	13760	Trip Lever Rear Spring Anchor
K-2360	1/4"-20 x 1-1/4" Flat Point Slotted Set Screw	13761	Spindle Advance Arm Pin
K-2361	1/4"-20 x 2" Flat Point Slotted Set Screw	13875	Shift Bearing Housing
K-2393	1/2" Bonnet	13876	Spindle Advance Arm
K-2396	1/2" Close Nipple—1-1/8" Long	13877	Clutch Shaft Front Bearing
K-2505	7/16"-12 x 1-1/2" Socket Cap Screw	13878	Spindle Bearing Housing
K-2644	1/4"-20P x 1-1/8" Square Head Dog Point Set Screw	13879	End Trip Lever
K-2733	No. 10-32 x 3/8" Flat Point Slotted Set Screw	13880	Shift Fork
K-2764	Coolant Pump Coupling	13881	Threading Spindle Support
K-2935	Lubrication Oil	13882	Threading Cam
K-3138	1/2" Dia. x 2" Long Dowel Pin	13884	Reverse Spring Loading Cam
K-3168	1/2" Pipe Elbow—45° Angle	13885	Drilling Cam Roller Lever
K-3216	1-1/4" x 90-1/2" No. 310 Endless Belt	13886	Centering Cam Roller Lever
K-3219	Bushing In Pulley	13887	Oil Guard Fork
K-3224	Nipple	13888	Cam Advance Arm Bracket
K-3225	Coolant Line	13889	Vertical Trip Lever
K-3226	Coolant Line Valve	13890	Horizontal Trip Lever
K-3227	Nipple	13891	Oil Cover
		13892	Spindle Shift Lever
		13893	Cams Trip Lever
		13894	Upper Trip Lever

Revolving Bushing Holder for 16-B

3-Spindle Attachment



PARTS LIST FOR GORTON AUTOMATIC MODEL 16-B—Continued

Part No.	Name	Part No.	Name
<i>3-Spindle Attachment—Continued</i>			
13895	Cam Advance Lever	13953	Shift Fork Shaft
13896	Spindle Cover	13954	Lower Lock Pin
13897	Spindle Housing	13955	Upper Lock Pin
13898	Rear Pivot Pin	13956	Long Driven Gear
13899	Front Pivot Pin	13957	Long Driven Gear Bushing
13900	Spindle Push Rod Bearing	13958	Drilling Shift Gear
13901	Spindle Push Rod Pin	13959	Driven Gear Shaft
13902	Centering Spindle Push Rod Cap	13960	Idler Gear
13903	Spindle Push Rod	13961	Main Shaft Drive Cone
13904	Threading Spindle Push Rod Cap	13962	Main Drive Shaft
13905	Drilling Spindle Push Rod Cap	13963	Threading Spindle Spacer
13906	Spring Anchor Button	13964	Threading Spindle Retainer
13907	Cam Advance Arm Pin	13965	Threading Spindle
13908	Oil Cover Stud	13966	Threading Spindle Rear Bearing
13909	Cam Trip Lever Roller Stud	13967	Threading Spindle Gear
13910	Cam Trip Lever Roller Screw	13968	Pulley Bearing Spacer
13911	Cam Trip Lever Roller	13969	Drive Pulley
13912	Vertical Trip Lever Shaft	13970	Idler Gear Shaft
13913	Upper Trip Lever Pin	13971	Cam Roller Pin
13914	Shift Bearing Housing Bushing	13972	Spindle Bearing Housing Guide
13915	Cam Advance Arm Pivot Pin	13973	Threading Spindle Support Guide
13916	Short Driven Gear Bushing	13974	Spindle Shift Lever Link
13917	Spindle Shift Lever Pivot Pin	13975	Spindle Shift Lever Link Spacer
13918	Shift Fork Stud	13976	Spindle Shift Lever Link Bushing
13919	Upper Trip Lever Stud	13977	Spindle Housing Spring
13920	Sliding Gear Spacer	13978	Threading Push Rod Spring
13921	Pulley Bearing Retainer	13979	Spindle Advance Shaft Spring—Large
13922	Trip Lever Plunger Collar	13980	Horizontal Trip Lever Spring
13923	Gear Shift Bracket Guide Pin	13981	Drilling Spindle Spring
13924	Cam Trip Lever Plunger	13982	Spindle Advance Shaft Spring—Small
13925	Main Spring Lower Anchor	13983	Upper Trip Lever Spring
13926	Collet Lock Nut	13985	Oil Guard Spring
13927	Centering Spindle Cap	13986	Centering Spindle Spring
13928	Vertical Trip Lever Adjusting Screw	13987	Trip Lever Front Spring
13929	Centering Cam Roller Lever Pin	13988	Threading Spindle Spring
13930	Drilling Cam Roller Lever Pin	13989	Plunger Spring
13932	Gear Shift Bracket Pin	13990	Horizontal Trip Lever Spring Anchor
13933	Adjusting Screw Knob	13991	Main Spring Upper Anchor
13934	Spindle Housing Bracket	13992	Upper Trip Lever Spring Anchor
13935	Spindle Advance Arm Guide Shaft	13993	Threading Push Rod Spring Anchor
13936	Drilling Spindle Bearing Spacer	13994	Threading Trip Block
13937	Spindle Advance Shaft Collar	13995	Gear Shift Block
13938	Spindle Advance Shaft	13996	Oil Reservoir Cover
13939	Drilling Spindle Bearing Inner Nut	13997	Gear Guard
13940	Threading Trip Block Shaft	13998	Centering Spindle Front Bearing
13941	Drilling Spindle Bearing Outer Nut	13999	Drilling & Centering Spindle Center Bushing
13942	Spindle Advance Shaft Guide Collar	14000	Threading Spindle Center Bushing
13943	Drilling Spindle	14001	Drilling Spindle Front Bearing
13944	Drill Spindle Key	14002	Threading Spindle Front Bearing
13945	Centering Spindle	14003	Spindle Adjusting Stud
13946	Short Driven Gear	14004	Shift Bearing Nut
13947	Threading Drive Gear	14005	Drilling Spindle Spring Anchor
13948	Drilling Drive Gear	14128	Pulley Adjusting Spacer
13949	Drive Gear Adapter	14037	Rear Pivot Pin Bushing
13950	Spring Adjusting Knob	14038	Shift Bearing Retainer Ring
13951	Spring Adjusting Screw	14039	Shift Bearing Housing
13952	Cam Trip Lever Stud	14132	Spindle Oiling Tube
		14445	Attachment Jack Screw
		14658	Cam Advance Arm Bracket Shim



PARTS LIST FOR GORTON AUTOMATIC MODEL 16-B—Continued

Part No.	Name	Part No.	Name
3-Spindle Attachment—Continued			
K-26	M-203 K Fulcrum Bearing	K-2763	No. 10-24 x 3/4" Slotted Set Screw
K-85	No. 3 Ball Bearing Lock Nut	K-2833	Fulcrum Ball Brg. No. 203L
K-133	1/4"-20 x 1/2" Flat Hd. Cop Screw	K-2837	Coilover Line Assembly
K-136	1/4"-20 x 7/8" Flat Hd. Cop Screw	K-2838	Reducing Bushing 3/8" to 1/4"
K-137	1/4"-20 x 1" Flat Hd. Cop Screw	K-2842	Work 2" Long
K-187	1/4"-20 x 1/4" Flat Pt. Sock. S.S.	K-3217	No. 1226 Gln. Oil Cup Style "L"
K-189	1/4"-20 x 5/16" Flat Pt. Sock. S.S.	K-3306	1/4"-20 x 1/4" Cone Pt. Sock. S.S.
K-190	1/4"-20 x 3/8" Cup Pt. Sock. S.S.	K-3307	1/4"-20 x 3/8" Cone Pt. Sock. S.S.
K-192	1/4"-20 x 5/8" Flat Pt. Sock. S.S.	985-1	Belt Tightener for 3 Spindle
K-260	5/16"-18 x 1" Hex. Cop Screw	Slitting Attachment for 16-B	
K-263	5/16"-18 x 1-1/4" Hex. Hd. Cop Screw	4987	Adjusting Screw for Micro. Collar
K-288	1/4"-20 Hex. Full Nut	7027	Collar for Brake Screw
K-291	5/16"-18 Hex. Full Nut	8922	Roller Yoke Nut
K-297	3/8"-16 Hex. Full Nut	10144	1/4" Finger Washer
K-304	1/2"-13 Hex. Full Nut	11994	1/8" dia. x 1/32" Brass Disc
K-334	1/4"-20 x 5/8" Flat Hd. Mach. Screw	11995	3/16" dia. x 1/16" Brass Disc
K-362	No. 8-32 x 7/16" Flat Hd. Mach. Screw	12086	Clamping Sleeve—Long
K-363	No. 10-32 x 1/2" Flat Hd. Mach. Screw	12087	Clamping Sleeve—Short
K-406	Socket Pipe Plug 1/8" P.	12098	Link Pin
K-461	No. 0 x 1" Taper Pin	12337	Rocker Arm Roller
K-471	No. 2 x 1-1/4" Taper Pin	12372	Washer for Set Screw
K-513	No. 301 Gln. Oiler	12377	Spring Anchor in Lever
K-514	No. 302 Gln. Oiler	12627	Slitting Attachment Idler Pulley Bracket
K-515	No. 303 Gln. Oiler	12628	Slitting Attachment Idler Pulley Shaft
K-520	No. 2 Woodruff Key	12629	Slitting Attachment Idler Pulley Collar
K-1234	5/16"-18 Hex. Half Nut	12720	Slitting Attachment Gear Box
K-1317	1/4"-20 Hex. Half Nut	12721	Slitting Attachment Gear Cover
K-1569	1/4"-20 x 3/8" Flat Pt. Sock. S.S.	12722	Slitting Attachment Gear
K-1986	No. 10-32 x 1/4" Flat Pt. Sock. S.S.	12723	Slitting Attachment Gear Shaft—Short
K-1987	No. 10-32 x 1/8" Flat Pt. Sock. S.S.	12724	Slitting Attachment Gear Shaft—Long
K-1988	No. 10-32 x 3/8" Flat Pt. Sock. S.S.	12725	Slitting Attachment Pulley Bracket—Upper
K-2000	1/4"-20 x 1" Sq. Hd. Set Screw	12727	Saw Drive Sprocket
K-2002	1/4"-20 x 1 1/2" Sq. Hd. Oval Pt. S.S.	12728	Saw Drive Sprocket
K-2003	1/4"-20 x 1 7/8" Full Dog Pt. Sock. S.S.	12729	Saw Idler Sprocket
K-2006	1/4"-20 x 1 1/2" Flat Pt. Sock. S.S.	13838	Frame
K-2008	1/4"-20 x 1-1/4" Flat Pt. Sock. S.S.	13840	Spindle Barrel
K-2023	5/16"-18 x 2" Sq. Hd. Flat Pt. S.S.	13841	Spindle
K-2262	1/4"-20 x 1/2" Flat Pt. Sock. S.S.	13842	Spindle Nut
K-2647	No. 10-32 x 1/4" Flat Pt. Slotted S.S.	13843	Spindle Sleeve
K-2649	1/4"-20 x 2-3/8" Sq. Hd. Set Screw	13844	Spindle Sleeve Nut
K-2666	No. 39 Fulcrum Brg.	13845	Adjusting Screw Collar
K-2694	No. 10-32 x 1/4" Cone Pt. Sock. S.S.	13846	Adjusting Screw
K-2695	1/4"-20 x 1-3/4" Sq. Hd. Cup Pt. Set Screw	13847	Disc Saw Collar
K-2697	884 Bunting Bush (3/8" x 3/8" x 1")	13848	Disc Saw Washer
K-2698	1/4"-20 x 1" Cup Pt. Sock. Set Screw	13849	Saw Guard
K-2699	No. 1208 Style "L" Gln. Oiler	13850	Saw Guard Spring
K-2700	1/4"-20 x 1/2" Cone Pt. Sock. Set Scr.	13851	Quadrant
K-2703	1/4"-20 x 2" Oval Hd. Pt. Sq. Hd. S.S.	13852	Quadrant Link
K-2704	5/16"-18 x 5/8" Flat Pt. Sock. S.S.	13853	Cam Lever Pivot Pin
K-2705	No. 8-32 x 1/8" Flat Hd. Mach. Screw	13854	Pinion
K-2706	A-520-5 Oiler Brg.	13855	Pinion Stud
K-2707	1/4"-20 x 5/16" Cone Pt. Sock. S.S.	13856	Transfer Arms
K-2709	No. 10-32 x 5/16" Flat Hd. Mach. Screw	13857	Transfer Arm Shaft Bearing
K-2710	1/4"-20 x 1" Flat Pt. Sock. S. Screw	13858	Transfer Arm Shaft
K-2711	1/4"-20 x 1/4" Full Dog Pt. Sq. Hd. Set Screw	13859	Transfer Arm Bushing
K-2713	No. 10-32 x 5/16" Flat Pt. Slotted Set Screw	13860	Transfer Arm Shaft
K-2762	No. 10-24 x 1/4" Slotted Set Screw	13861	Knock Out Pin Arm



PARTS LIST FOR GORTON AUTOMATIC MODEL 16-B

Part No.	Name	Part No.	Name
<i>Slotted Attachment for 16-B—Continued</i>			
13862	Bell Cam Lever	K-2646	1/4" -20 x 1" Hex. Hd. Cop. Screw
13863	Bell Cam Roller Guide	K-2647	10-32 x 1/4" Slotted Set Screw
13864	Bell Cam Lever Roller	K-2649	1/4" -20 x 2-3/8" Flat Pl. Sq. Hd. Set Screw
13865	Bell Cam Lever Roller Pin	K-2650	9-32 x 5/16" Flat Pl. Sock. Set Screw
13866	Quadrant Pin	K-2747	Oilite Bushing F-1101-5
13867	Cam Roller Pin	K-2797	Oilite Bushing A-710-6
13868	Knock Out Pin Arm Shaft	K-2776	3/8" Flank Chain 118 Pitches
13869	Spring Anchor—Short	K-2713	No. 10-32 x 5/16" Flat Pl. Slotted Set Screw
13870	Spring Anchor—Long	K-2871	No. 10-32 x 5/16" Round Head Machine Screw
13871	Guide Collar	<i>Planex Attachment for 16-B</i>	
13872	Adjusting Block	11995	3/16" Dia. x 1/16" Brass Disc
13873	Spring—Long	12268	Planex Bell Crank Pivot Pin
13874	Spring—Short	12669	Planex Adjusting Screw Knob
14043	Long Gear Shaft Collar	12271	Planex Adjusting Screw
14044	Main Shaft Pulley	12272	Planex Slide End Plate
14045	Idle Pulley	12275	Planex Bell Crank
14104	Cam Segment	12276	Planex Bell Crank Heel
14105	Flat Cam	12277	Planex Bell Crank Heel Stud
14106	Bell Cam	12278	Planex Cam Follower
14221	Chain Guard Plate	12279	Planex Shoe
14222	Chain Guard Plate Flange	12982	Planex Adjusting Screw Stud
14443	Chain Guard Bracket	12984	Planex Adjusting Screw Plate
14666	Cam Segment Pin	12285	Planex Spring Front Anchor
15208	Plate for Gear Box	12816	Planex Spring Rear Anchor
K-113	No. 0233 Boring Bushing	12817	Planex Spring
K-135	1/4" -20 x 3/4" Sock. Hd. Cop. Scr.	12288	Planex Gibs
K-151	3/8" -16 x 1/8" Sock. Cop. Screw	12356	No. 1 & 2 Tool Adj. Brkt. Stop Stud
K-161	1/2" -13 x 1 1/2" Sock. Hd. Cop. Screw	12753	1/8" Dia. x 7/8" Drill Rod Pin
K-185	1/4" -20 x 1 1/4" Sock. Set Screw	12274	Planex Bracket
K-187	1 1/4" -20 x 1/4" Flat Pl. Sock. S.S.	12273	Planex Cam Oil Trough
K-188	1/4" -20 x 5/16" Cop. Pl. Sock. S.S.	12496	Planex Screw Collar
K-189	1/4" -20 x 5/16" Flat Pl. Sock. S.S.	12259	1/4" x 3/4" D.R.P.
K-201	5/16" -18 x 1 1/2" Flat Pl. Sock. S.S.	12611	1/4" x 1" D.R.P.
K-288	1/4" -20 Hex. Nut	14625	Oil Trough Thumb Screw
K-291	5/16" -18 Hex. Nut	14626	Planex Adjusting Screw Wheel Bracket
K-299	3/8" -24 Hex. Nut	14651	Planex Slide
K-452	1/4" Std. Wrought Washer	15175	Planex Gibs Screw
K-479	No. 3 x 1 1/2" Taper Pin	K-175	7/16" Std. Wrought Washer
K-513	No. 301 Gits Oiler	K-199	5/16" -18 x 1/2" Cup Pl. Sock. Set Screw
K-514	No. 302 Gits Oiler	K-258	3/16" -18 x 3/4" Hex. Hd. Cop. Screw
K-585	No. 9 Woodruff Key	K-287	No. 10-32 Hex. Nut
K-1985	8-32 x 1/2" Sock. Hd. Cop. Screw	K-453	5/16" Std. Wrought Washer
K-1986	10-32 x 1/4" Flat Pl. Sock. Set S.	K-513	No. 301 Gits Oiler
K-1987	10-32 x 5/16" Flat Pl. Sock. S.S.	K-1354	5/16" -18 Hex. Half Nut
K-1991	19-32 x 1/2" Sock. Hd. Cop. Screw	K-1618	1/4" -20 x 5/8" Socket Head Cop. Screw
K-2001	1/4" -20 x 1 1/2" Sq. Hd. Set Screw	K-1991	No. 10-32 x 1/2" Socket Cop. Screw
K-2010	5/16" -18 x 3/4" Flat Pl. Sock. S.S.	K-1994	1/4" -20 x 3/8" Flat Point Sq. Head Set Scr.
K-2024	No. 1612 Alemite Grease Fitting	K-1999	1/4" -20 x 7/8" Flat Point Sq. Head Set Scr.
K-2027	5/16" -18 x 1 1/4" Sock. Hd. Cop. Screw	K-2003	1/4" -20 x 2" Flat Point Sq. Head Set Screw
K-2030	3/8" -16 x 1 1/8" Sock. Cop. Screw	K-2014	No. 5/0 x 5/8" Taper Pin
K-2176	1/2" Washer	<i>Milling Attachment for 16-B</i>	
K-2469	1/8" x 1/2" Rivet	4987	Thumb Screw
K-2572	No. C-200 Boring Bushing	11995	Brass Plug
K-2575	3/78" O.D. Gates Pulley	12357	Rockers Arm Roller
K-2577	No. 2310 Gates Belt	12338	Rockers Arm Roller Pin
K-2578	No. 2650 Gates Belt		
K-2644	1/4" -20 x 1-1/8" Flat Pl. Sq. Hd. S.S.		



PARTS LIST FOR GORTON AUTOMATIC MODEL 16-B—Continued

Part No.	Name	Part No.	Name
<i>Milling Attachment for 16-B—Continued</i>			
12377	Spring Anchor	14205	Spindle Segment
14468	Spindle Sleeve	14306	Spindle Advance Lever Roller Stud
14469	Rocker Arm	14307	Die Head Trip Lever
14470	Gear Cover	14308	Spindle Advance Inner Lever
14471	Spindle Barrel	14309	Spindle Advance Short Link
14472	Bracket	14310	Advance Lever Shaft
14473	Cutter Shaft	14311	Spindle Advance Long Link
14474	Pulley	14312	Advance Lever Link Pin
14475	Spindle	14313	Clamping Sleeve
14476	Driven Gear	14314	Clamping Sleeve Nut
14477	Drive Gear	14315	Spindle Segment Guide
14478	Spindle Lock Nut	14316	Coolant Bracket Guide Pin
14479	Spindle Sleeve Nut—Rear	14317	Adjusting Screw Bracket Stop
14480	Spindle Sleeve Nut—Front	14318	Coolant Advance Bracket
14481	Milling Cutter Adapter (7/8")	14319	Coolant Bracket
14482	Milling Cutter Adapter (3/4")	14320	Spindle
14483	Pulley Shaft	14321	Spindle Cap
14484	Pulley Shaft Bushing	14322	Coolant Gland
14485	Take Up Bushing	14323	Coolant Connector
14486	Gear Cover Pin	14324	Spring Guide
14487	Pinot Pin	14325	Spindle Advance Shaft Guide
14488	Spacer	14326	Die Closing Rod
14489	Rocker Arm Pivot Pin	14327	Spindle Advance Bud
15182	Spindle Barrel Bushing	14328	Advance Link Spring Retainer
K-185	1/4"-20 x 1/4" Cup Point Socket Set Screw	14329	Spindle Key
K-187	1/4"-20 x 1/4" Socket Flat Point Set Screw	14330	Die Closing Sleeve
K-263	5/16"-18 x 1-1/4" Hex. Head Cap Screw	14331	Spindle Sleeve
K-299	3/8"-24 Hex. Nut	14332	Rear Spindle Bearing Spacer
K-470	No. 2 x 1" Taper Pin	14333	Front Spindle Bearing Spacer
K-513	Oil Cup	14334	Spindle Lock Nut
K-551	No. 3 Woodruff Key	14335	Rear Spindle Bearing Oil Retainer
K-2017	No. 1 x 1 1/4" Taper Pin	14336	Rear Spindle Bearing Housing Cap
K-2529	No. 10-32 x 1/4" Cup Point Socket Set Scr.	14337	Front Spindle Bearing Housing Cap
K-2708	1/4"-20 x 1-3/4" Sq. Head Oval Ph. Set Scr.	14338	Front Spindle Bearing Oil Retainer
K-2756	No. 4-74.6 Bunting Bushing	14339	Spindle Key Roller
<i>13-16" Die Head Adapter</i>			
14230	13-16" Die Head Adapter	14340	Oil Retainer Felt Washer
K-2918	No. 8-32 x 1/4" Socket Cone Point Set Screw	14341	Spindle Housing
<i>Single Spindle Threading Attachment with Self Opening Die Head</i>			
3271	Screw	14445	Support Stud
6361	Long Spring	14446	Centering Stud
6715	Felt Washer	14687	Spindle Key
9442	Felt Washer	14690	Attachment Pulley
11428	Short Spring	14691	Attachment Pulley
11994	Brass Plug	15231	Cam
12372	Washer	15232	Cam
12835	Pin 3/16" Dia. x 7/16" Long	K-2027	5/16"-18 x 1-3/4" Socket Cap Screw
12910	Filister Head Screw	K-187	1/4"-20 x 1/4" Socket Set Screw
13911	Cam Roller	K-263	5/16"-18 x 1-1/4" Hex. Head Cap Screw
14301	Spindle Pulley	K-288	1/4"-20 Hexagon Nut
14302	Adjusting Screw Bracket	K-291	5/16" Hexagon Nut
14303	Spindle Advance Lever	K-304	1/2"-13 Hexagon Nut
14304	Guide Ferrule	K-405	Pipe Plug 1/8"
		K-478	No. 3 x 1/4" Taper Pin
		K-513	No. 301 Oil Cup
		No. 10-32 x 1/4" Socket Set Screw	
		K-1987	No. 10-32 x 5/16" Socket Set Screw
		K-1990	No. 10-32 x 5/8" Socket Cap Screw
		K-2000	1/4"-20 x 1" Square Head Set Screw
		K-2447	1/2"-20 x 5/8" Socket Set Screw
		K-2449	No. 206K Single Row Radial Ball Bearing



PARTS LIST FOR GORTON AUTOMATIC MODEL 16-B—Continued

Part No.	Name	Part No.	Name
Single Spindle Threading Attachment with Self-Opening Die Head—Continued		13287	Rawhide Support Tube—Long
K-2643	No. 10-32 x 1.2" Square Head Set Screw	13288	Rawhide Lock Screw
K-2649	1.4" x 2.0 x 2.3/8" Square Head Set Screw	13289	Stock Gripper Pivot Stud
K-2700	1.4" x 2.0 x 1.2" Cone Point Set Screw	K-1323	1.4" x 2.0 x 1.2" Socket Cap Screw
K-2708	1.4" x 2.0 x 1.3/4" Square Head Set Screw	K-1354	5/16"-18 Hexagon Half Nut
K-2709	No. 10-32 x 5/16" Flat Head Machine Screw	K-1986	No. 10-32 x 1.4" Socket Set Screw
K-2725	No. 1451 Gits Oil Cap	K-2834	No. 10-32 x 7/8" Full Dog Socket Set Screw
K-2733	No. 10-32 x 3/8" Flat Point Socket Set Screw	K-2835	1/8" dia. x 1.1/4" long rawhide
K-2734	No. 10-32 x 1/4" Flat Point Socket Set Screw	K-2836	1/8" dia. x 2-1/2" long rawhide
K-2827	No. 134-4 Threlax Metal Hose	Work Separator Attachment	
K-2838	3/8" to 1.4" Pipe Reducer	12377	Spring Anchor
K-2922	Bunting Bushing "BA-4"	13911	Cam Trip Lever Roller
K-2923	Bunting Bushing "C-10"	14059	Work Separator Bracket
K-2927	No. 10-32 x 1.1/8" Socket Cap Screw	14060	Work Separator Trough
K-2928	1.4" x 2.0 x 7/8" Dog Point Set Screw (Slotted)	14061	Work Separator Lever
K-3214	11/16" x 1.1/4" x 95" Belt	14062	Separator Trough Cam
K-3302	Flat Head Machine Screw No. 6-40 x 1.1/8"	14063	Lever Pivot Stud
Oil Spray Guard for 16-B Screw Machine		14064	Trough Cam Roller Pin
14086	Frame Channel	14065	Cam Roller Pin
14087	Frame Channel Spacer	14066	Trough Pivot Pin Bushing
14088	Front Upper Frame Angle	14067	Trough Pivot Pin
14089	Front Side Frame Angles (L.H.)	14068	Cam Tension Spring
14090	Front Side Frame Angles (R.H.)	14069	Trough Tension Spring
14091	Front Lower Frame Angle	14070	Work Separator Cam
14092	Front Glass Panel	K-185	1.4" x 2.0 x 1.4" Cup Point Socket Set Screw
14094	Rear Upper Hanger Panel	K-2653	5/16"-18 x 1.1/4" Hexagon Head Cap Screw
14095	Rear Upper Hanger Channel	K-305	1.2" x 1.5 Hexagon Half Nut
14097	Rear Side Frame Angle (L.H.)	K-1618	1.4" x 2.0 x 5/8" Socket Cap Screw
14098	Rear Side Frame Angle (R.H.)	Attachment Spindle Advance Roller	
14099	Rear Lower Frame Angle	10814	Washer
14100	Rear Glass Panel	13235	Attachment Spindle Advance Roller
14102	Rear Lower Hinge Panel	13236	Attachment Spindle Advance Roller Stud
14231	Frame Channel Hinge	K-373	No. 10-32 x 3/8" Filler Head Machine Screw
14232	Frame Angle Hinge	K-1354	5/16"-18 Hexagon Nut
K-287	No. 10-32 Hexagon Nuts	Knurling Tool	
K-362	No. 8-32 x 7/16" Flat Head Machine Screw	12594	1/8" x 7/16" Long Drill Rod Pin
K-1763	No. 8-32 Hexagon Nuts	14036	Knurl Holder
K-2871	No. 10-32 x 5/16" Round Head Machine Scr.	14357	Straight Knurl (40 Pitch)
K-2872	1.2" x 1.3/4" x 5.1/4" Hexagon Head Cap Screw	14358	Straight Knurl (60 Pitch)
K-3133	No. 159 Speed Nut 5/8" x 3/8" Wide	14359	Straight Knurl (80 Pitch)
K-3134	3/16" dia. x 7/16" long rivet (Soft Iron)	14360	Diamond Knurl (Fine Pitch)
K-3196	No. 8-32 x 5/16" Flat Head Machine Screw	14361	Diamond Knurl (Med. Pitch)
		14362	Diamond Knurl (Coarse Pitch)
Stock Gripper Attachment for 16-B		Multiple Feed	
11999	1/8" x 1/16" Brass Disc	3336	5/16"-24 Hexagon Nut
12574	1/8" x 5/8" Drill Rod Pin	10144	1/4" Dia. Washer
13280	Stock Gripping Lever	10233	5/16" Dia. Washer
13281	Lever Support Bracket	12243	Collar Closing Bearing Stud
13282	Stock Gripper Cam	12753	1/8" x 7/8" Drill Rod Pin
13283	Stock Gripper Cam Follower	12768	1/4" x 7/8" Drill Rod Pin
13284	Cam Follower Pivot Stud	12874	1/8" x 5/8" Drill Rod Pin
13285	Support Tube—Bracket	14051	Collar Opening Tool—Inner
13286	Rawhide Support Tube—Short	14085	Collar Opening Tool—Outer



Photo 153

PARTS LIST FOR GORTON AUTOMATIC MODEL 16-B—Continued

Part No.	Name	Part No.	Name
<i>Bell Cam Attachment for 16-B—Continued</i>		12496	Planex Adjusting Screw Collar
12358	Bell Cam Roller Yoke	12497	Oil Trough Thumb Screw
12359	Bell Cam Attachment Lever Pin	12299	1/4" x 3/4" D.R.P.
12360	Bell Cam Attachment Lever	K-192	1/4" x 20 x 5/8" Flat Point Socket Set Screw
12361	Bell Cam Adjusting Screw	K-513	No. 301 Gits Oil Cup
12362	Bell Cam Adjusting Toe	K-1354	5/16" x 18 Hexagon Half Nut
12363	Bell Cam Toe	K-1618	1/4" x 20 x 5/8" Socket Head Cap Screw
12364	Bell Cam Attachment Shoe Holder	K-2289	No. 10-32 x 1/2" Half Dog Pin Socket Set Ser.
12365	Bell Cam Shoe	K-1991	No. 10-32 x 1/2" Socket Cap Screw
12366	Bell Cam Attachment Bracket	K-2014	No. 5/0 x 5/8" Taper Pin
		K-175	7/16" x 14 x 1-1/4 H.K. Socket Cap Screw



Photo 154

DECIMAL EQUIVALENTS

$\frac{1}{64}$ — .015625	$\frac{33}{64}$ — .515625
$\frac{1}{32}$ — .03125	$\frac{17}{32}$ — .53125
$\frac{3}{64}$ — .046875	$\frac{35}{64}$ — .546875
$\frac{1}{16}$ — .0625	$\frac{9}{16}$ — .5625
$\frac{5}{64}$ — .078125	$\frac{27}{64}$ — .578125
$\frac{3}{32}$ — .09375	$\frac{19}{32}$ — .59375
$\frac{7}{64}$ — .109375	$\frac{39}{64}$ — .609375
$\frac{1}{8}$ — .125	$\frac{5}{8}$ — .625
$\frac{9}{64}$ — .140625	$\frac{41}{64}$ — .640625
$\frac{5}{32}$ — .15625	$\frac{21}{32}$ — .65625
$\frac{11}{64}$ — .171875	$\frac{43}{64}$ — .671875
$\frac{3}{16}$ — .1875	$\frac{11}{16}$ — .6875
$\frac{13}{64}$ — .203125	$\frac{45}{64}$ — .703125
$\frac{7}{32}$ — .21875	$\frac{23}{32}$ — .71875
$\frac{15}{64}$ — .234375	$\frac{47}{64}$ — .734375
$\frac{1}{4}$ — .25	$\frac{3}{4}$ — .75
$\frac{17}{64}$ — .265625	$\frac{49}{64}$ — .765625
$\frac{9}{32}$ — .28125	$\frac{25}{32}$ — .78125
$\frac{19}{64}$ — .296875	$\frac{51}{64}$ — .796875
$\frac{5}{16}$ — .3125	$\frac{13}{16}$ — .8125
$\frac{21}{64}$ — .328125	$\frac{53}{64}$ — .828125
$\frac{11}{32}$ — .34375	$\frac{27}{32}$ — .84375
$\frac{23}{64}$ — .359375	$\frac{55}{64}$ — .859375
$\frac{3}{8}$ — .375	$\frac{7}{8}$ — .875
$\frac{25}{64}$ — .390625	$\frac{57}{64}$ — .890625
$\frac{13}{32}$ — .40625	$\frac{29}{32}$ — .90625
$\frac{27}{64}$ — .421875	$\frac{59}{64}$ — .921875
$\frac{7}{16}$ — .4375	$\frac{15}{16}$ — .9375
$\frac{29}{64}$ — .453125	$\frac{61}{64}$ — .953125
$\frac{15}{32}$ — .46875	$\frac{31}{32}$ — .96875
$\frac{31}{64}$ — .484375	$\frac{63}{64}$ — .984375
$\frac{1}{2}$ — .5	1 — 1.



Photo 155

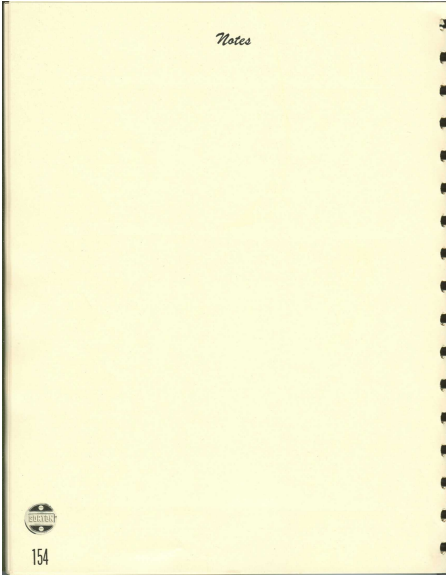
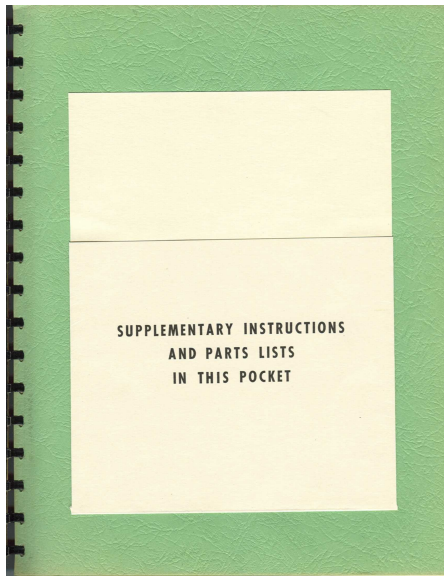


Photo 156



HAVE YOU THESE NEW GORTON BOOKS ?

The advertisement displays ten Gorton books arranged in a grid. Each book cover is shown at an angle, revealing its title and publisher information. The books are: 1. Pentagraph (No. 1320 B), 2. GORTON Die Castings Instruction Book and Parts Catalog (No. 1325 A), 3. GORTON Die Castings Instruction Book and Parts Catalog (No. 1325 C), 4. GORTON Die Castings Instruction Book and Parts Catalog (No. 1325 D), 5. MILLING MACHINES (No. 1400 C), 6. DIE & MOLD COPIERS (No. 1310 B), 7. GORTON Accessories Instruction Manual and Parts Catalog (No. 1370 C), 8. GORTON Machine-Tools Instruction Manual and Parts Catalog (No. 1400 D), 9. GORTON Hydraulic Presses (No. 1300 B), and 10. HANDBOOK (No. 1325 A). A central circular logo for GORTON MACHINE TOOLS is positioned above a paragraph of text.

SEND FOR *Yours* TODAY

You will find them filled with practical ideas for enlarging the usefulness of your Gorton Machines and improving operating methods.

Our Engineers Will Gladly Consult with You